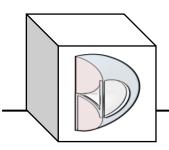


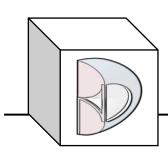


- The following licenses are required to create this DMU Kinematic simulation:
 - Digital Mockup Kinematics
 - Mechanical Part Design
 - Generative Shape Design
 - Assembly Design



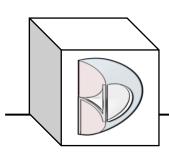


- To create this DMU Kinematic simulation, we must begin with several known parameters.
- Known:
 - All suspension "hard points".
 - Pivot points and lines
 - Angles of axes and planes
 - Min/Max Command values
 - Driveshaft rotation (-720deg, 720deg)
 - Shock down/up (-49.5mm, 35.5mm)



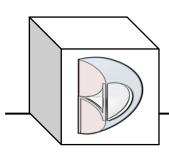


Vehicle Suspension Hardpoints														
Rear Suspension														
Scrub (Pivot) Radi	us =			162.3	mm									
Kingpin Inclination	Angle =			-7.1	deg									
Caster Angle =				1.2	deg									
Mechanical (or cas	ster) trail (if applicable) =			6.83	mm									
Toe Angle = -0.01 de			deg	+/- 0.10	deg									
Camber Angle = -0.18 de				+/- 0.5	+/- 0.5 deg									
Knu	uckle Attachment Points relative to Sus	spension /	Analysis /	Axis										
	Various links and arms depend upon the Rear Suspension configuration. (i.e. Dependent vs. Semi-Independent vs. Independent Suspension)													
Configuration:	GM 5-link Independent Rear Suspensi	ion (axle l	nalf-shaft	acts as 5t	th link)									
List each Knuckle	Attachment Point relative to Susp Ana	lysis Axi	s = (x, y,	z)										
Camber Strut Pivo	t Point (inboard) = (x, y, z)													
Camber Strut Pivo	t Dt (inhoard) =	Х	Υ	Z										
Camber Strut Pivo	t Pt (Inboard) =	-2486.13	153.90	201.63	mm									
Camber Strut Pivot Point (outboard) = (x, y, z)														
Comban Charle Direct Dt (code cond)		Х	Υ	Z										
Camber Strut Pivo	ot Pt (outboard) =	-2486.14	603.67	187.51	mm									



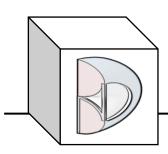


Tie Rod Pivot Point (inboard) = (x, y, z)										
Tie Rod Pivot Pt (inboard) =	Х	Υ	Z	mm						
He Rout Worl E (Illocato) -	-2687.5	30.00	304.36	"""						
Tie Rod Pivot Point (outboard) = (x, y, z)										
Tie Rod Pivot Pt (outboard) =	Х	Y	Z	mm						
	-2687.6	526.35	288.72	111111						
Upper Trailing Arm Pivot Point (forward) = (x, y, z)										
Upr Trailing Arm Pivot Pt (forward) =	Х	Υ	Z	mm						
opi Halling Alli Pivot Pt (lorward) =	-2158.46	528.78	410.39] ''''' [
Upper Trailing Arm Pivot Point (rearward) = (x, y, z)										
Upr Trailing Arm Pivot Pt (rearward) =	Х	Υ	Z	mm						
opi Halling Alli Pivot Pt (lealward) =	-2437.84	528.78	407.14							
Lower Trailing Arm Pivot Point (forward) = (x, y, z)										
Lwr Trailing Arm Pivot Pt (forward) =	Х	Υ	Z	mm						
LWI Halling Alli Pivot Pt (lolward) =	-2122.80	528.78	312.57	111111						
Lower Trailing Arm Pivot Point (rearward) = (x, y, z)										
Lwr Trailing Arm Pivot Pt (rearward) =	Х	Υ	Z	mm						
Lwi Halling Alli Hvot Ft (lealward) =	-2438.99	528.78	242.55	111111						



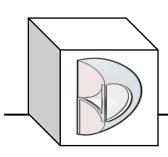


II laint Divet Daint (inhaard) = (v. v. a)											
U-Joint Pivot Point (inboard) = (x, y, z)											
U-Joint Pivot Pt (inboard) =		Х	Υ	Z	mm						
		-2488.90	197.89	339.18	'''''						
U-Joint Pivot Point	(outboard) = (x, y, z)										
U-Joint Pivot Pt (outboard) =		Υ	Z	mm							
		-2488.91	620.91	325.90	1 111111						
	Knuckle/Brakes										
(i.e. hub dia/thickness, pilot dia, hub face to knuckle, stud si					ia, etc.)						
Hub Dimensions:	See CA	AD Data									
Brake Rotor Dimer	nsions:										
Rotor Thickness	=			26.0							
Rotor Diameter =				305.0	mm						
Inner Face of Hu	b to Inner Rotor Surface =			42.0	mm						
Inner Hub Diame	ter =			182.0	mm						
Outer Hub Diameter =				202.0 mm							
Center Hole Diameter =					70.0 mm						
Brake Caliper to Wheel (min clearance) =					mm						
Brake Drum Dimensions (if drum brakes) (i.e. drum dia/depth, ctr hole dia, inner hub face to inner drum surf, etc.)					/a						



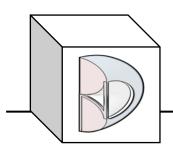


Springs/Shock Absorbers											
Carina Diananaiana	e length, Ground ends, e										
Spring Dimensions:	Se	ee CAD Da	ata								
Spring Rate =				122.5	N/mm						
Shock Absorber Adjustable	Damping (see chart)										
Shock Absorber Type:	Strange - Double Adjustable	Coil-over	- S5003								
Extension Length =					mm						
Compressed Length =					241.30 mm						
Stroke Length (without bump stop) =					85.34 mm						
Bump Stop (length) =				14.30							
Shock Mounting Pivot Point	s relative to Susp Analysis A	xis = (x,)	/, Z)								
Shook Upper Direct Daint -		Χ	Υ	Z	100,100						
Shock Upper Pivot Point =		-2330.68	363.45	408.43	mm						
Shock Lower Pivot Point =	81 11 81 18 11		Υ	Z	100,100						
Shock Lower Pivot Point =		-2421.97	470.59	153.70	mm						
Spring and Shock Installation Angle =				70.6/72.9	deg						
Spring and Shock Absorber Motion Ratio @ Ride Ht =											



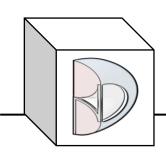


ARB (anti-roll bar)										
ARB (anti-roll bar) Dimensions:										
Virtual torque arm (A):	216.15	mm								
Length of center (B):			819.91	mm						
Physical torque arm (C):			226.50	mm						
OD in millimeters: (D)			20.00	mm						
ID in millimeters: (if hollow)			0.00	mm						
ARB (anti-roll bar) Spring Rate =	29.20	N/mm								
ARB (anti-roll bar) Motion Ratio @ Ride Ht =	0.749	unit/unit								
ARB Bushing Rates (per deflection at loads)										
Design Results										
Tire Envelope =			(see CA	AD Data)						
Tool Clearance =			(see CA	AD Data)						
Routing Clearance (i.e. hoses, brake lines, fuel lines, elect	trical, etc.) =	:	(see CA	AD Data)						
Roll Center Height =			23.39	mm						
Anti-squat Instantaneous Center relative to Susp Analys										
Anti-nevet Instant Contra	Х	Υ	Z							
Anti-squat Instant Center =	-1654.6	n/a	416.25	mm						
Wheel Rate incl ARB =	81.31	N/mm								
Ride Rate incl ARB =	63.26	N/mm								



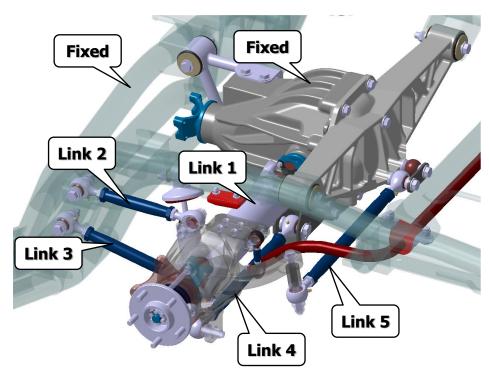


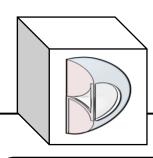
Step 1: Understand the suspension system





- Begin with the completed Rear Suspension Assembly.
- This will help to understand all the pivot and links within this 5-link system.





1) The Differential and all its immoveable attached parts would be the **Fixed** part in this simulation

2) The Input Yoke will rotate with a gear (with command) reduction ratio to the Output Yoke

3) The Inner U-joint will **rotate (pivot)** within the Output Yoke

4) The Halfshaft will rotate (pivot) about the Inner U-joint

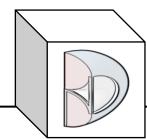
5) The Halfshaft will **rotate (pivot)** about the Outer U-joint

7) The Hub Assy will **rotate** within the Knuckle Assy

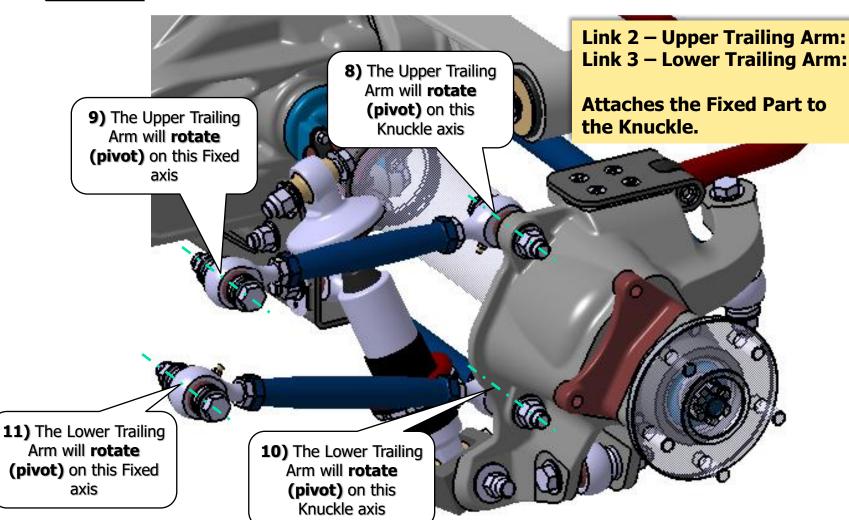
Link 1 – Halfshaft Assy:

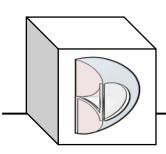
Rotates and attaches Fixed Part to the Knuckle.

