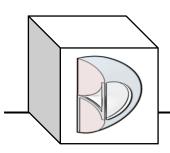


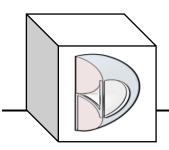


# Front Suspension Simulation using CATIA DMU Kinematics



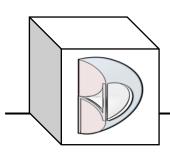


- The following licenses are required to create this DMU Kinematic simulation:
  - Digital Mockup Kinematics
  - Mechanical Part Design
  - Generative Shape Design
  - Assembly Design



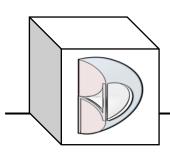


- To create this DMU Kinematic simulation, we must begin with several known parameters.
- Known:
  - All suspension "hard points".
    - Pivot points and lines
    - Angles of axes and planes
  - Min/Max Command values
    - Shock down/up (-50mm, 48mm)
    - Steering left/right (-55mm, 55mm)



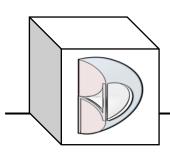


	nts	-													
Front Suspension															
Configuration:															
Scrub (Pivot) Radio	ıs =				10.0										
Steering (Kingpin)	Inclination Angle (SA	AI) =			8.8	deg									
Caster Angle =		6.5 deg	+0.4	deg	+/- 0.5										
Mechanical (or caster) trail =						mm									
Toe Angle = +0.0			+0.04	deg	+/- 0.10	deg									
Camber Angle = -(				deg	+/- 0.5										
	Ball Joint Pivot Poir	nts relative to Suspe	nsion Ana	alysis Axis	s										
Upper (Double wisl	hbone, Multi-link, Tr	ailing Arm, etc.) = (x	, y, z)												
Upper Ball Joint =			Х	Y	Z	mm									
Opper Ball Joint =			-17.51	663.90	468.99	111111									
Lower (Double wisl	Lower (Double wishbone, Multi-link, Trailing Arm, etc.) = (x, y, z)														
D 11.1.1		Χ	Y	Z											
Lower Ball Joint =			16.11	709.94	171.58	mm									
Lower only (MacPherson strut) = (x, y, z)					n,	/a									
Upper Strut Attachment/Pivot Point (MacPherson strut) = (x, y, z)				n,			-								



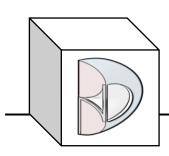


Control Arm Chassis Attachment Points relative	-								
Forward Upper (Double wishbone, Multi-link, Trailing Arm,									
UCA Fwd Mnt =	Х	Υ	Z	mm					
	125.23	408.97	490.14	mm					
Rearward Upper (Double wishbone, Multi-link, Trailing Arn	n, etc.) = (	k, y, z)							
UCA Rwd Mnt =	Х	Υ	Z	mm					
	-158.38	394.43	431.36	mm					
Forward Lower (Double wishbone, Multi-link, Trailing Arm,	etc.) = (x,	y, z)							
LCA Fwd Mnt =	Х	Υ	Z						
LCA FWG IVITE =	161.41	330.47	171.45	mm					
Rearward Lower (Double wishbone, Multi-link, Trailing Arn	n, etc.) = (	k, y, z)							
LCA Rwd Mnt =	Х	Υ	Z	mm					
LCA RWG WITH =	-223.59	330.47	171.79	111111					
Fwd Lower only (MacPherson strut) = (x, y, z)		n,	/a						
Rwd Lower only (MacPherson strut) = (x, y, z)	n,	/a							



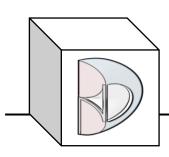


Knuckle/Brakes/Steering											
Llub Dimensions	circle dia,	etc.)									
Hub Dimensions:	Hub Dimensions: See CAD Data										
Brake Rotor Dimen	sions:										
Rotor Thickness :	=			32.0	mm						
Rotor Diameter =				325.0	mm						
Inner Face of Hul	to Inner Rotor Surface =			38.0	mm						
Inner Hub Diamet	er =			190.0							
Outer Hub Diame	ter =			217.0							
Center Hole Diam	neter =			70.0							
Brake Caliper to W	heel (min clearance) =			6.5							
Steering Rack Cen	terline relative to Susp Analysis Axis = (	x, y, z)									
Steering Rack		Х	Υ	Z	- mm						
Centerline =		124.11	n/a	273.78							
Tie Rod Pivot Poin	t (inboard) relative to Susp Analysis Axis	s = (x, y, z)	)								
Tie Rod Pivot Pt		X	Υ	Z	mm						
(inboard) =		124.11	370.00	273.78	mm						
Tie Rod Pivot Point (outboard) relative to Susp Analysis Axis = (x, y, z)											
Tie Rod Pivot Pt		Х	Υ	Z	mm						
(outboard) = 120.10 724.65					mm						



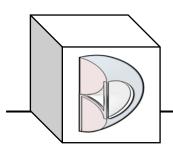


	-			-	-	 -					
Caring Dimensional	Fround ends,	etc.)									
Spring Dimensions:	Se	e CAD Da	ata								
Spring Rate =	87.5	N/mm									
Shock Absorber Adjustable [	Damping (see chart)										
Shock Absorber Type:	Strange - Double Adjustable	Coil-over	- S5004								
Extension Length =				351.50	mm						
Compressed Length =	Compressed Length =										
Stroke Length (without bump	stop) =			98.04	mm						
Bump Stop (length) =				14.30							
Shock Mounting Pivot Points	relative to Susp Analysis Axis	= (x, y, z)									
Charle I Innay Divert Daint		Χ	Υ	Z							
Snock Opper Pivot Point =	Shock Upper Pivot Point =		495.07	464.33	mm						
Shock Lower Pivot Point =		Х	Υ	Z							
		12.43	646.20	204.22	mm						
Spring and Shock Installation Angle =					deg						
Spring and Shock Absorber Motion Ratio @ Ride Ht =					unit/unit						



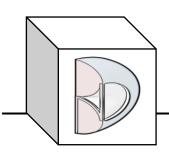


ARB (anti-roll bar)										
ARB (anti-roll bar) Dimensions:										
Virtual torque arm (A):	226.1									
Length of center (B):			812.8	mm						
Physical torque arm (C):			274.3	mm						
OD in millimeters: (D)			30.0	mm						
ID in millimeters: (if hollow)			21.0	mm						
ARB (anti-roll bar) Spring Rate =			95.2	N/mm						
ARB (anti-roll bar) Motion Ratio @ Ride Ht =	0.639	unit/unit								
ARB Bushing Rates (per deflection at loads)										
Design Results										
Tire Envelope =			See CA							
Tool Clearance =			See CA							
Routing Clearance (i.e. hoses, brake lines, fuel lines, electric	cal, etc.) =		See CA							
Roll Center Height relative to Susp Analysis Axis =			13.28	mm						
Anti-dive Instant Center relative to Susp Analysis Axis = (x	(, y, z)									
Anti dina lastant Cantar	Х	Υ	Z							
Anti-dive Instant Center =	-1405.9	n/a	172.84	mm						
Bushing Rates (charted from Suspension Analysis Softwa										
Wheel Rate (including ARB) =	89.25									
Ride Rate (including ARB) =				65.88 N/mm						



