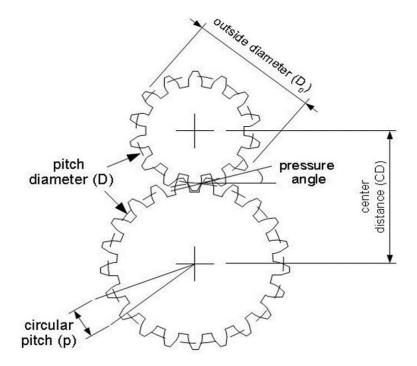


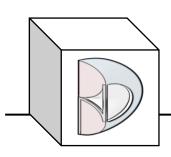
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TRANSVERSE MODULE INVOLUTE HELICAL GEAR

- This is step by step guide of how to create an involute helical gear using CATIA V5.
- This document assumes that you know basic gear geometry.

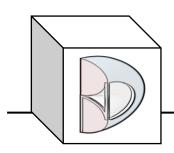
GEAR NOMENCLATURE Addendum circle Addendum Circular pitch Elsux Pitch Width Tooth thickness of space Dedendum Clearance -Fillet radius Dedendum Clearance circle circle





TRANSVERSE MODULE INVOLUTE HELICAL GEAR

- Helical gears are one type of cylindrical gears with slanted tooth trace.
 Compared to spur gears, they have the larger contact ratio and excel in quietness and less vibration and able to transmit large force. A pair of helical gears has the same helix angle but the helix hand is opposite.
- When the reference section of the gear is in the normal plane, by tilting the hobbing tool, the spur gear hobbing machine and hobbing tool can be used to produce helical gears. Because of the twist of teeth, their manufacturing has the disadvantage of more difficult production.
- While spur gears do not generate axial thrust forces, because of the twist in the tooth trace, helical gears produce axial thrust force. Therefore, it is desirable to use thrust bearings to absorb this force. However, combining right hand and left hand helical gears making double helical gears will eliminate the thrust force.
- Helical gears are often used in automotive transmissions by replacing spur gears.



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TRANSVERSE MODULE INVOLUTE HELICAL GEAR

- Helical gears can be classified into two groups by the reference section of the gears being in the rotating plane (transverse module) and normal plane (normal module).
- If the reference section is in the rotating plane (transverse module), the center distance is identical to spur gears as long as they are the same module and number of teeth. This allows for easy swapping with spur gears. However, in this case, they require special hobbing cutters and grinding stones, leading to higher production cost.
- On the other hand, if the reference section is in the normal plane (normal module), it is possible to use spur gear hobbing tools and grinding stones. However, the same module and number of teeth in spur gears no longer match the center distance of helical gears, and swapping becomes very difficult. In addition, the center distance is usually not an integer.

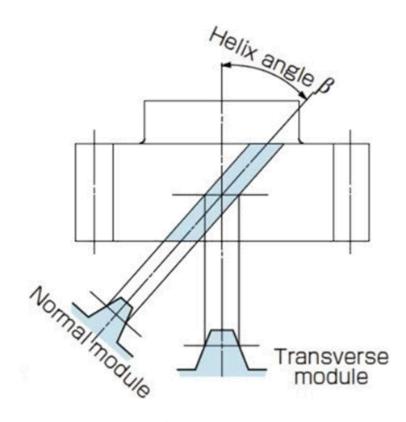
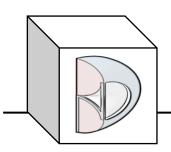


Fig. 2.9 Right-handed Helical Gear

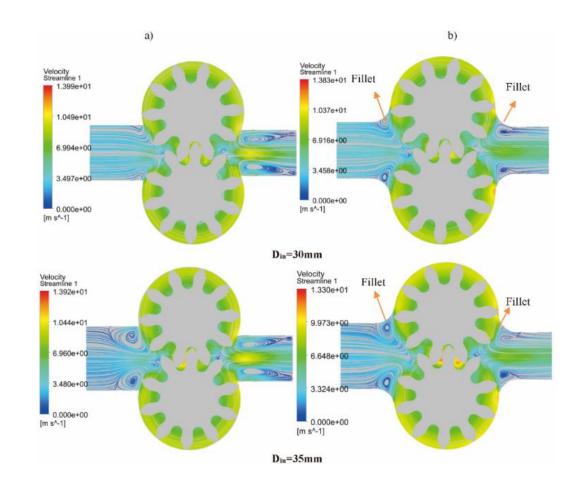


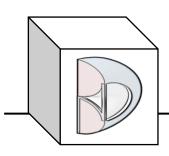
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MODELLING AN INVOLUTE

Q: Why would you spend the extra time to model an involute gear tooth profile?