6) The Outer U-joint will **rotate (pivot)** within the Hub Assy

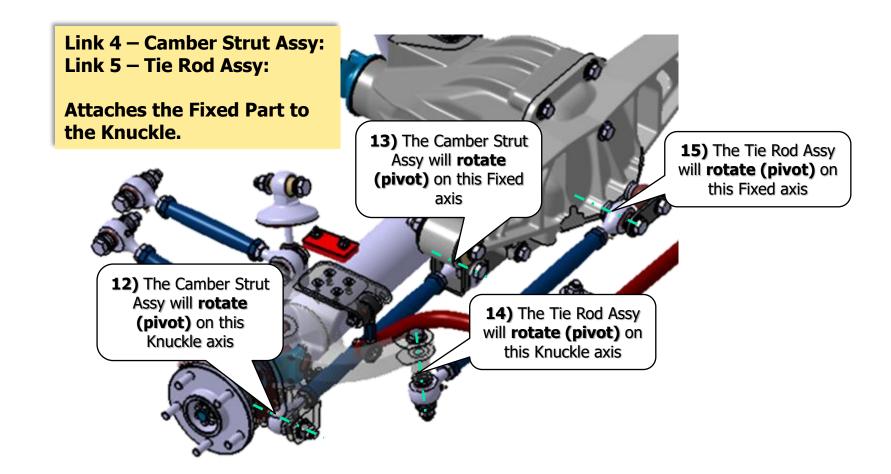


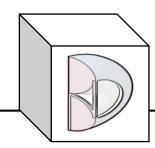




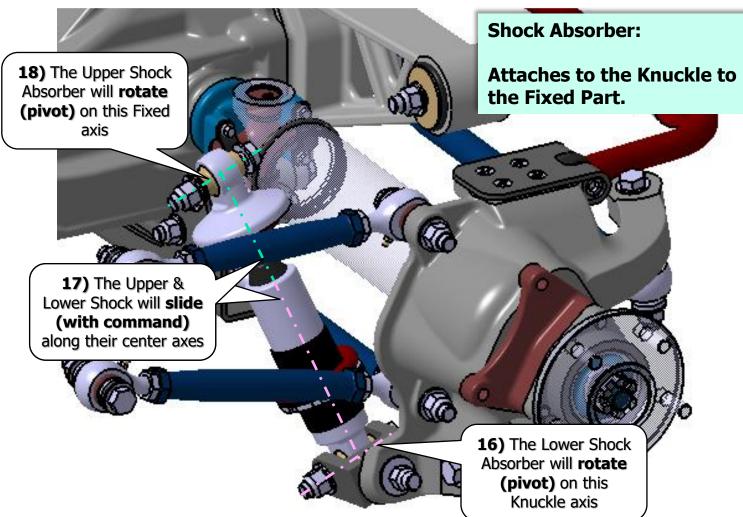


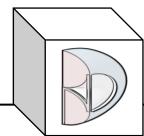


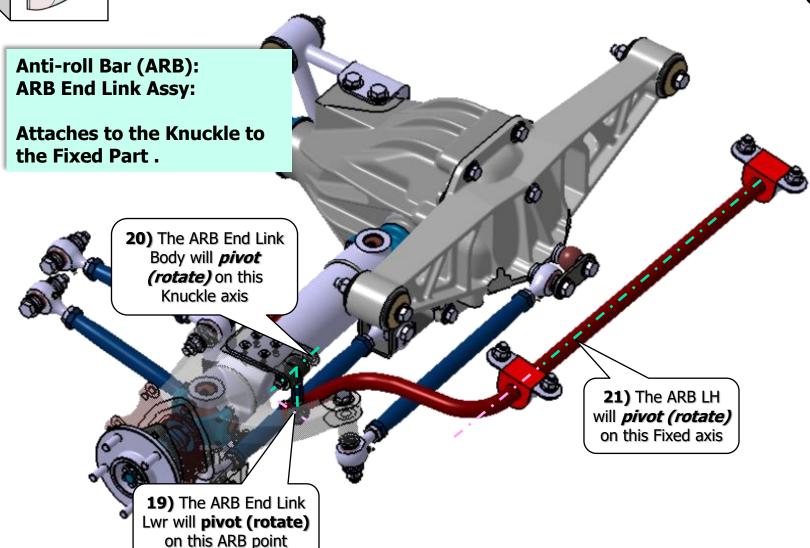


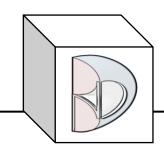






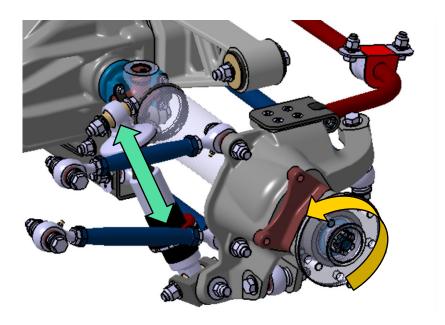


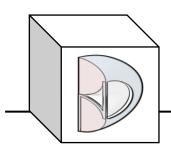






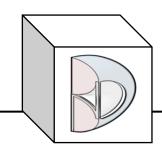
- Understand which joints need commands.
 - Min/Max Command values
 - Driveshaft rotation (-720deg, 720deg)
 - Shock down/up (-49.5mm, 35.5mm)





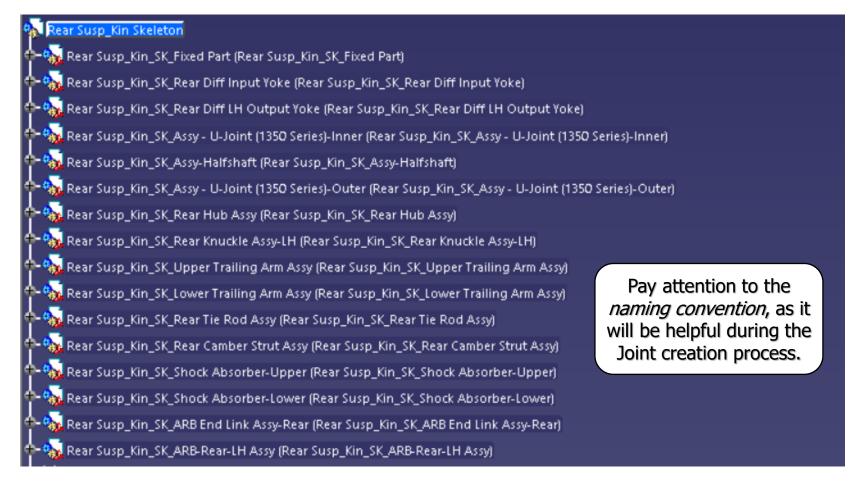


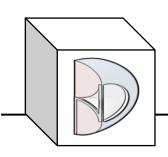
Step 2: Create a kinematic skeleton structure





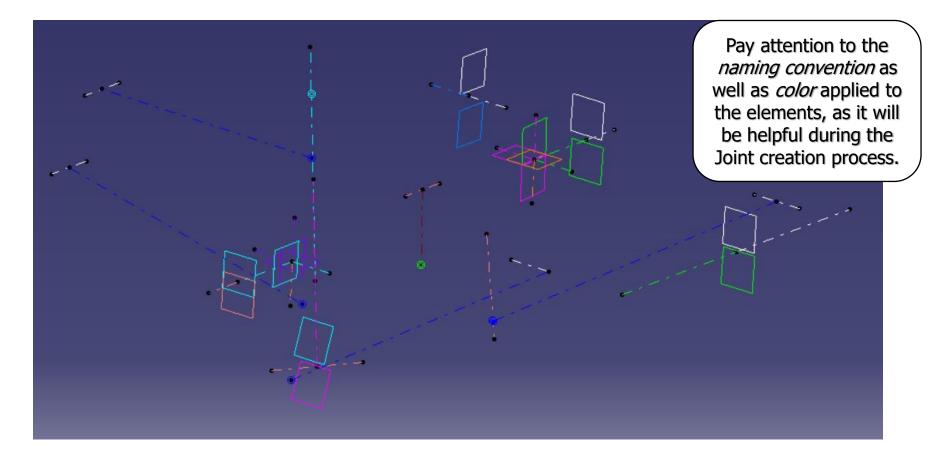
 Create a Skeleton Product and all Parts required for the kinematic simulation.

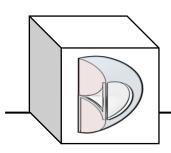






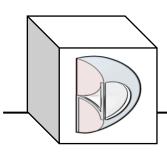
 Create all points, lines, & planes inside each part within the kinematic structure.