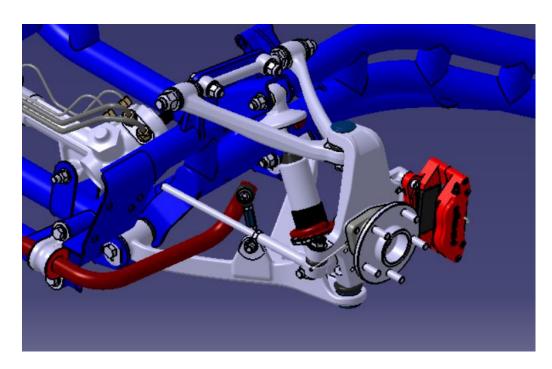


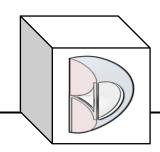
# Step 1: Understand the suspension system



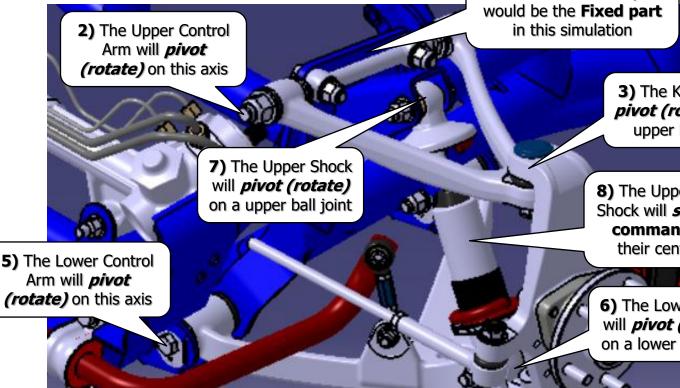


- Begin with the completed Front Suspension Assembly.
- This will help to understand all the pivot and links within the system.









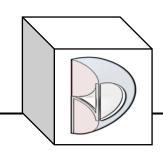
3) The Knuckle will pivot (rotate) on a upper ball joint

1) The frame and all its immoveable attached parts

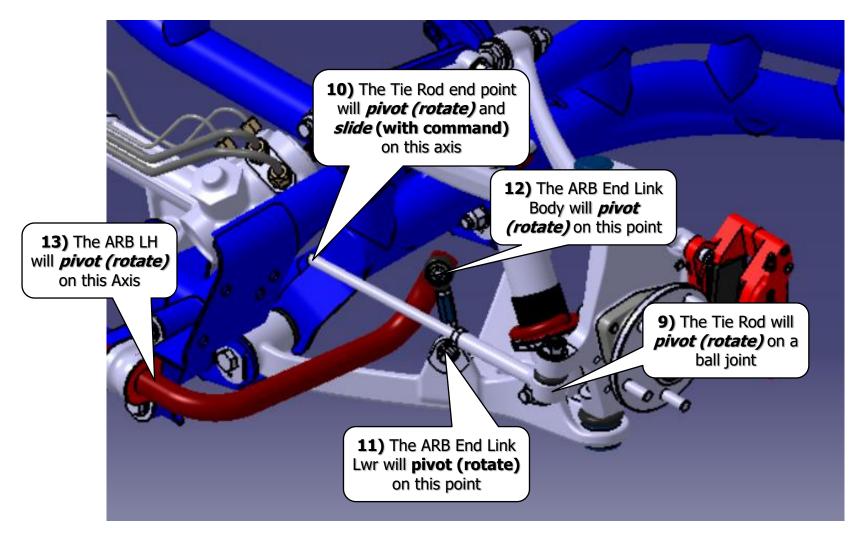
> 8) The Upper & Lower Shock will slide (with command) along their center axes

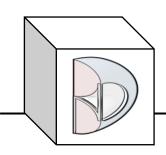
**6)** The Lower Shock will *pivot (rotate)* on a lower ball joint

> 4) The Knuckle will pivot (rotate) on a lower ball joint



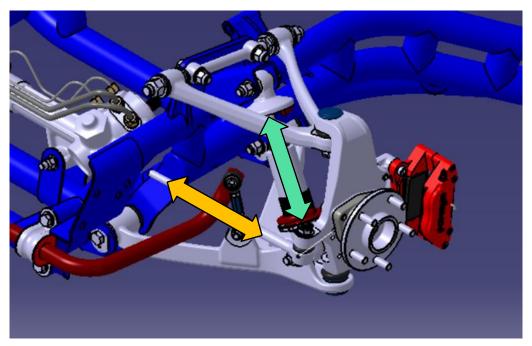


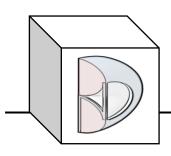






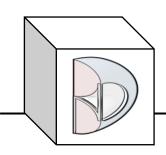
- Understand which joints need commands.
  - Min/Max Command values
    - Shock down/up (-50mm, 48mm)
    - Steering left/right (-55mm, 55mm)





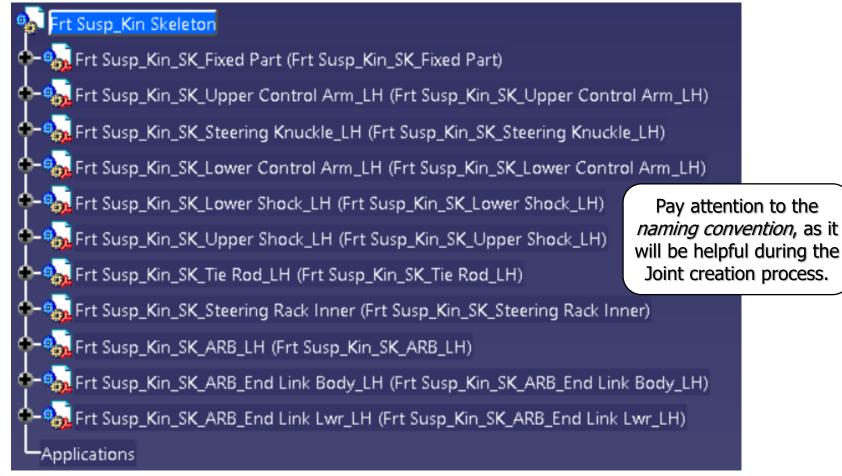


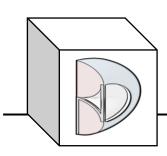
# Step 2: Create a kinematic skeleton structure





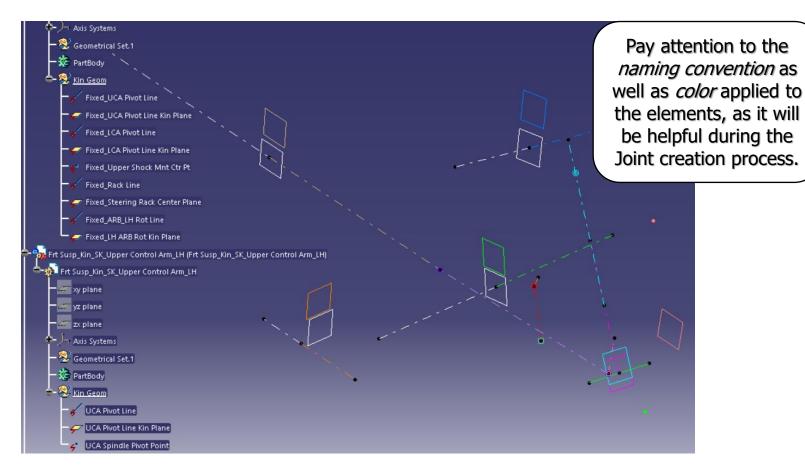
 Create a skeleton product and all parts required for the kinematic simulation.