A: CFD (Computational Fluid Dynamics). Take for example an external, positive displacement oil pump with helical gears. For CFD to accurately simulate the flow of the pump, the gears must be modeled as they will be manufactured.

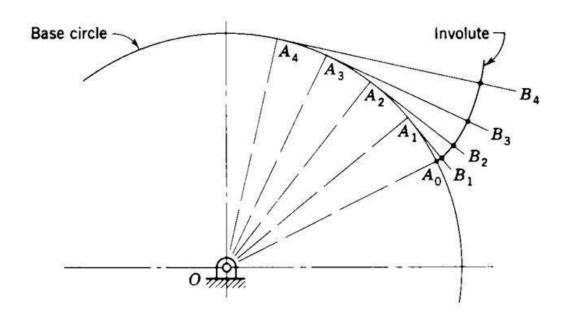


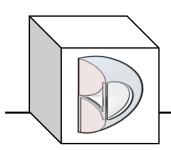


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MODELLING AN INVOLUTE

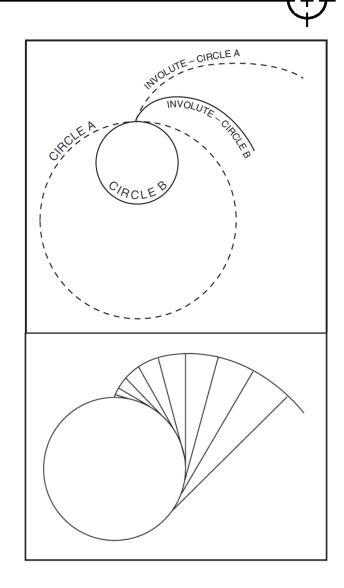
- Most CAD systems don't support the exact creation of involute curves.
- They must be represented using splines through a series of points.
- Of course the distance between the points will have a direct correlation to the accuracy of the spline.

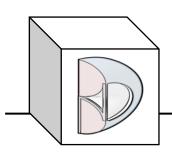




HELICAL GEARS INVOLUTE FORM

- Gear teeth could be manufactured with a wide variety of shapes and profiles. The involute profile is the most commonly used system for gearing today.
- An involute is a curve that is traced by a point on a taut cord unwinding from a circle, which is called a BASE CIRCLE. The involute is a form of spiral, the curvature of which becomes straighter as it is drawn from a base circle and eventually would become a straight line if drawn far enough.
- An involute drawn from a larger base circle will be less curved (straighter) than one drawn from a smaller base circle. Similarly, the involute tooth profile of smaller gears is considerably curved, on larger gears is less curved (straighter), and is straight on a rack, which is essentially an infinitely large gear.

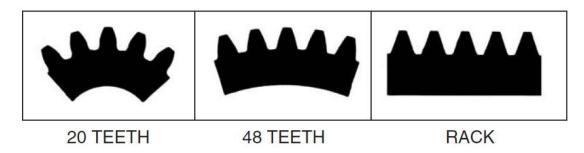




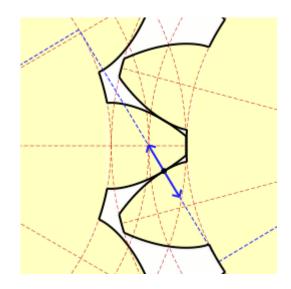
HELICAL GEARS INVOLUTE FORM

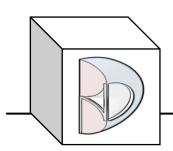


 Involute gear tooth forms and standard tooth proportions are specified in terms of a basic rack which has straight-sided teeth, for involute systems.



 Two involute gears, the left driving the right: Blue arrows show the contact forces between them; (1) downward force applied by the left gear and (2) upward resistance by the right gear. The force line (or line of action) runs along the long leg of dashed blue line which is a tangent common to both base circles.

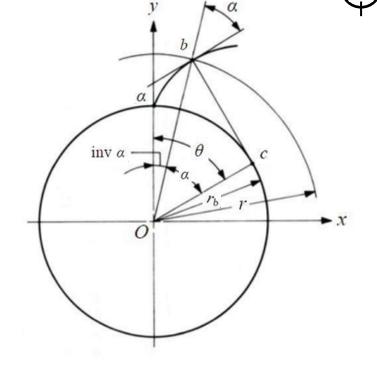




HELICAL GEARS INVOLUTE FORM

The definition of involute curve is the curve traced by a point on a straight line which rolls without slipping on the circle. The circle is called the base circle of the involutes.

inv α stands for Involute Angle (Involute α). The units for inv α is radians. θ is called involute rolling angle.