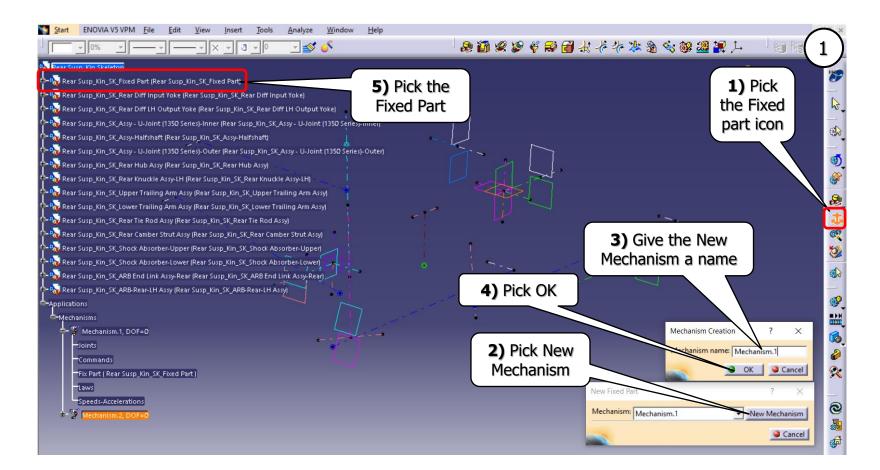


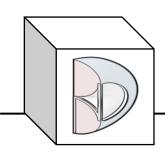
Step 3: Create the kinematic Joints





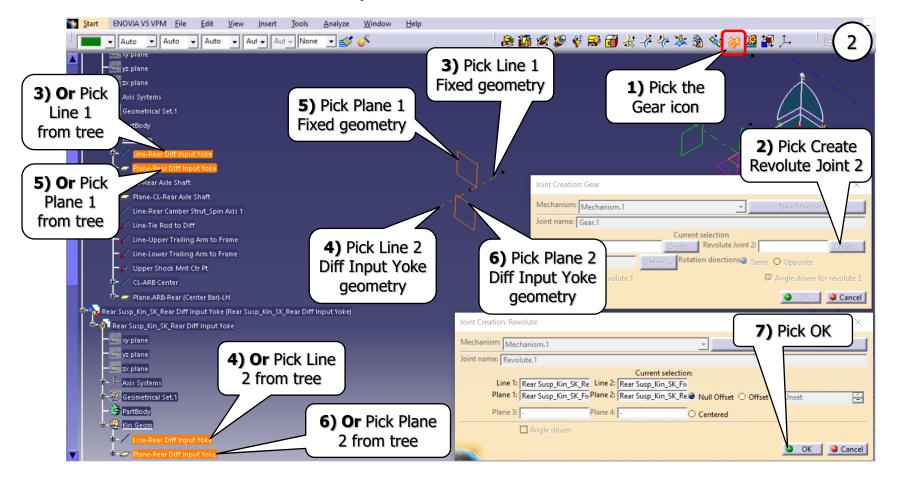
Create the Fixed Part and name the Mechanism.

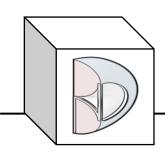






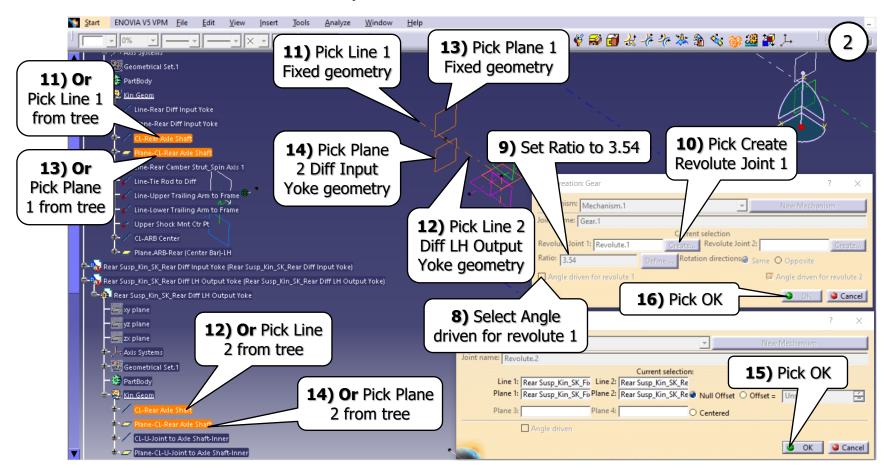
 Create a Gear Joint between the Diff Input Yoke and the Diff LH Output Yoke.

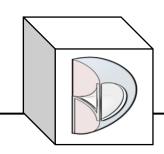






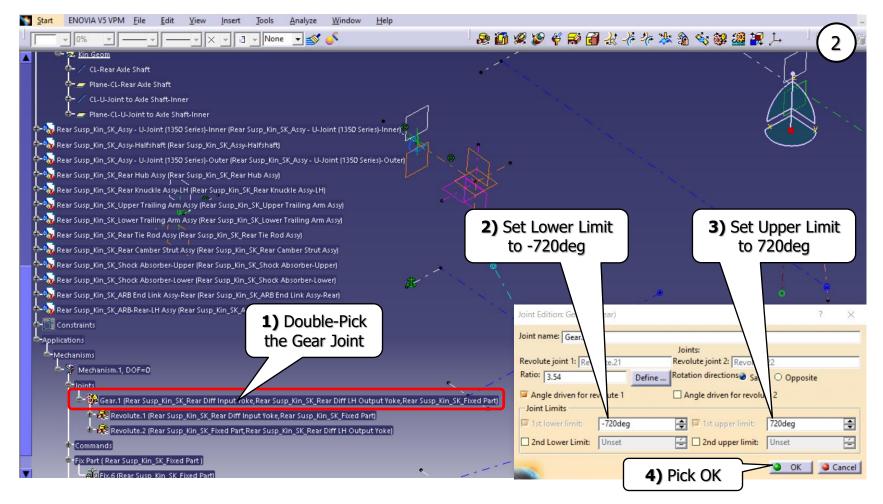
 Create a Gear Joint between the Diff Input Yoke and the Diff LH Output Yoke.

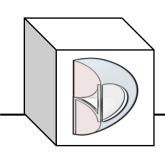






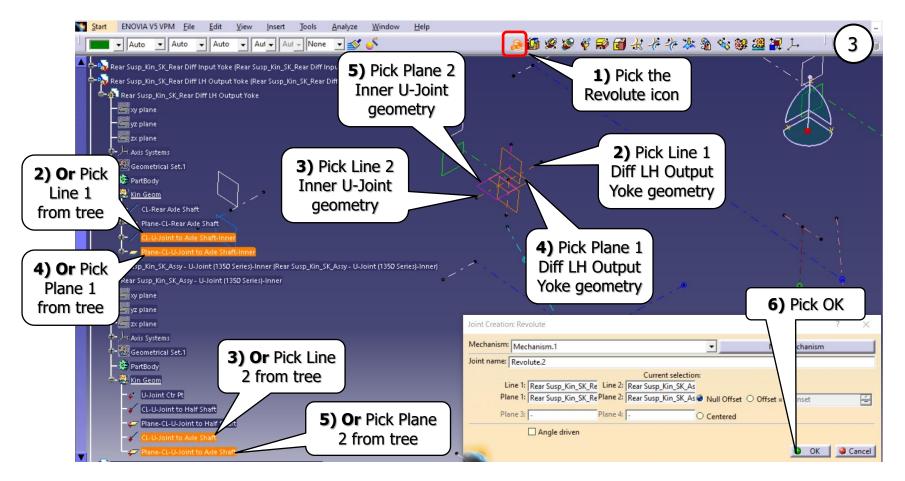
Set the Min/Max values for the Gear Joint.

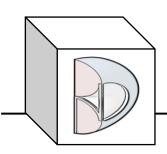






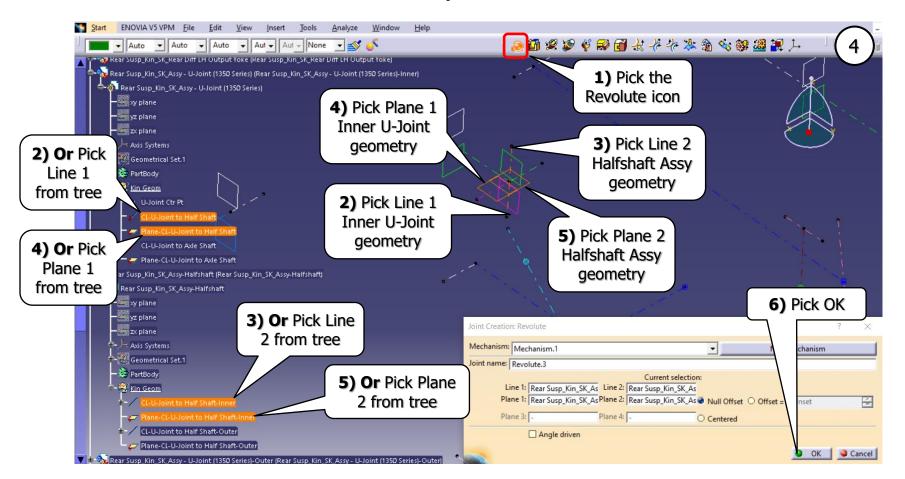
 Create a Revolute Joint between the Diff LH Output Yoke and the Inner U-Joint.

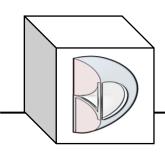






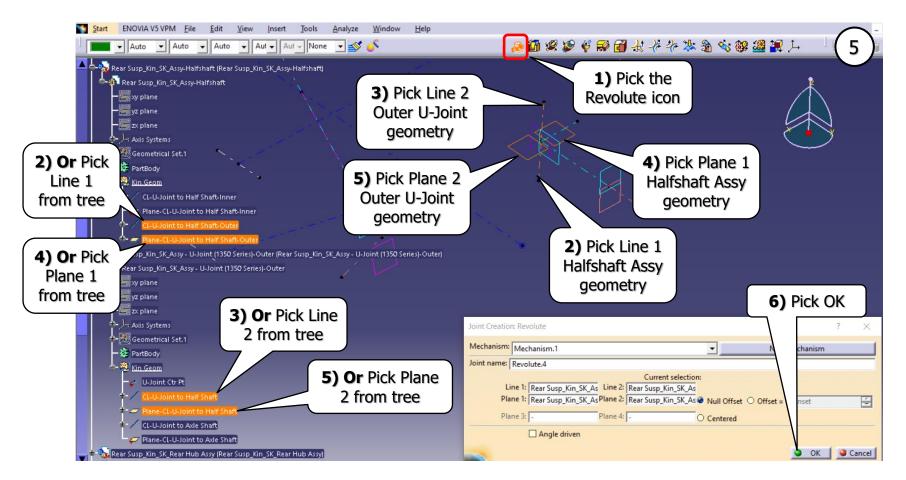
 Create a Revolute Joint between the Inner U-Joint and the Halfshaft Assy.