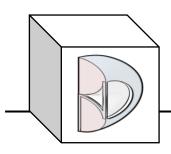






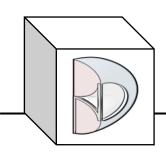
 Create all points, lines, & planes inside each part within the kinematic structure.





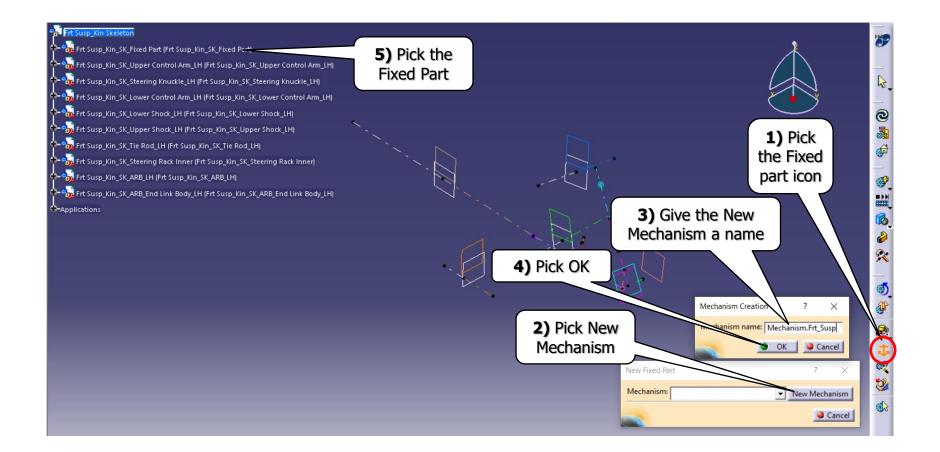


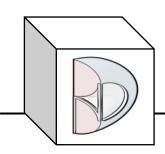
# Step 3: Create the kinematic Joints





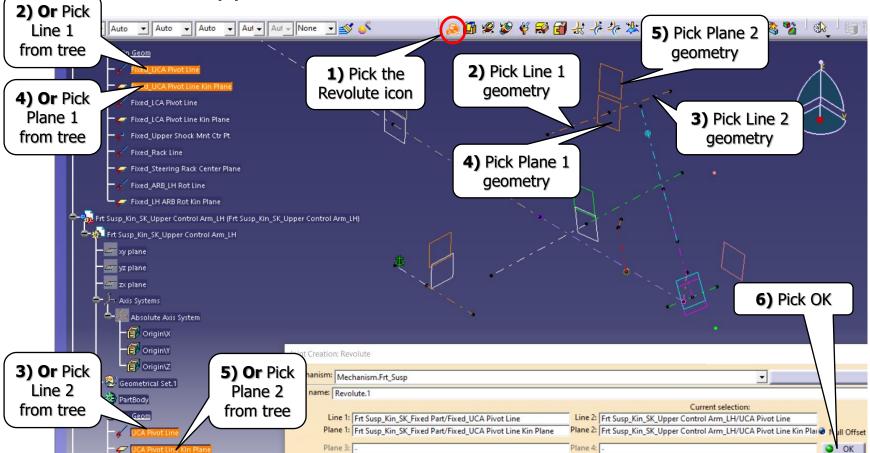
Create the Fixed Part and name the Mechanism.

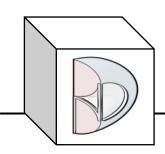






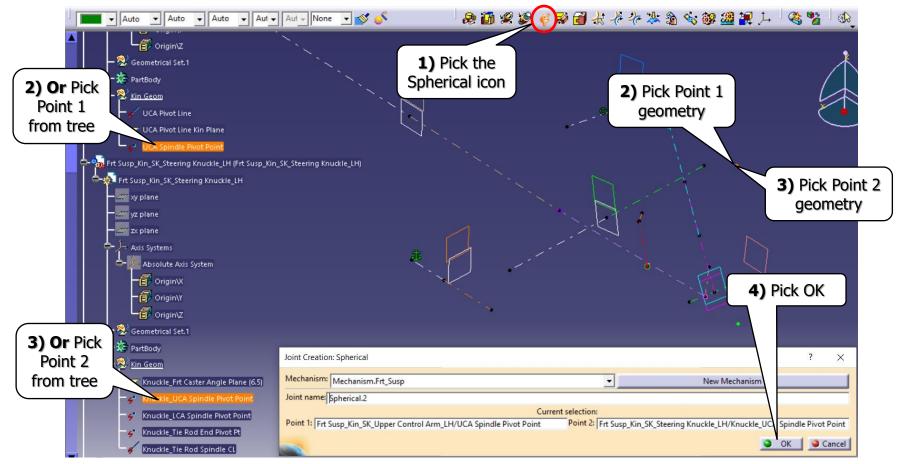
 Create a Revolute Joint between the Fixed part and the Upper Control Arm.

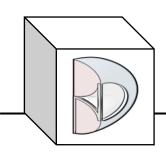






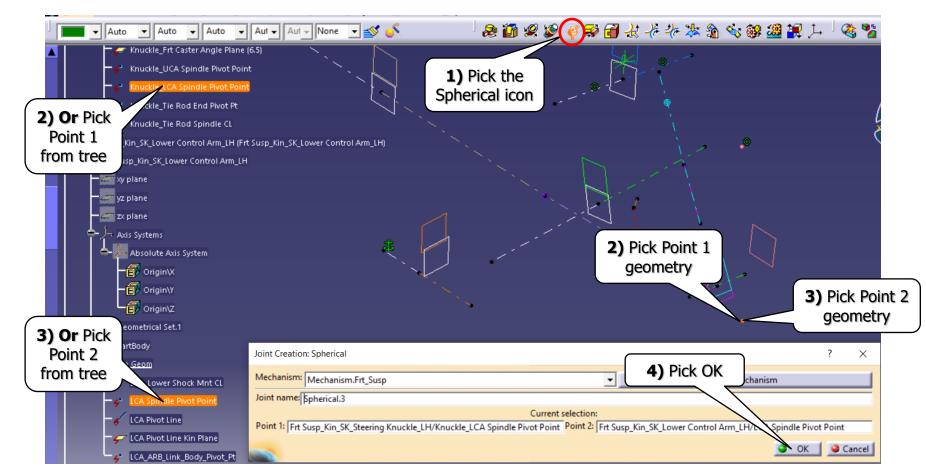
 Create a Spherical Joint between the Upper Control Arm and the Knuckle.