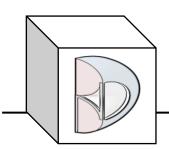
inv
$$\alpha = \tan \alpha - \alpha$$
 (radians)

With the center of the base circle O at the origin of a coordinate system, the involute curve can be expressed by values of x and y as follows:

$$\alpha = \cos^{-1} \frac{r_b}{r}$$

$$x = r \sin(inv \alpha)$$

$$y = r \cos(inv \alpha)$$



HELICAL GEAR TOOTH INVOLUTE FORM



Fw –Face width // length parameter [Fw =100mm]

m -module // length parameter [m = 8mm]

z -number of teeth // real parameter [z = 11]



Ca -Cylinder helix angle // angle parameter [Ca = 8deg]

s -symmetry angle // angle parameter [s = 90deg/z]

Pd -Pitch diameter // length parameter [Pd = z * m]

Bd -Base diameter // length parameter [Bd = Pd * cos(Tpa)]

Ad -Addendum diameter // length parameter [Ad = Pd+(2*m)].....(no profile shift)

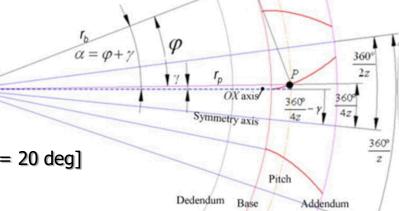
Dd -Dedendum diameter // length parameter [Dd = Pd-(2.5*m)].....(no profile shift)

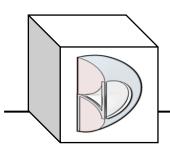
tr -tooth radius at dedendum circle // length parameter [tr = 0.38*m]

The parametric equations for involute curve points in CATIA:

x=r*sin((tan(acos(`r (Pt1)`/r))-(acos(`r (Pt1)`/r))))

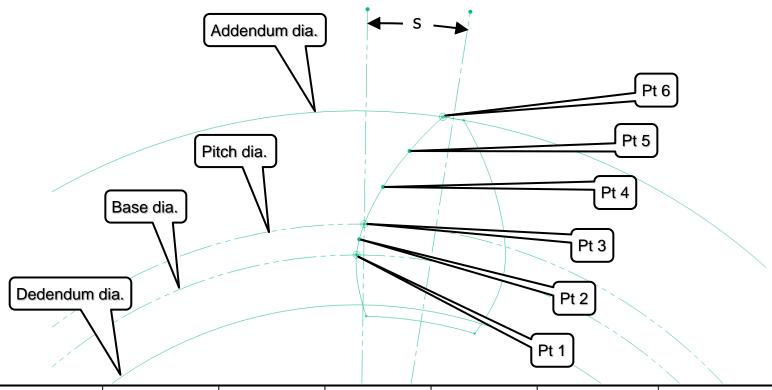
y=r*cos((tan(acos(`r (Pt1)`/r))-(acos(`r (Pt1)`/r))))



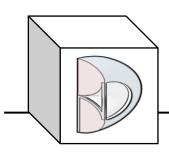




HELICAL GEAR TOOTH INVOLUTE POINTS

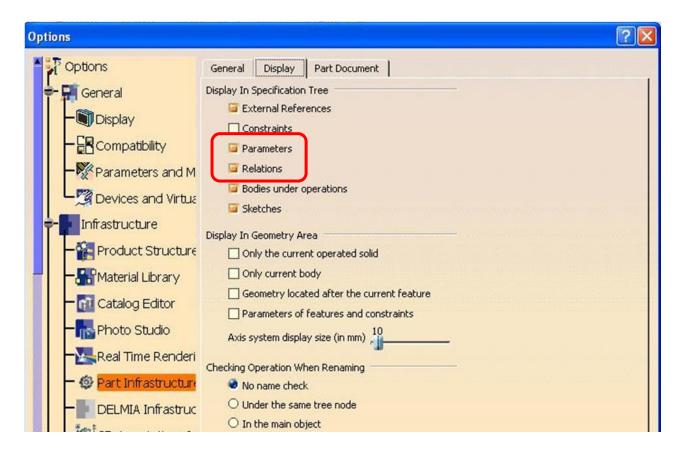


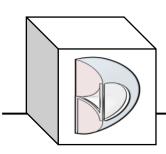
Point	r (formula)	r (mm)	α (radians)	inv α (radians)	x (mm)	y (mm)
Pt 1	[Base] Br	41.346	0.0000	0.0000	0.000	41.346
Pt 2	(Pr+Br)/2	42.673	0.2500	0.0053	0.228	42.673
Pt 3	[Pitch] Pr	44.000	0.3491	0.0149	0.656	43.995
Pt 4	Pr+(Ar-Pr)*.33333	46.667	0.4822	0.0412	1.922	46.627
Pt 5	Pr+(Ar-Pr)*.66667	49.333	0.5770	0.0739	3.642	49.199
Pt 6	[Addendum] Ar	52.000	0.6516	0.1111	5.766	51.679