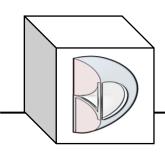






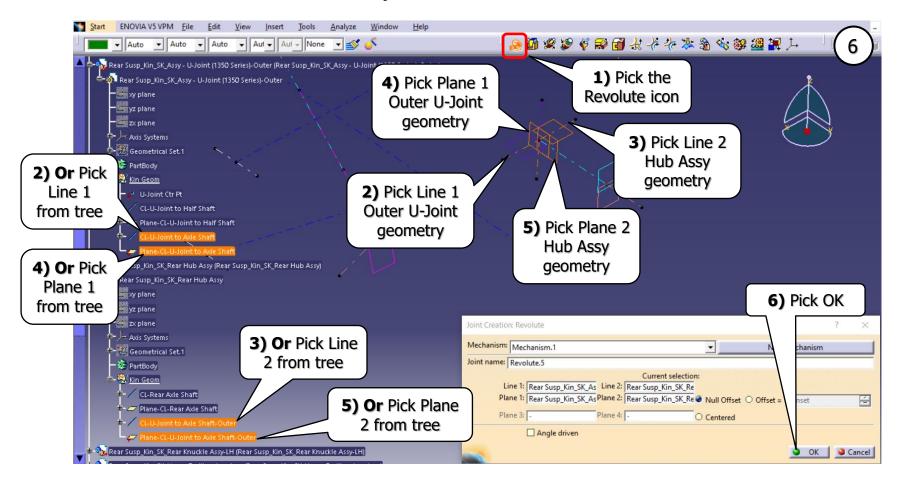
 Create a Revolute Joint between the Halfshaft Assy and the Outer U-Joint.

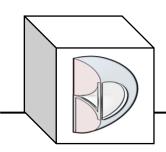






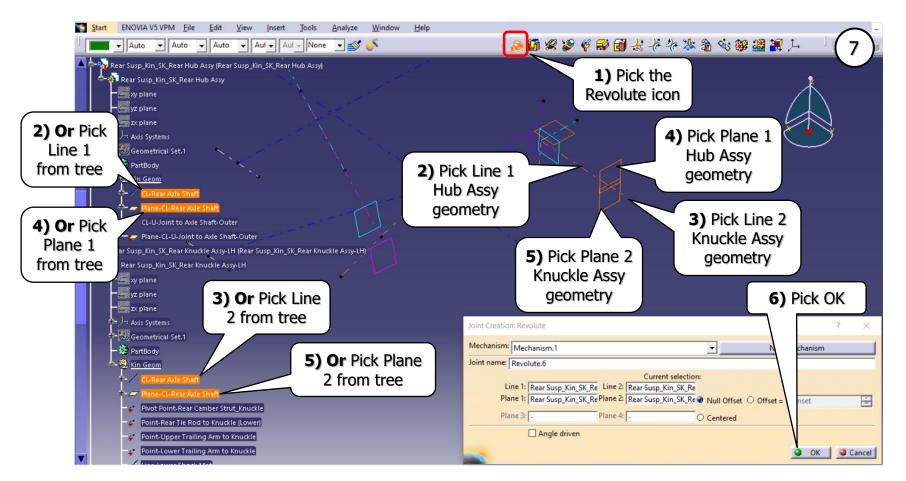
 Create a Revolute Joint between the Outer U-Joint and the Hub Assy.

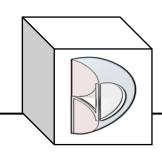






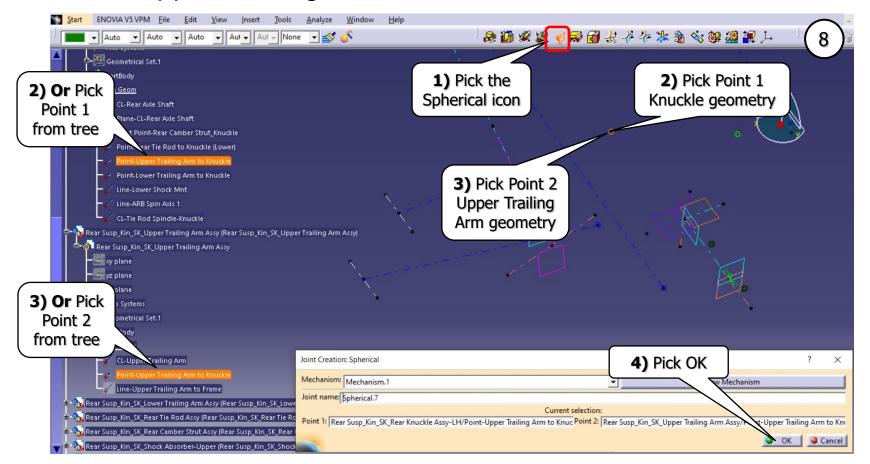
 Create a Revolute Joint between the Hub Assy and the Knuckle Assy.

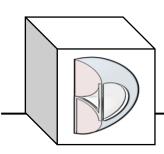






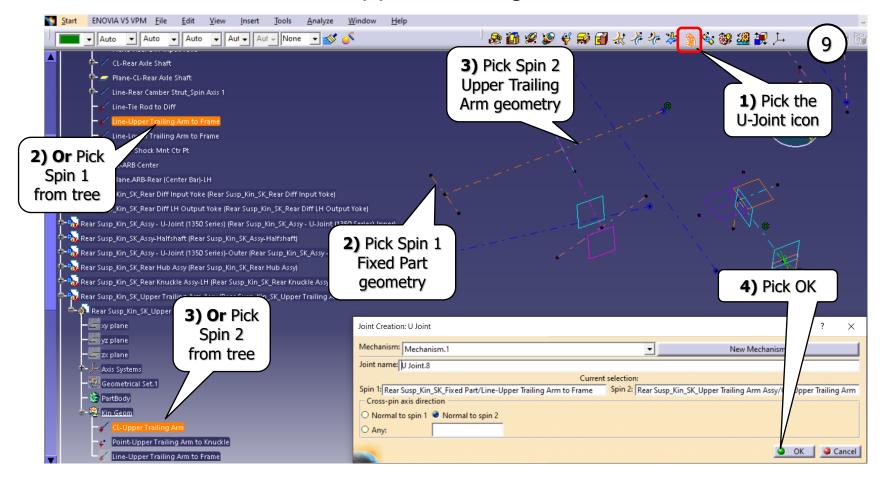
 Create a Spherical Joint between the Knuckle and the Upper Trailing Arm.

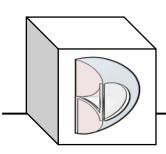






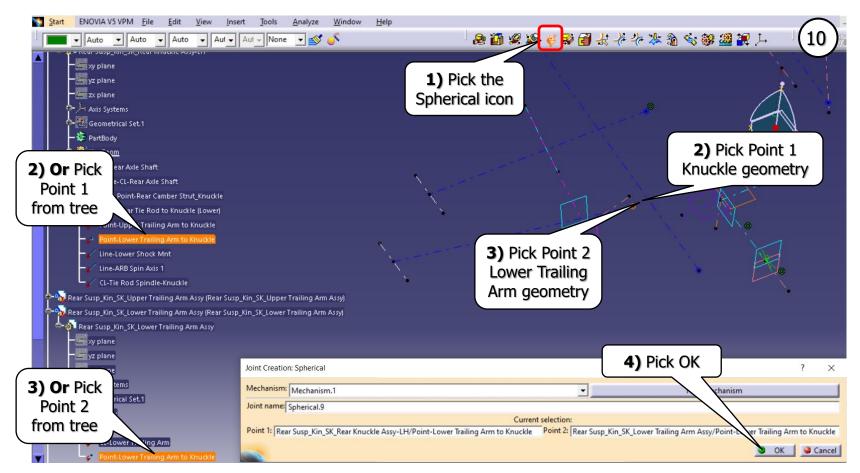
 To avoid binding in the mechanism*, create a Universal Joint between the Upper Trailing Arm and the Fixed Part.

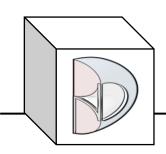






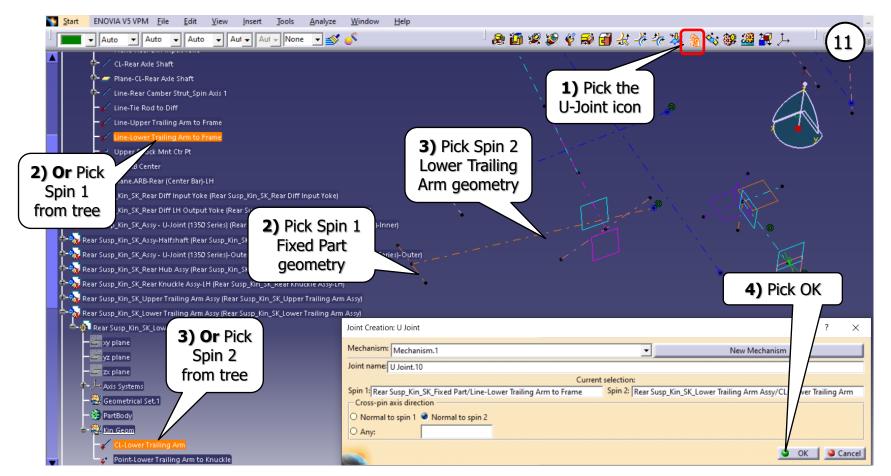
 Create a Spherical Joint between the Knuckle and the Lower Trailing Arm.