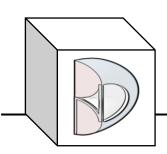






 Create a Spherical Joint between the Knuckle and the Lower Control Arm.



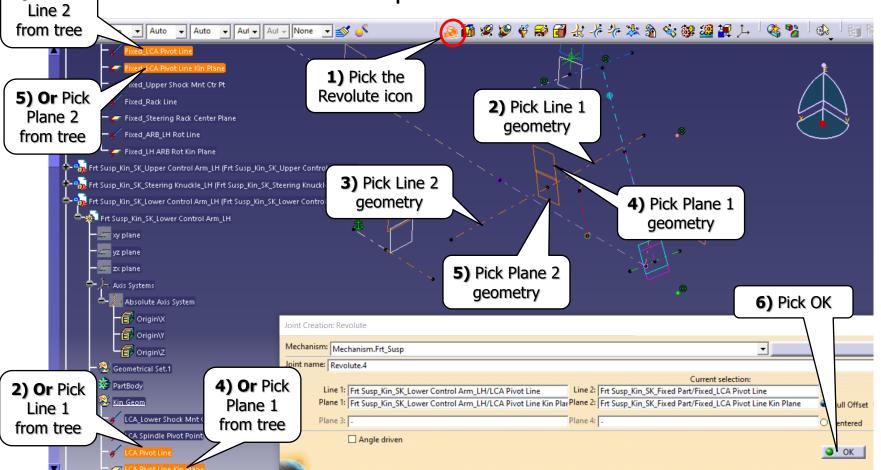


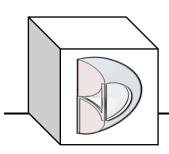
3) Or Pick

#### **BND TechSource**



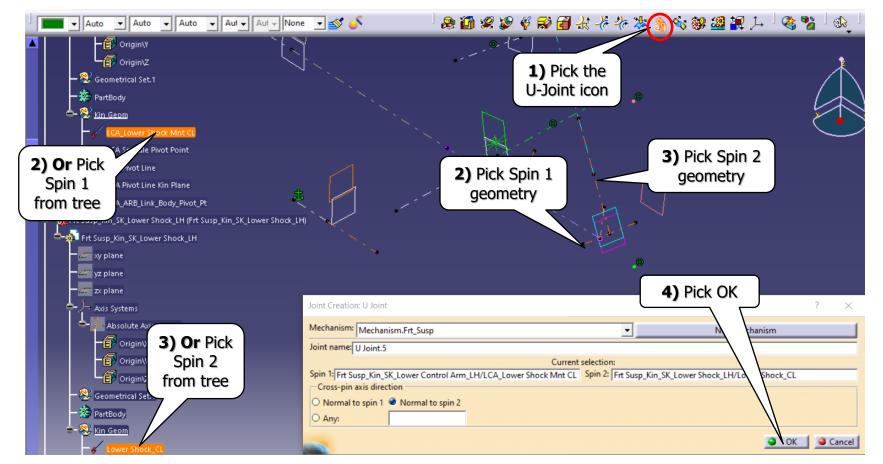
 Create a Revolute Joint between the Lower Control Arm and the Fixed part.

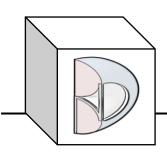






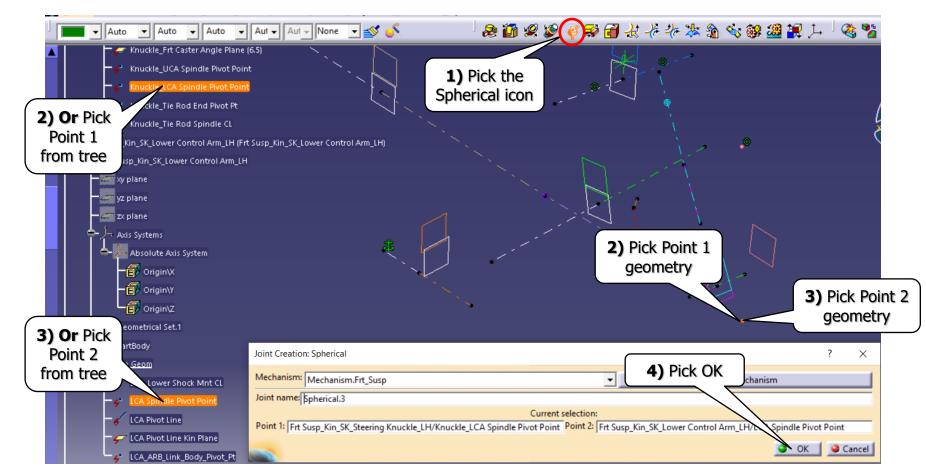
 To avoid binding in the mechanism, we will create a Universal Joint between the LCA and the Shock Lower.

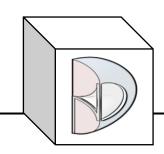






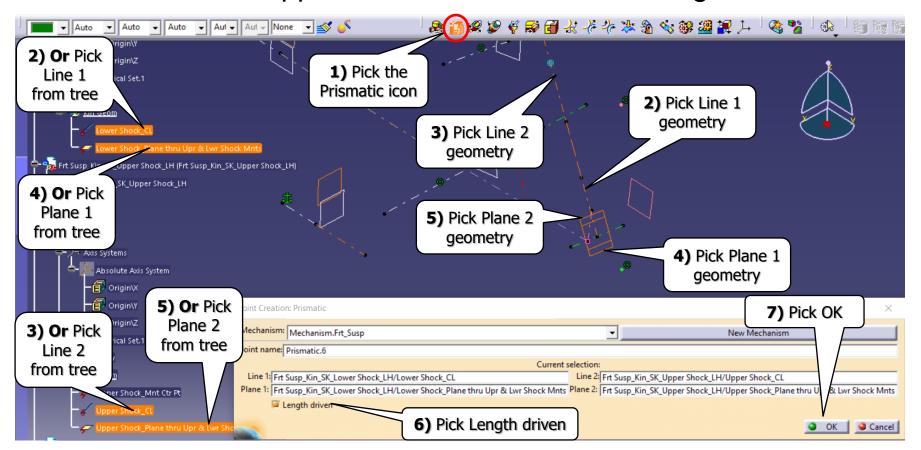
 Create a Spherical Joint between the Knuckle and the Lower Control Arm.

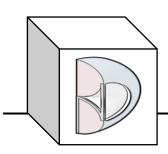






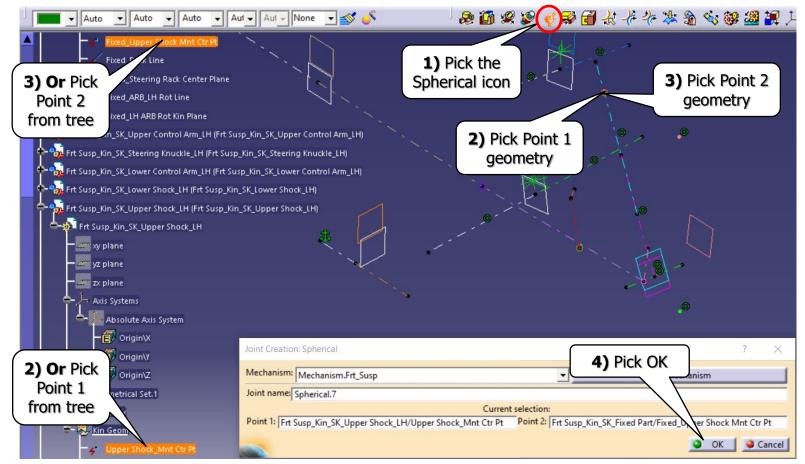
 Create a Prismatic Joint between the Lower Shock CL and the Upper Shock CL and make it Length driven.