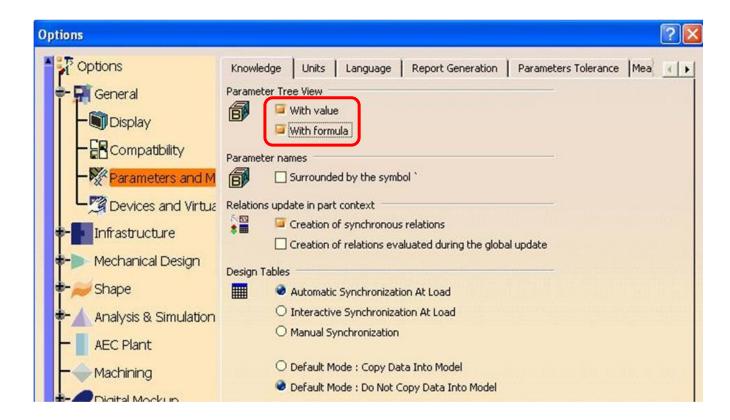
- When you start CATIA, go to TOOLS->OPTIONS->Infrastructure->
- Part Infrastructure and in Display select "Parameters" and "Relations".

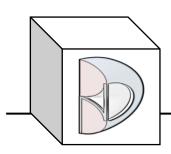






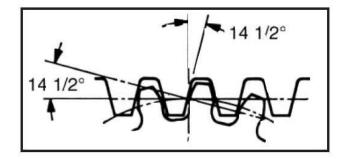
 Then in Options->General in Parameters and Measures select "With Value" and "With Formula" in Parameters Tree View.

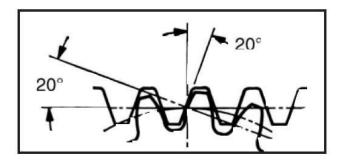


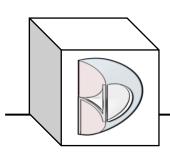


PRESSURE ANGLE

- Pressure angle is the angle at a pitch point between the line of pressure which is normal to the tooth surface, and the plane tangent to the pitch surface. The pressure angle refers to the angle when the gears are mounted on their standard center distances.
- Standard pressure angles (PA) are 14-1/2° and 20°. While 20° PA is generally recognized as having higher load carrying capacity, 14-1/2° PA gears have extensive use. The lower pressure angle results in less change in backlash due to center distance variation and concentricity errors. It also provides a higher contact ratio and consequent smoother, quieter operation provided that undercut of teeth is not present.

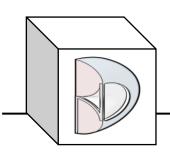






BACKLASH

- Backlash, a clearance between mating gear teeth, is built into speed reducers to let the gears mesh without binding and to provide space for a film of lubricating oil between the teeth. This prevents overheating and tooth damage.
- For smooth rotation of meshed gears, backlash is necessary. Backlash is the amount by which a tooth space exceeds the thickness of a gear tooth engaged in mesh. Backlashes are classified in the following ways:
 - Circumferential Backlash (j_t) Circumferential Backlash is the length of arc on the pitch circle. The length is the distance the gear is rotated until the meshed tooth flank makes contact while the other mating gear is held stationary.
 - Normal Backlash (j_n) The minimum distance between each meshed tooth flank in a pair of gears, when it is set so the tooth surfaces are in contact.
 - 3) Angular Backlash (j_{θ}) The maximum angle that allows the gear to move when the other mating gear is held stationary.
 - 4) Radial Backlash (j_r) The radial backlash is the shrinkage (displacement) in the stated center distance when it is set so the meshed tooth flanks of the paired gears contact each other.
 - 5) Axial Backlash (j_x) The axial backlash is the shrinkage (displacement) in the stated center distance when a pair of bevel gears is set so the meshed tooth flanks of the paired gears contact each other.