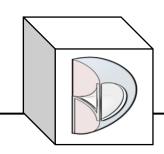






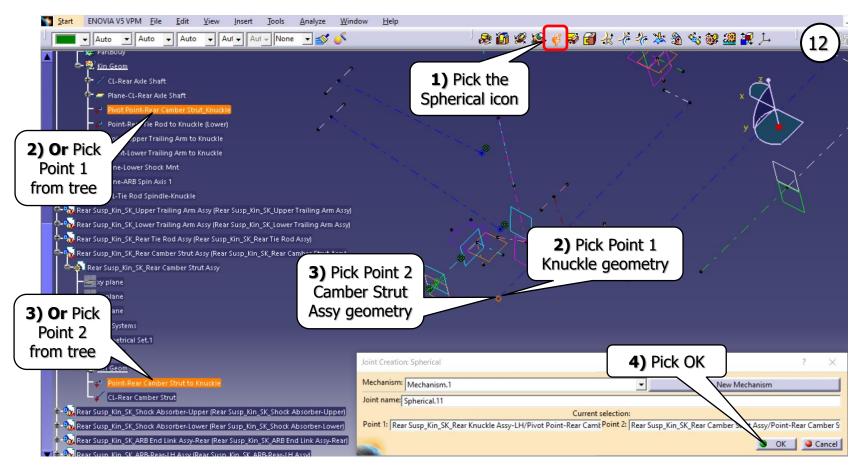
 To avoid binding in the mechanism*, create a Universal Joint between the Lower Trailing Arm and the Fixed Part.

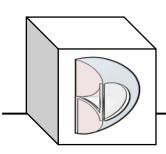






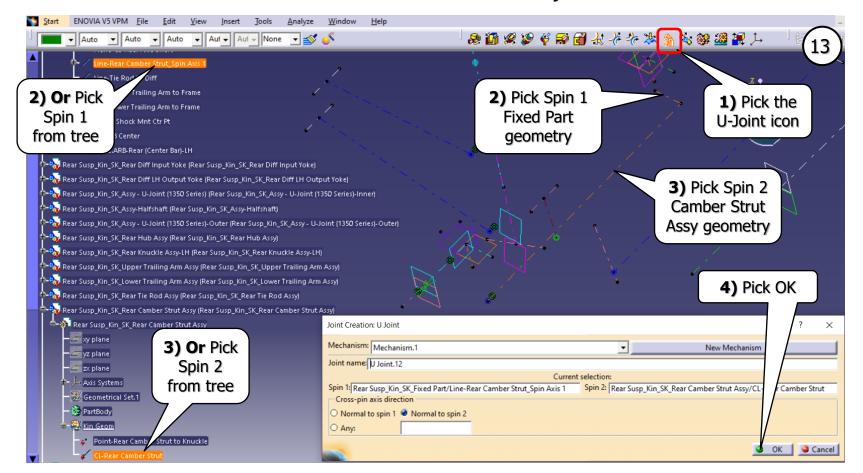
 Create a Spherical Joint between the Knuckle and the Camber Strut Assy.

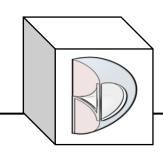






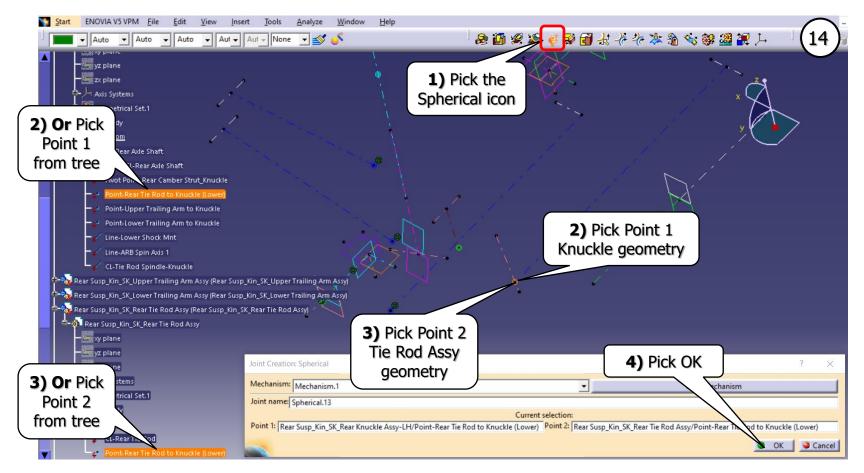
 To avoid binding in the mechanism*, create a Universal Joint between the Camber Strut Assy and the Fixed Part.

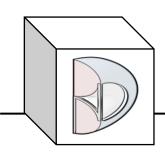






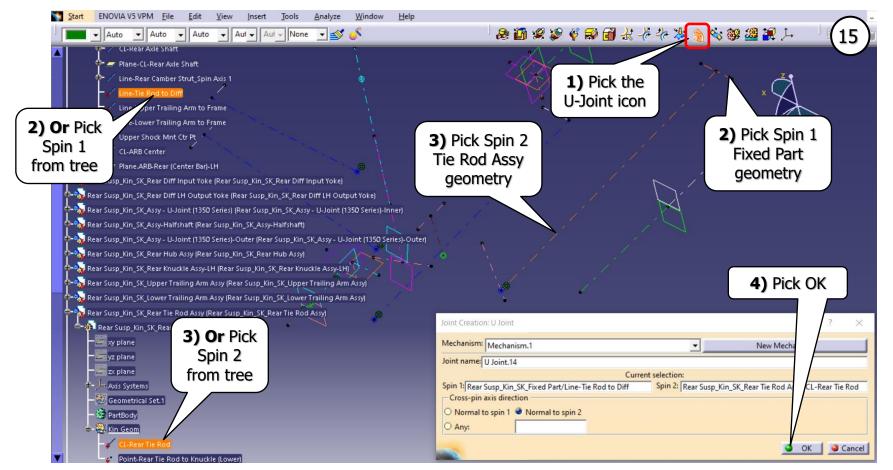
 Create a Spherical Joint between the Knuckle and the Tie Rod Assy.

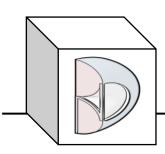






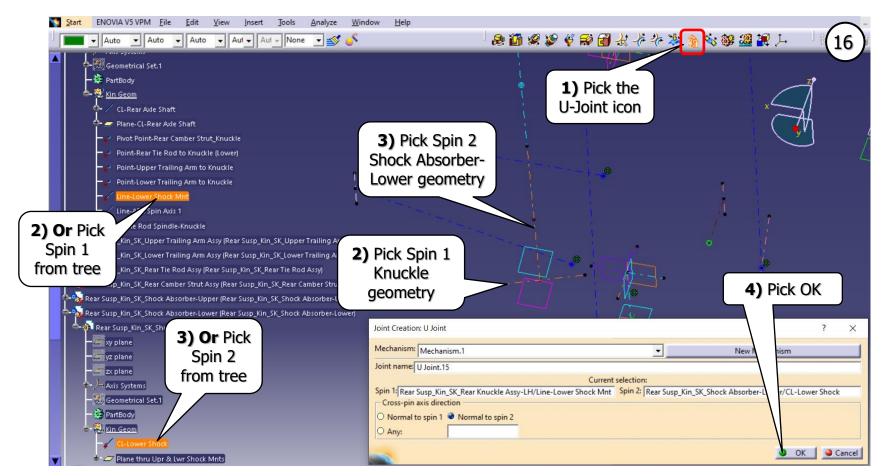
 To avoid binding in the mechanism*, create a Universal Joint between the Tie Rod Assy and the Fixed Part.

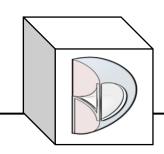






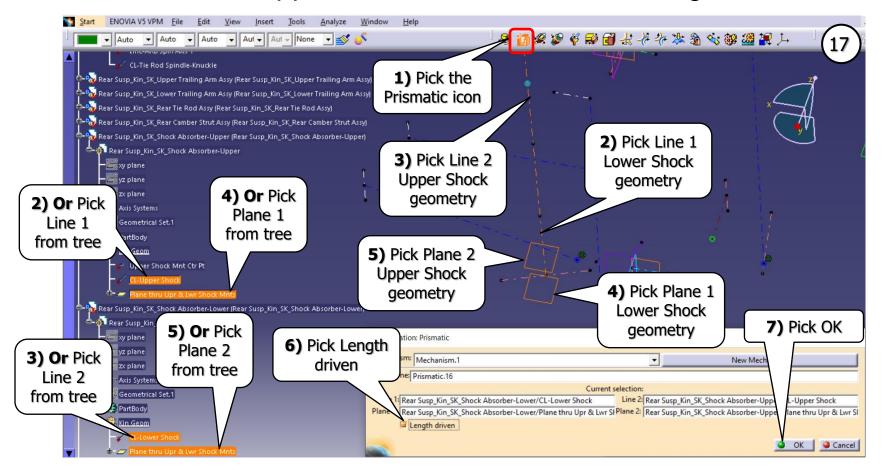
 To avoid binding in the mechanism*, create a Universal Joint between the Knuckle and the Shock Absorber-Lower.