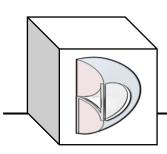






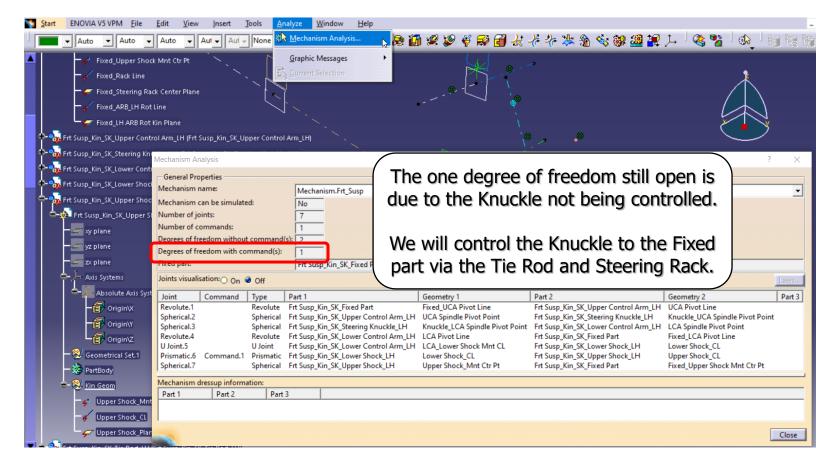
 Create a Spherical Joint between the upper Shock mounting point and the Fixed part mounting point.

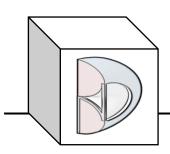






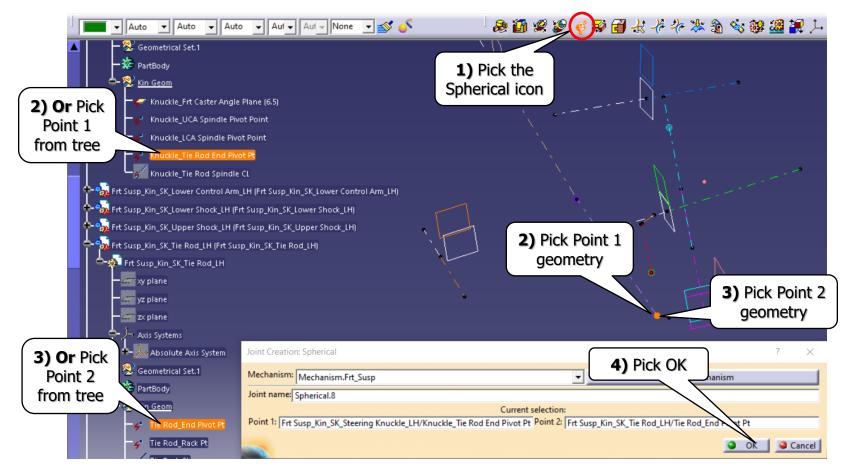
 Use Analyze + Mechanism Analysis to understand there is still one degree of freedom open.

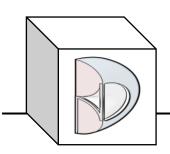






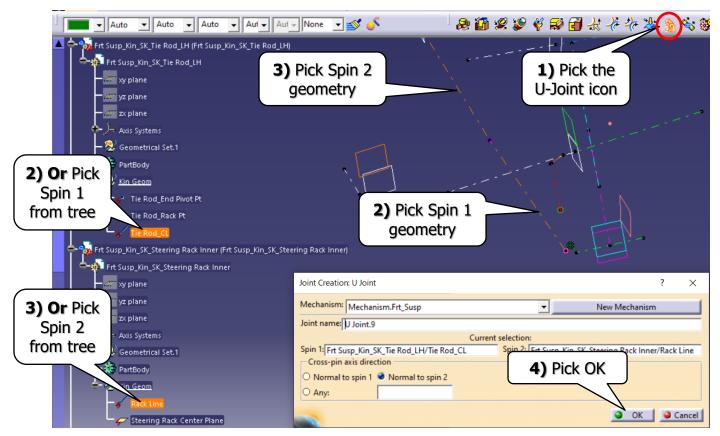
 Create a Spherical Joint between the Knuckle spindle point and the Tie Rod spindle point.

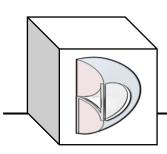






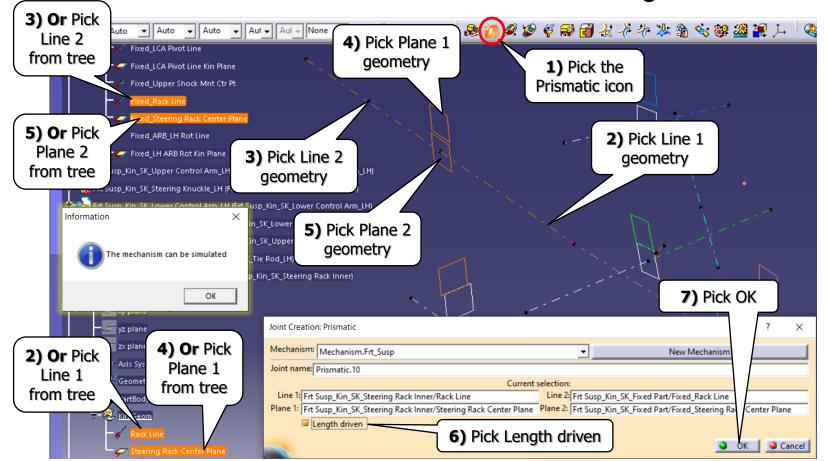
 To avoid binding in the mechanism, we will create a Universal Joint between the Tie Rod and the Steering Rack Inner.

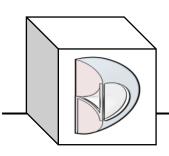






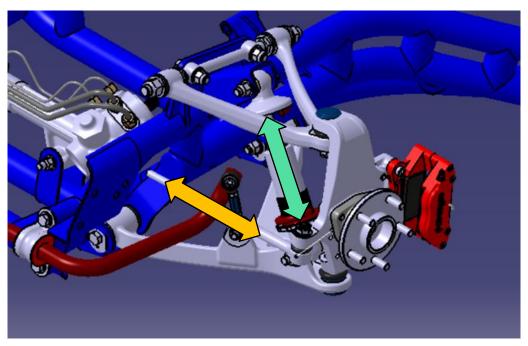
 Create a Prismatic Joint between the Steering Rack Line and the Fixed Rack Line and make it Length driven.