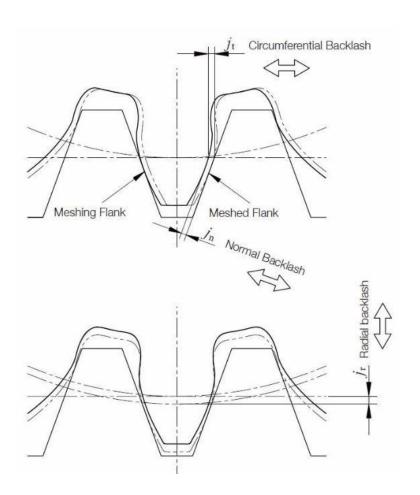


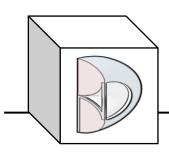
BACKLASH



- Circumferential Backlash
- Normal Backlash

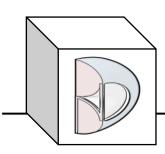
Radial Backlash





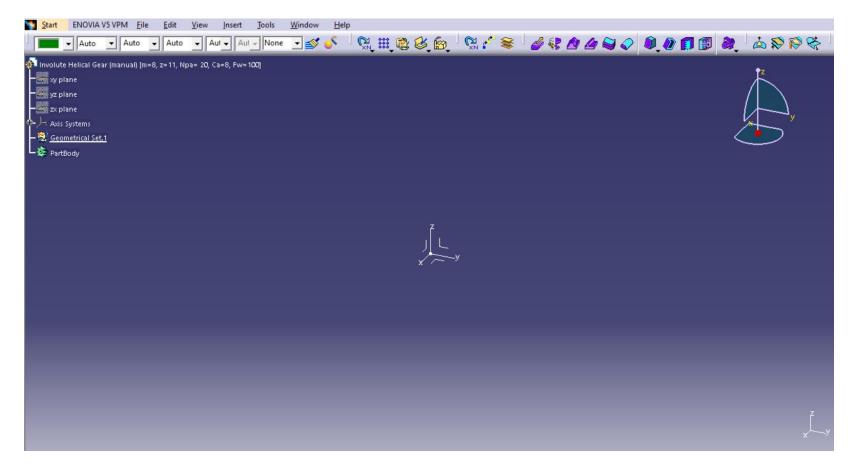


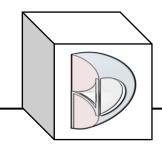
Create the Parameters





 Open a new CATPart and give it the name: Transverse Involute Helical Gear [Fw=100, m=8, z=11, Tpa=20, Ca=8]





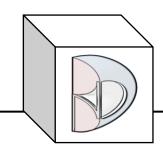


- Create the following Eleven formulae:
 - Known:
 - Fw // length parameter [Face width: Fw =100mm]
 - m // length parameter [Module: m = 8mm]
 - z // real parameter [Number of teeth: z = 11]
 - Tpa // angle parameter [Transverse pressure angle: Tpa = 20 deg]
 - Ca // angle parameter [Cylinder helix angle: Ca = 8deg]

Resultant:

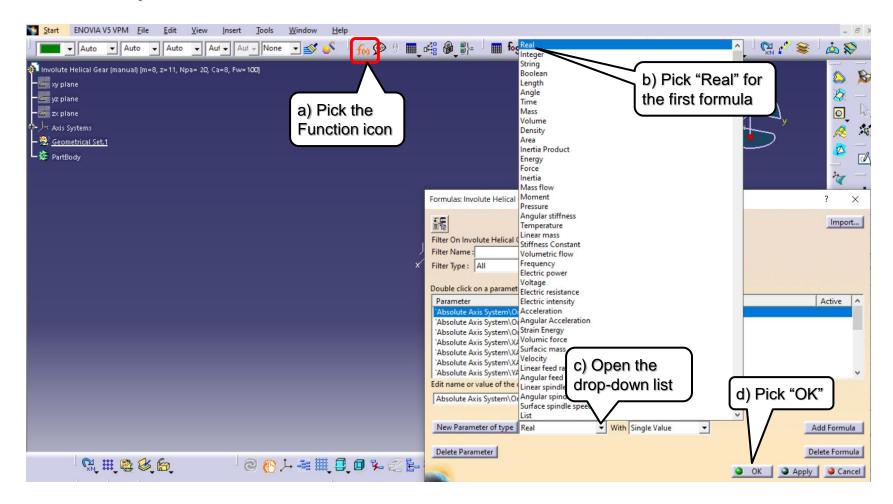
- s // angle parameter [Symmetry angle: s = 90deg/z]
- Pd // length parameter [Pitch diameter: Pd = z*m]
- Bd // length parameter [Base diameter: Bd = Pd * cos(Tpa*1rad)]
- Ad // length parameter [Addendum diameter: Ad = Pd+(2*m)]
- Dd // length parameter [Dedendum diameter: Dd = Pd-(2.5*m)]
- tr // length parameter [tooth radius at dedendum circle: tr = 0.38*m]

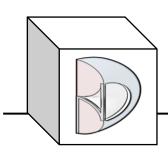
CATIA sees a diametral dimension as a radius. Even though the dimension displays as a diameter.