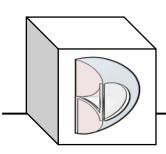






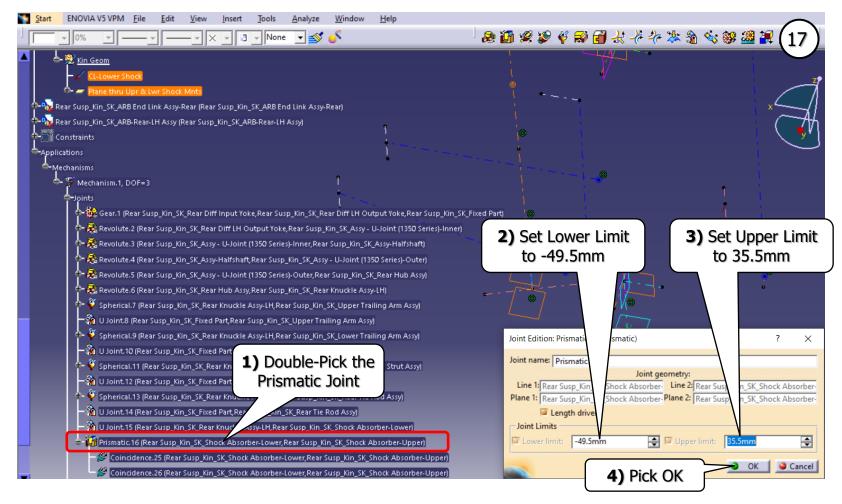
 Create a Prismatic Joint between the Lower Shock CL and the Upper Shock CL and make it Length driven.

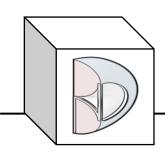






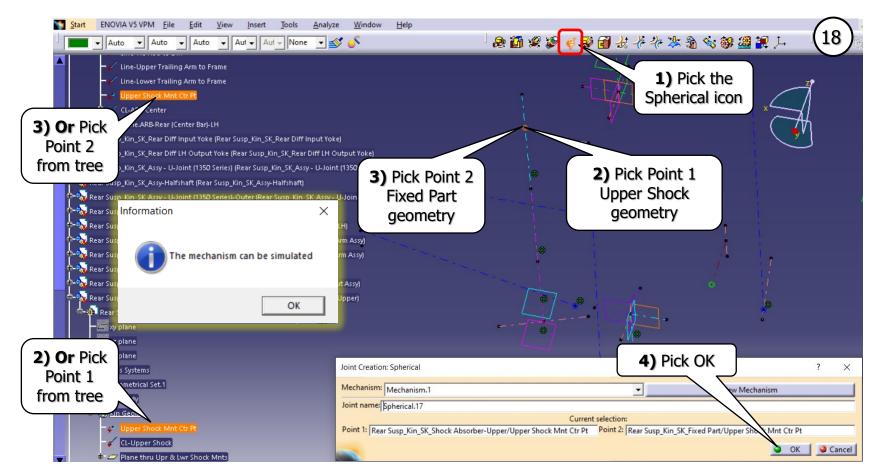
Set the Min/Max values for the Prismatic Joint.

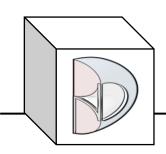






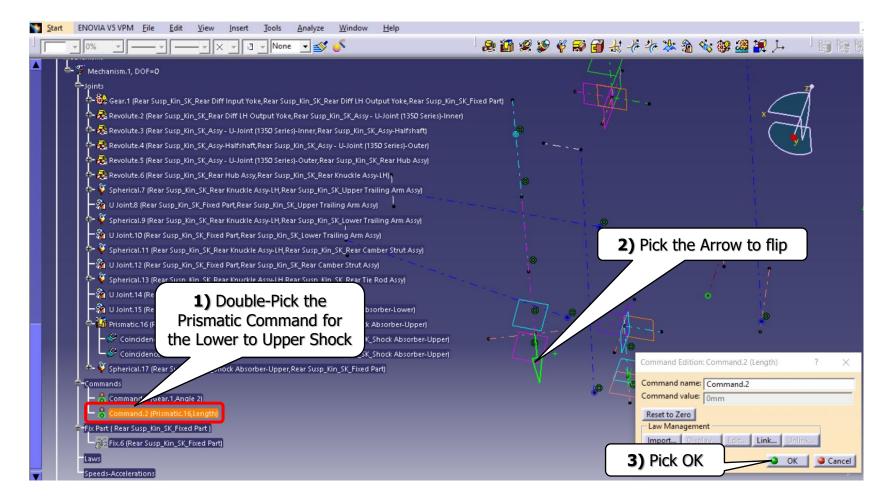
 Create a Spherical Joint between the Shock Absorber-Upper and the Fixed Part.

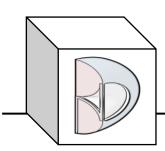






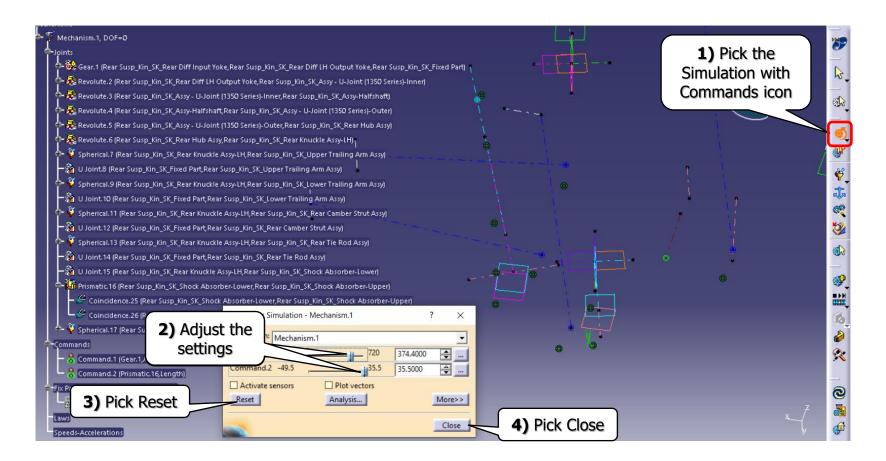
If 35.5mm is not upwards and -49.5mm not downwards:

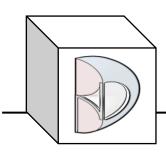






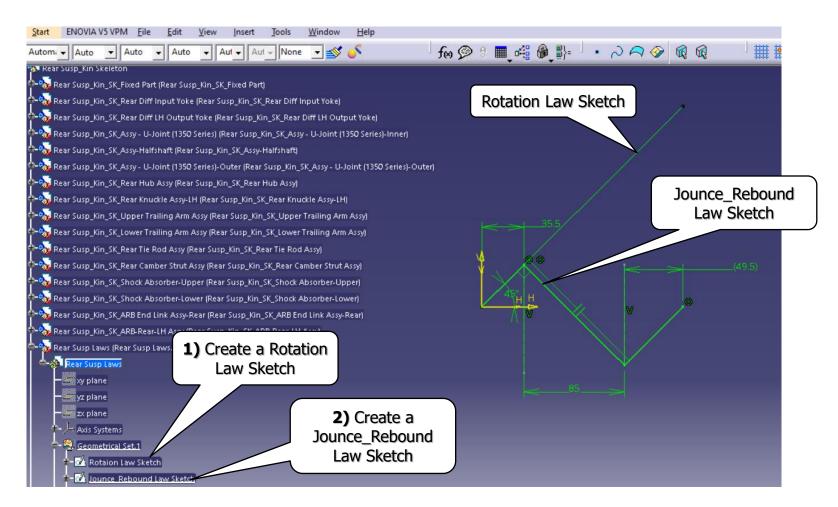
Run Simulation with Commands.

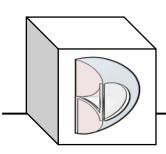






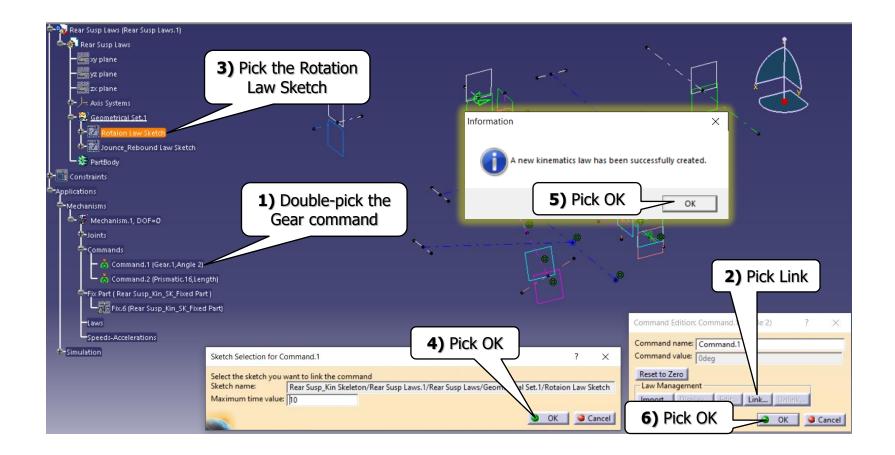
Create Laws for the mechanism commands.

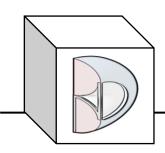






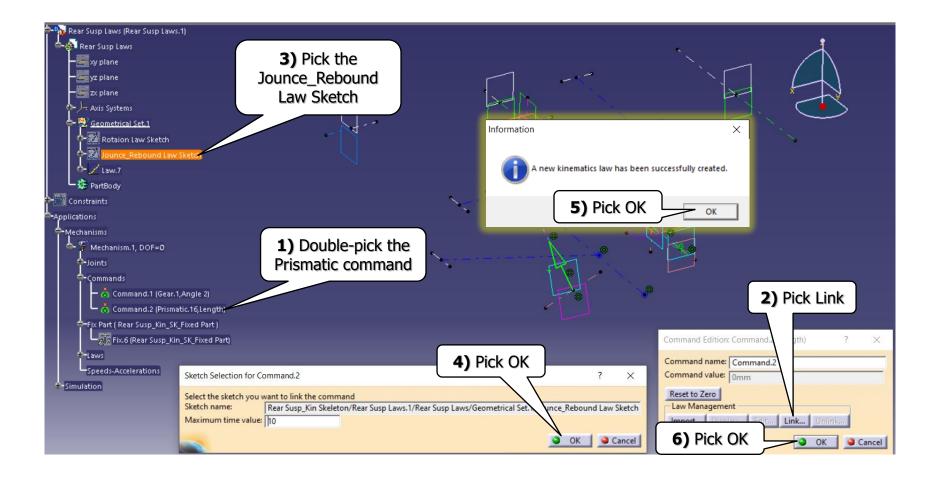
Attach the Laws to the mechanism Simulation.