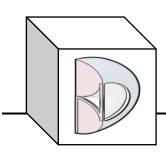






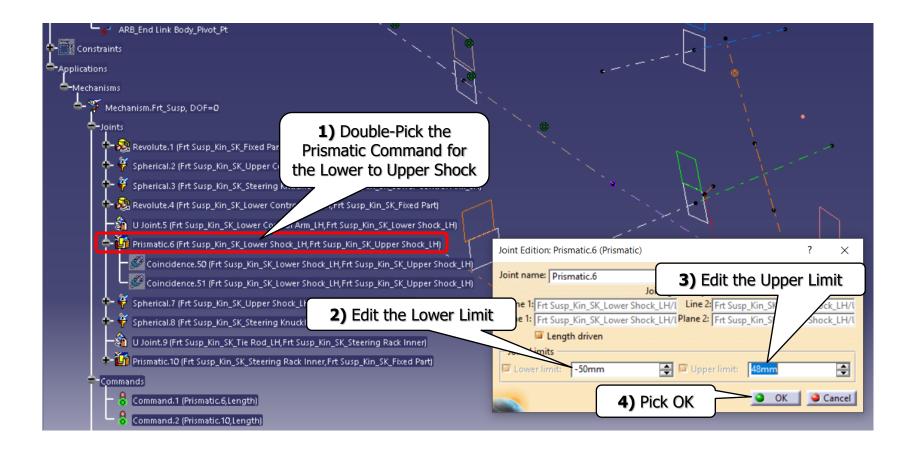
- Edit the Command values for the driven commands.
  - Min/Max Command values
    - Shock down/up (-50mm, 48mm)
    - Steering left/right (-55mm, 55mm)

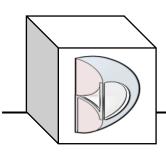






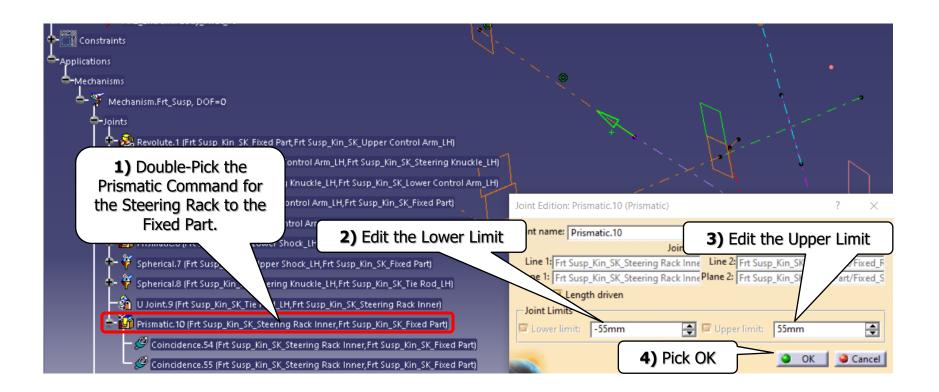
Edit the Command values for the driven commands.

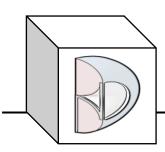






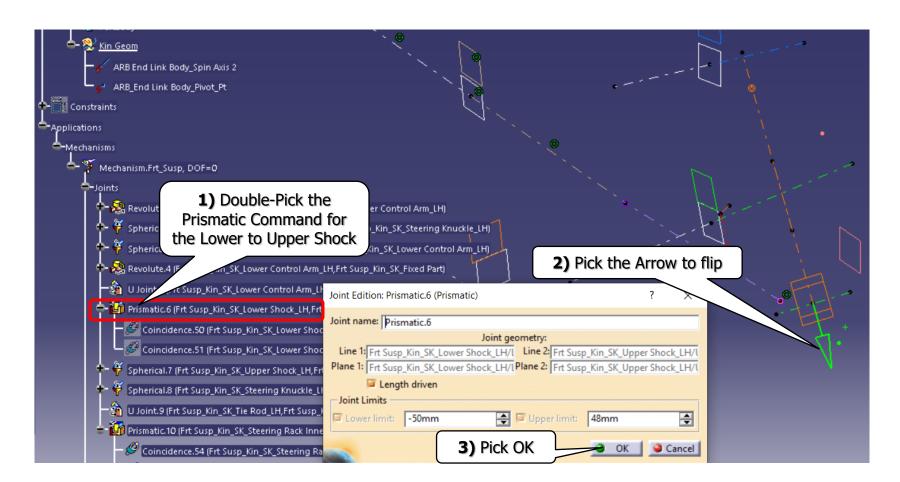
Edit the Command values for the driven commands.

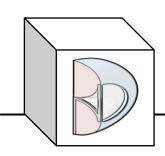






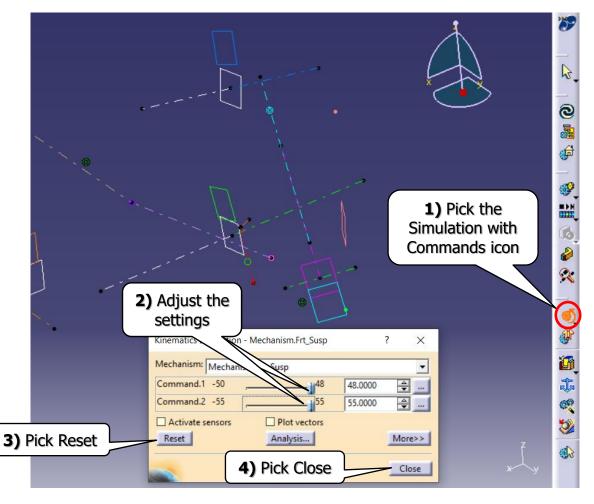
If 48mm is not upwards and -50mm not downwards:

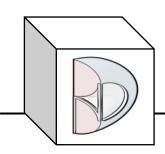






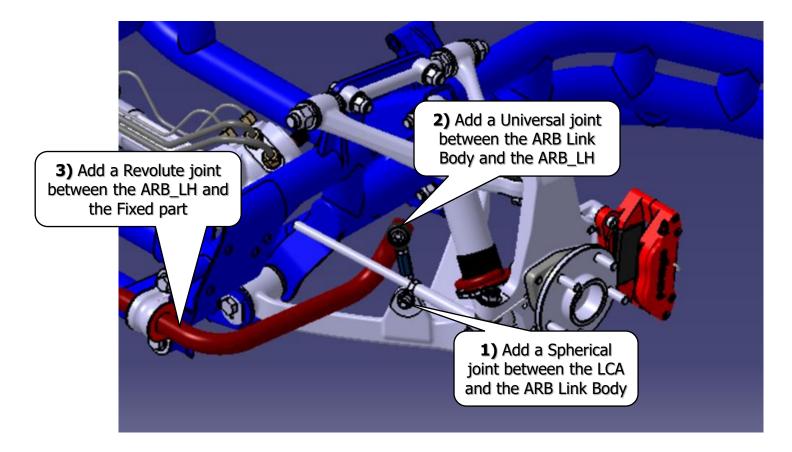
Run Simulation with Commands.