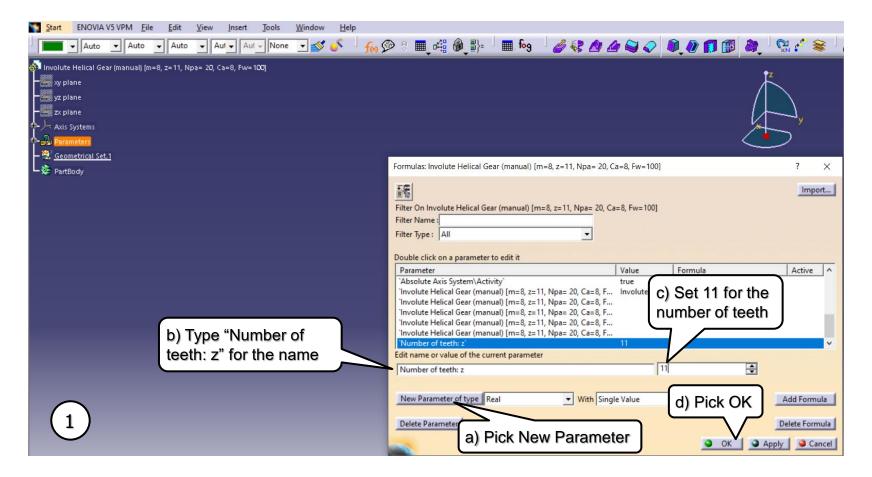
Open the Formulas window

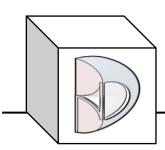






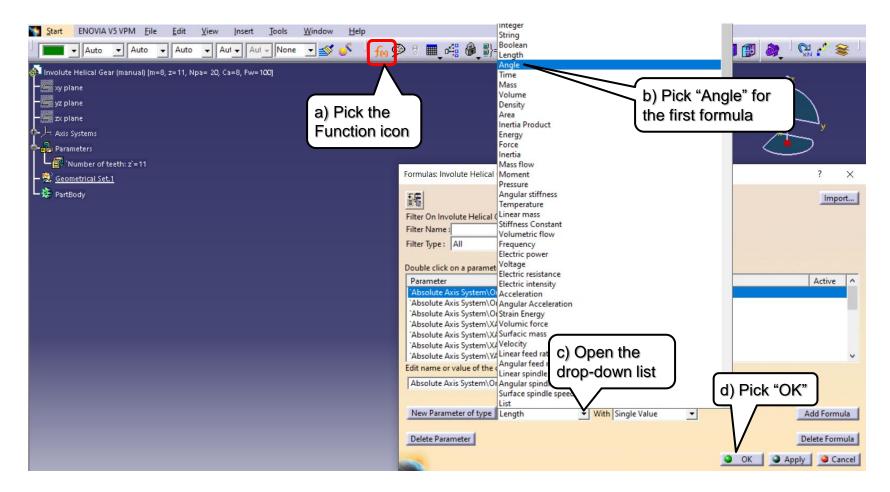
1) z // real parameter [Number of teeth: z = 11]