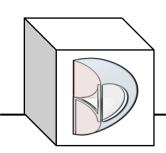






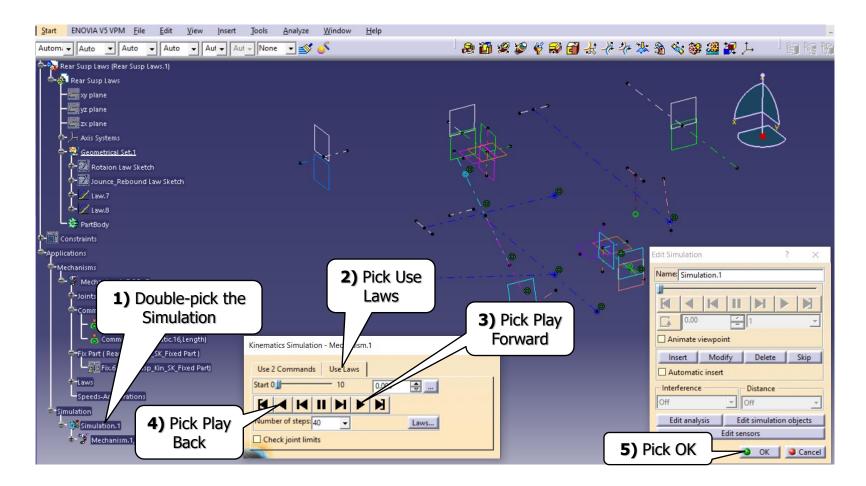
Attach the Laws to the mechanism Simulation.

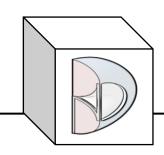






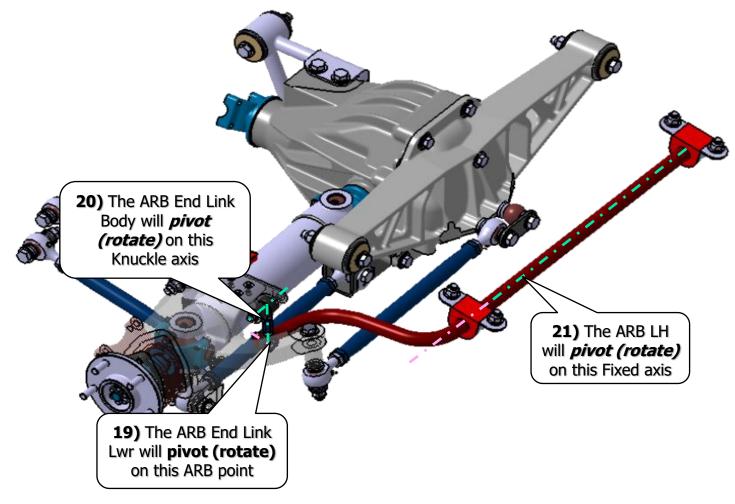
Run the Simulation with Laws.

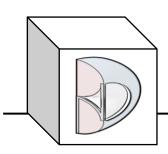






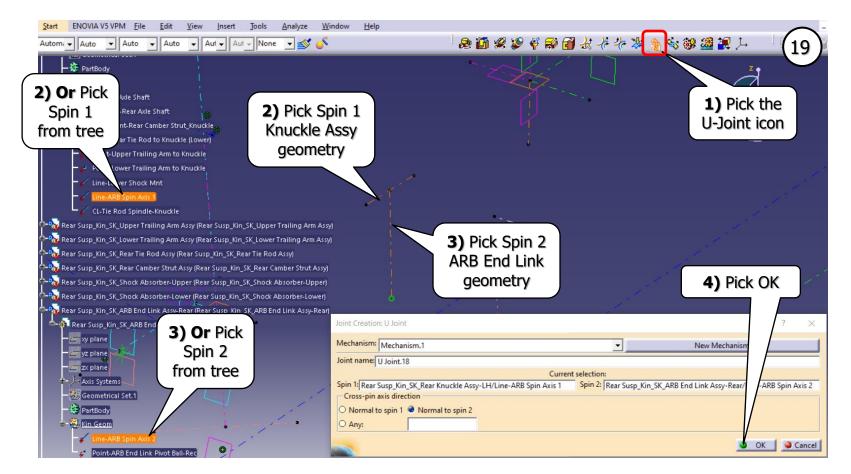
To finish the simulation, add the ARB into the mechanism.

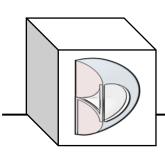






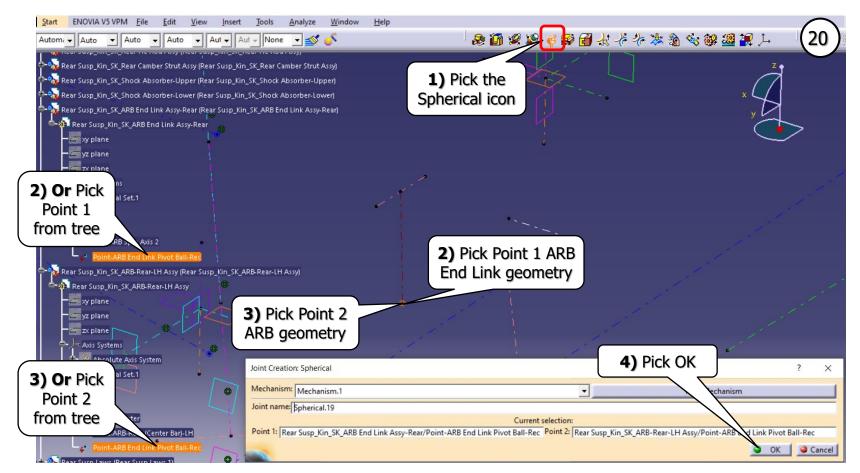
 To avoid binding in the mechanism*, create a Universal Joint between the Knuckle Assy and the ARB End Link.

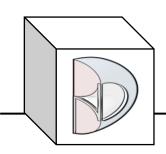






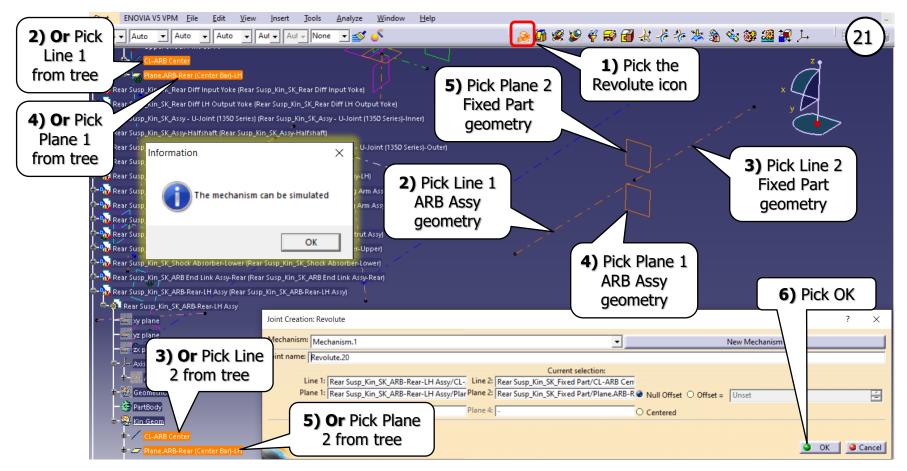
 Create a Spherical Joint between the ARB End Link and the ARB Assy.

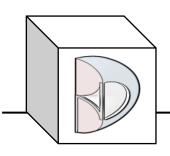






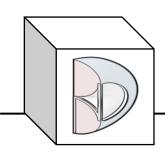
 Create a Revolute Joint between the ARB Assy and the Fixed Part.





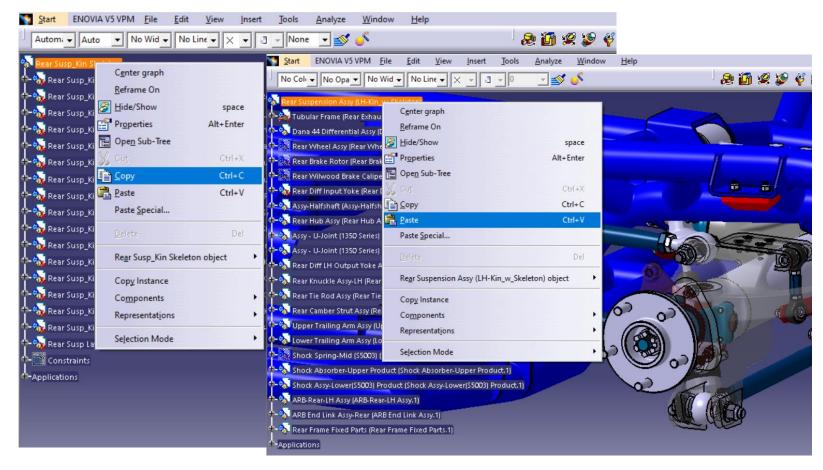


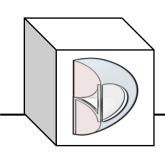
Step 4: Link the Kinematic Skeleton to the Main parts assembly via Mechanism Dressup





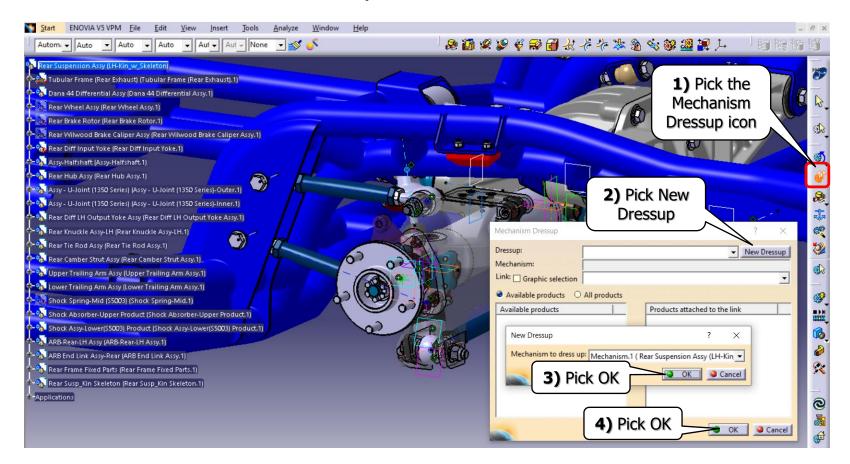
 Copy the Skeleton product and paste it into the Suspension parts assembly.