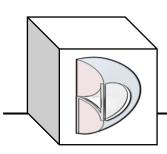






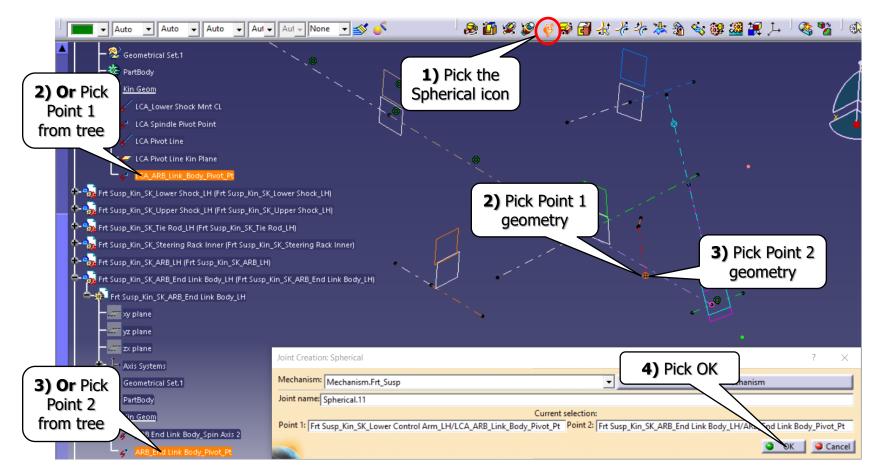
 To finish the simulation, add the ARB into the mechanism.

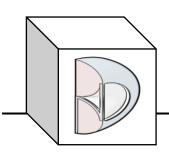






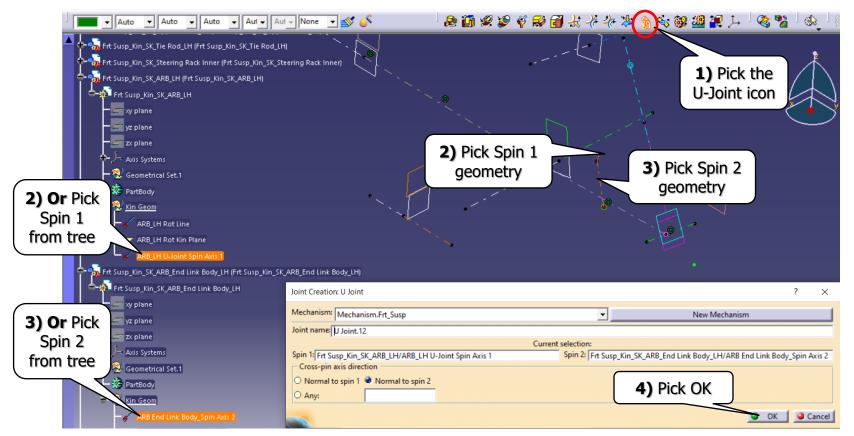
 Create a Spherical Joint between the LCA ARB Link Body point and the ARB Link Body point.

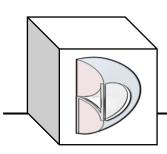






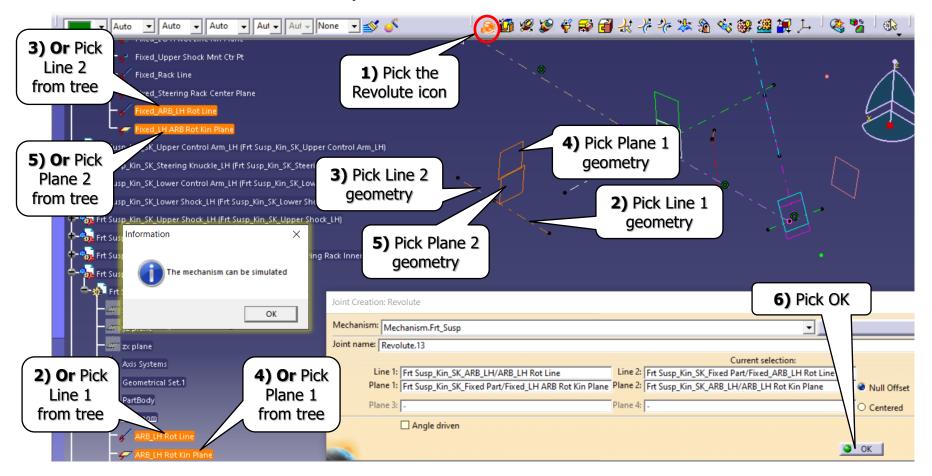
 To avoid binding in the mechanism, we will create a Universal Joint between the ARB LH Spin Axis 1 and the ARB Link Spin Axis 2.

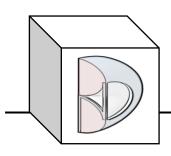






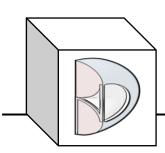
 Create a Revolute Joint between the ARB LH Rot line and the Fixed part Rot line.





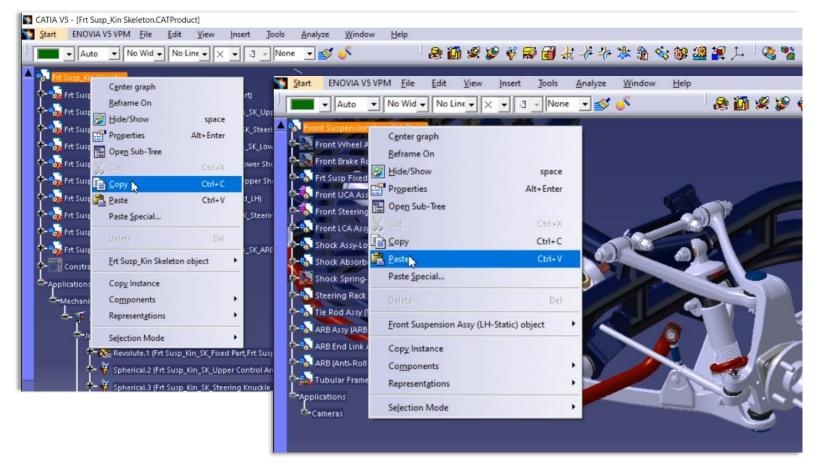


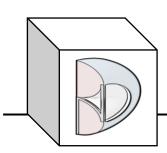
# Step 4: Link the Kinematic Skeleton to the Main parts assembly via Mechanism Dressup





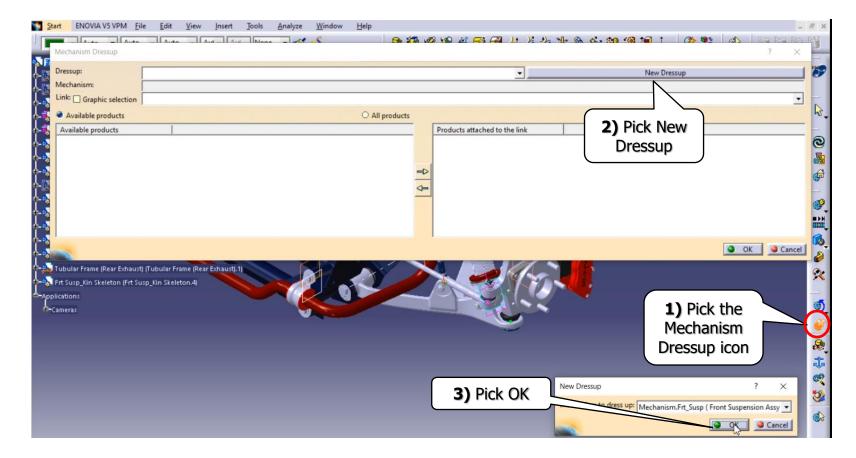
 Copy the skeleton product and paste it into the suspension parts assembly.