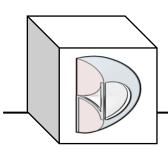






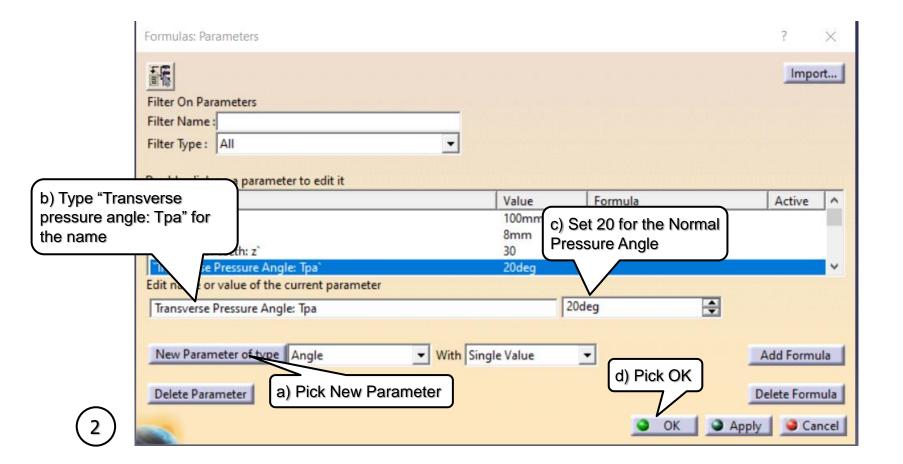
Open the Formulas window

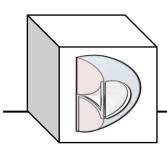






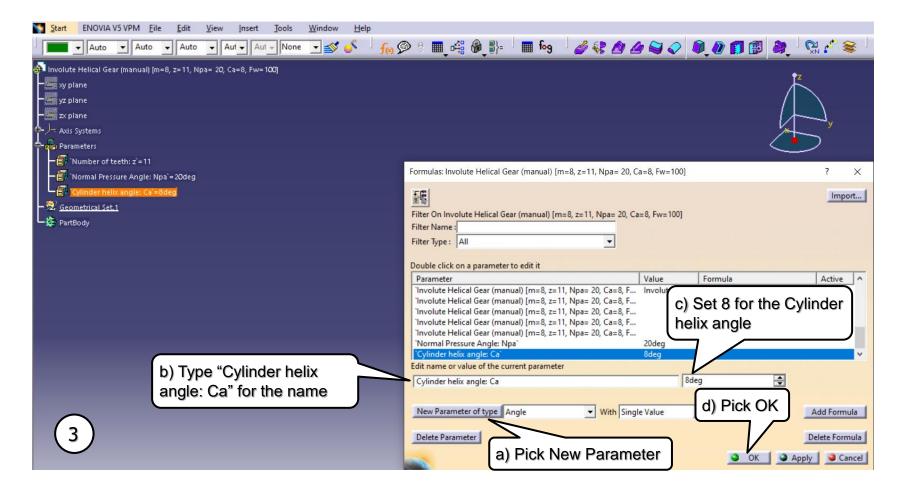
2) Tpa // angle parameter [Transverse pressure angle: Tpa = 20 deg]

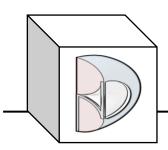






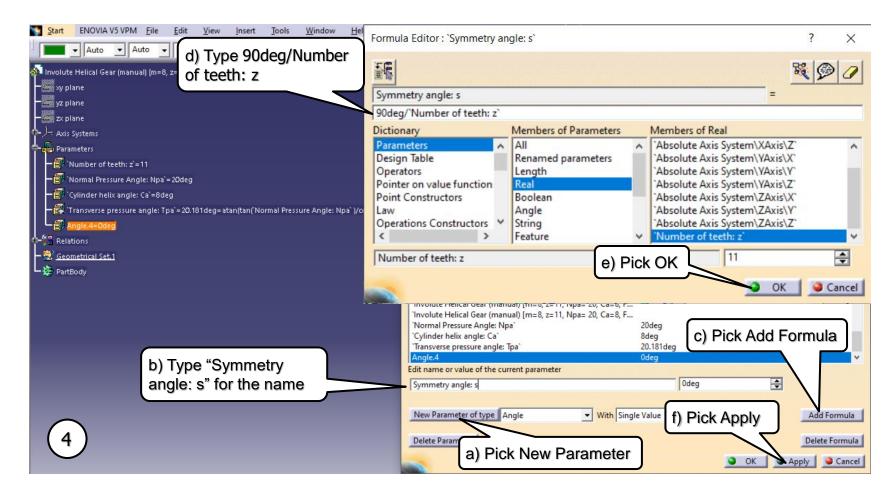
3) Ca // angle parameter [Cylinder helix angle: Ca = 8deg]