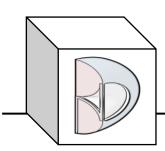






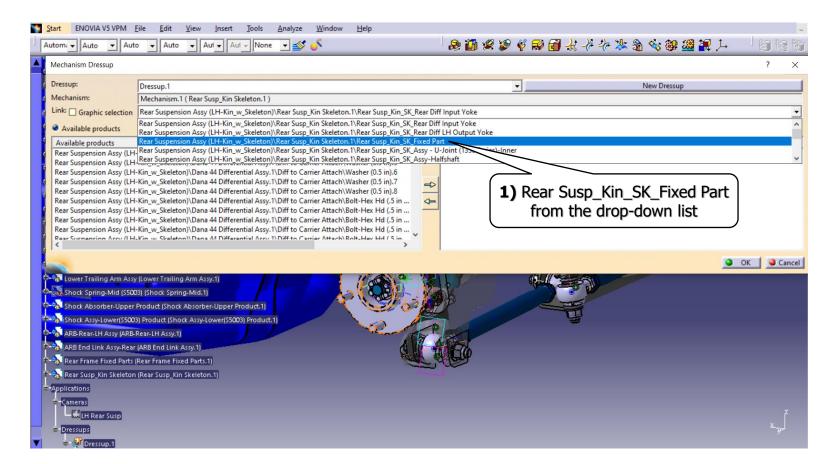
 Link the parts to the Kinematic Skeleton using Mechanism Dressup.

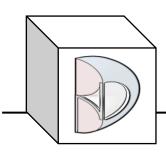






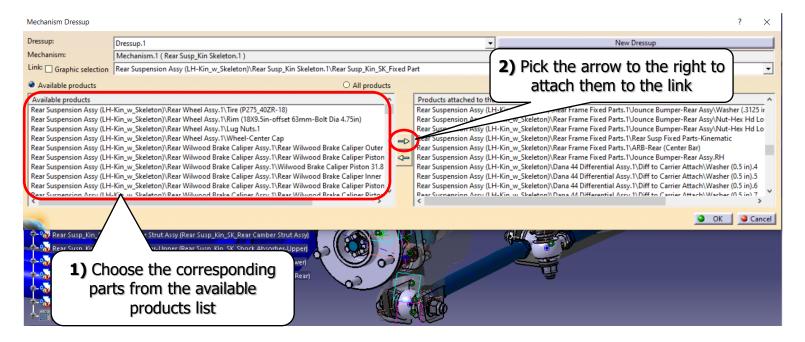
 Step A: Choose Rear Susp_Kin_SK_Fixed Part from the drop-down list.



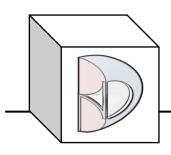




 Step B: Choose the parts from the available products list and pick the arrow to the right to attach them to the link.

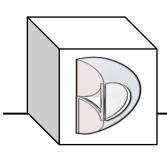


 Repeat Steps A & B until all the links on the drop-down list have been attached to the available products.



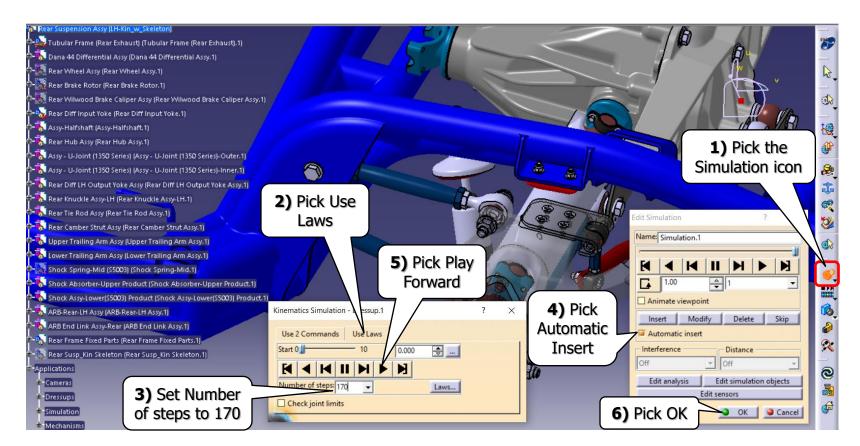


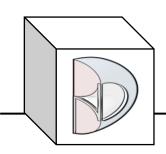
Step 5: Create and Save a Kinematic Simulation





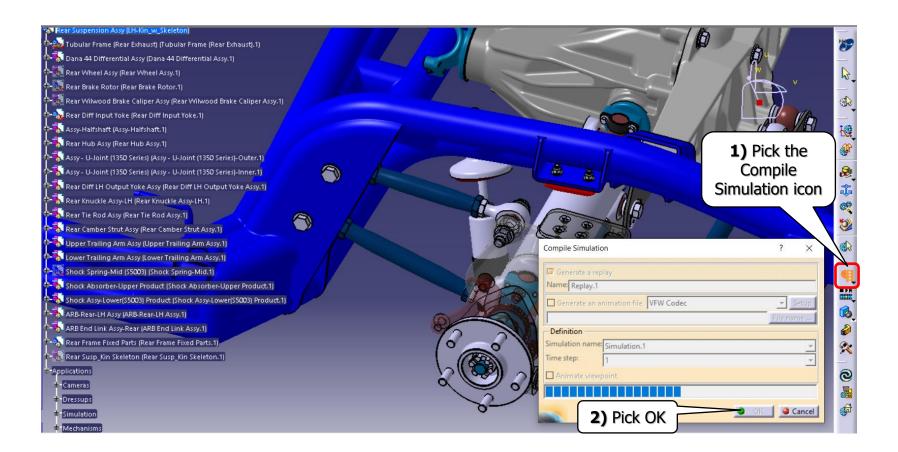
 At this point we can create a Simulation using Laws to record all parts are now moving according to the kinematic mechanism.

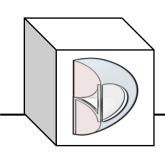






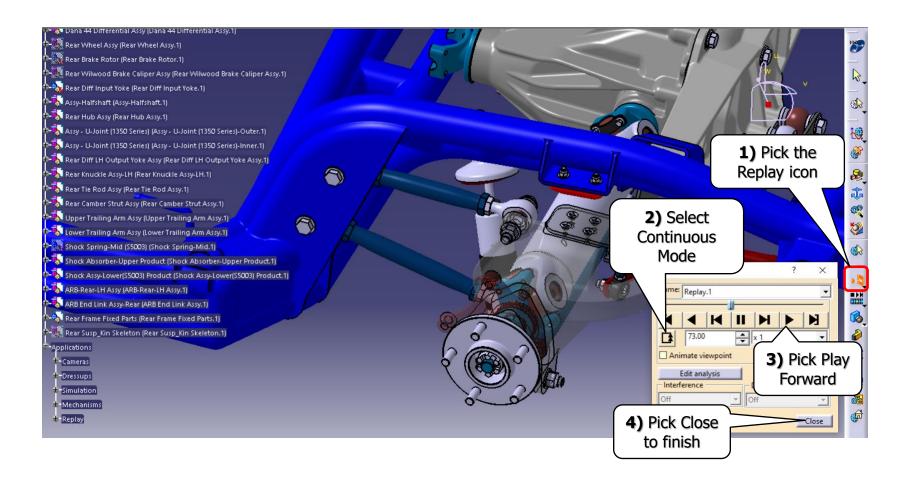
Compile a Replay of the Simulation.

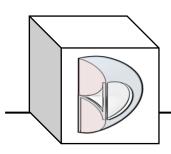






Replay the Simulation.





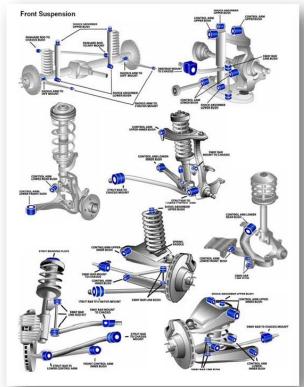


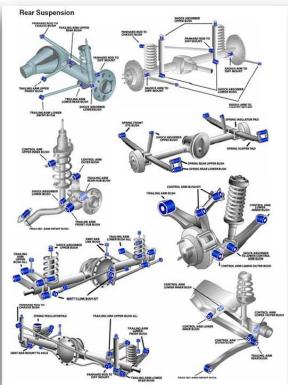
* Binding the mechanism:

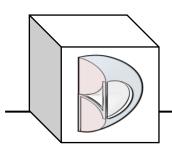
Bushing Deflection (suspension compliance)

There are many rubber bushings within a vehicle suspension to consider when analyzing suspension compliance.

CATIA DMU Kinematics does not include bushing deflection (compliance). To emulate the compliance with CATIA, we use a U-Joint at one end and a Spherical joint at the other (instead of two revolute joints). This allows for a small bit of extra movement within the mechanism to avoid binding.









Conclusion:

This is an example of how to create a CATIA DMU Kinematic simulation for a Rear Independent 5-link suspension.

We hope this will help those who need this type of simulation.

As always, we are open to any discussions this may bring.

Please *subscribe* to our YouTube channel!