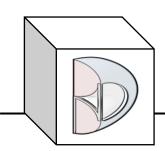






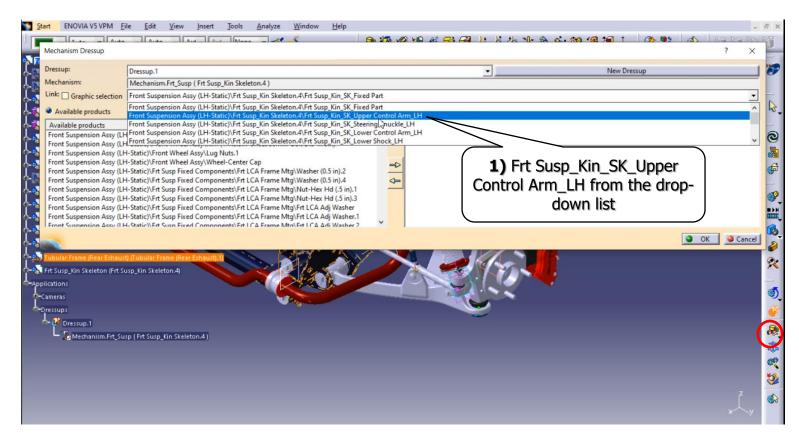
 Link the parts to the kinematic skeleton using Mechanism Dressup.

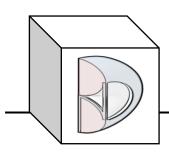






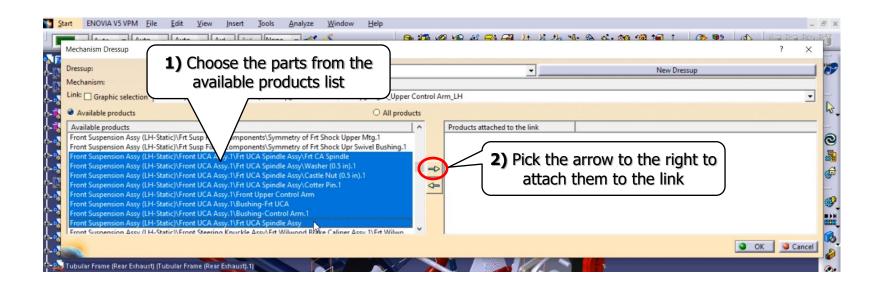
 Step A: Choose Frt Susp\_Kin\_SK\_Upper Control Arm\_LH from the drop-down list.



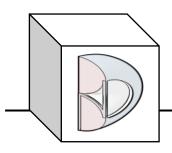




 Step B: Choose the parts from the available products list and pick the arrow to the right to attach them to the link.

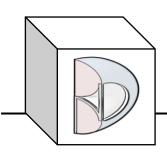


 Repeat Steps A & B until all the links on the drop-down list have been attached to the available products.



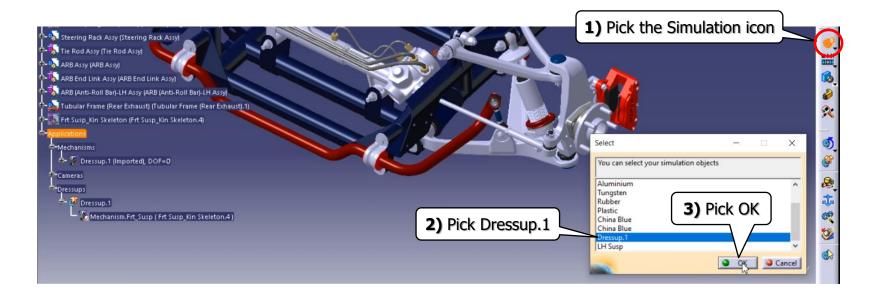


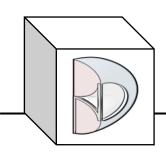
## Step 5: Create and Save a Kinematic Simulation





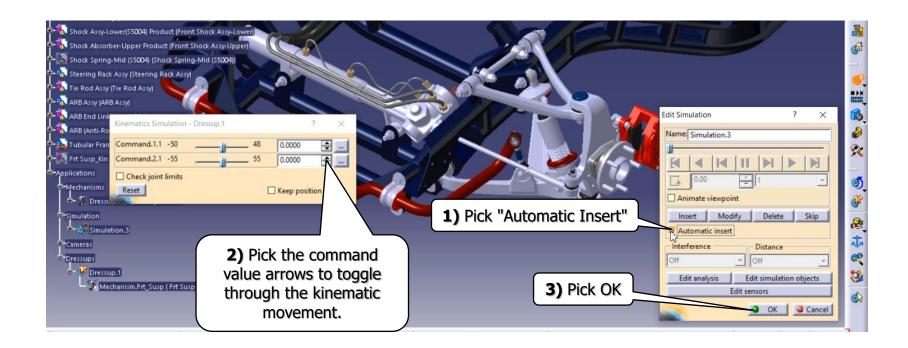
 At this point we can run a simulation and see that all parts are now moving according to the kinematic mechanism.

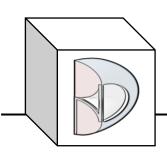






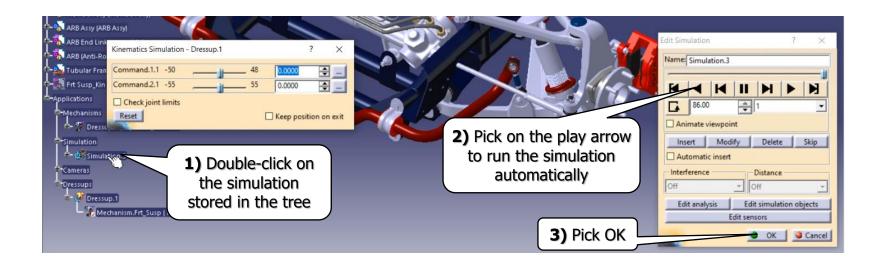
 Create a simulation which can be stored and run automatically.

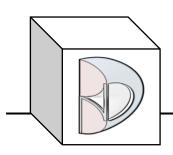






Run the Simulation.







#### Conclusion:

This is an example of how to create a CATIA DMU Kinematic simulation for a front independent suspension.

We hope this will help those who need this type of simulation.

As always, we are open to any discussions this may bring.

Please *subscribe* to our YouTube channel!