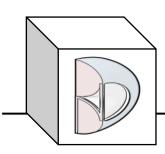






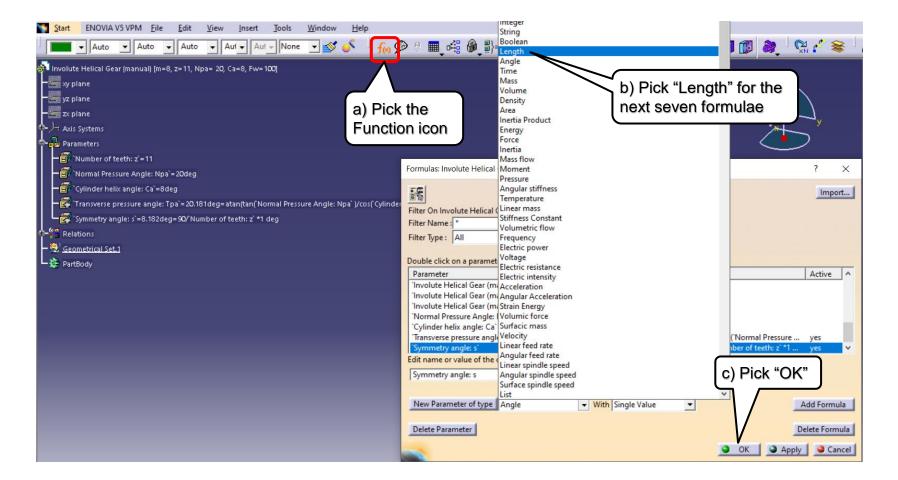
4) s // angle parameter [Symmetry angle: s = 90deg/z]

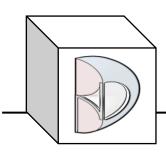






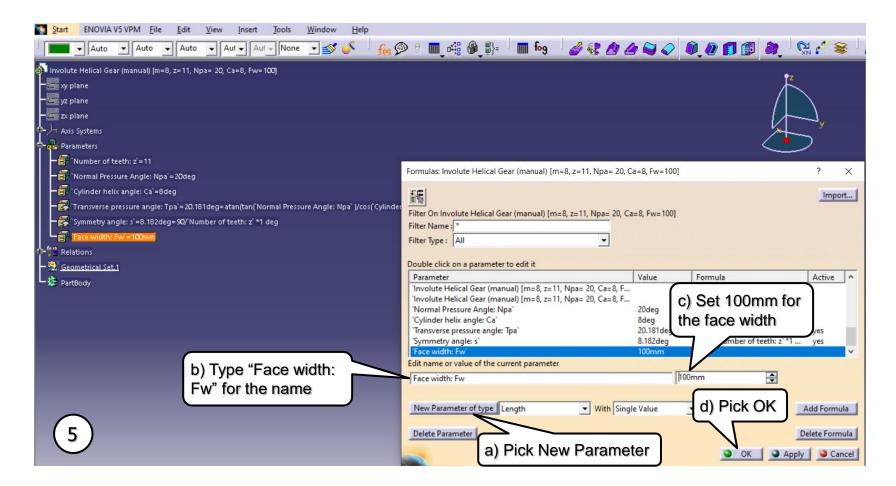
The next seven Parameters will be Length type.

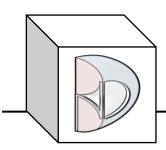






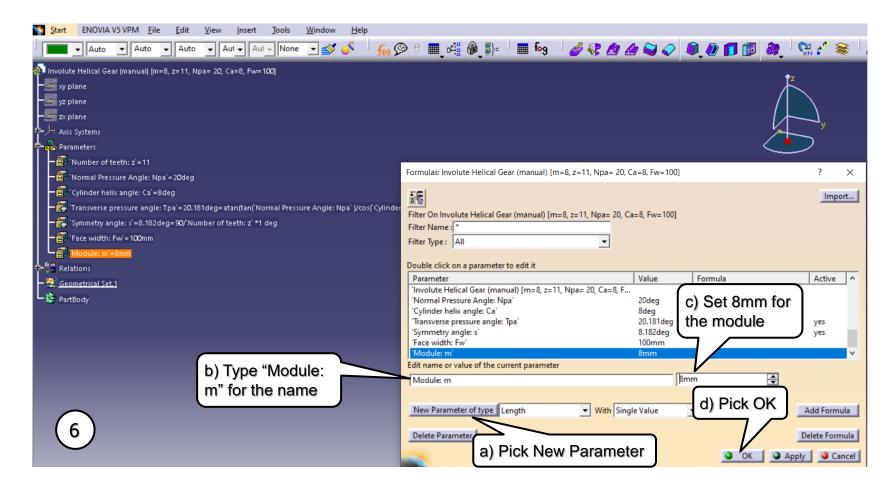
5) Fw // length parameter [Face width: Fw = 100mm]

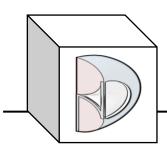






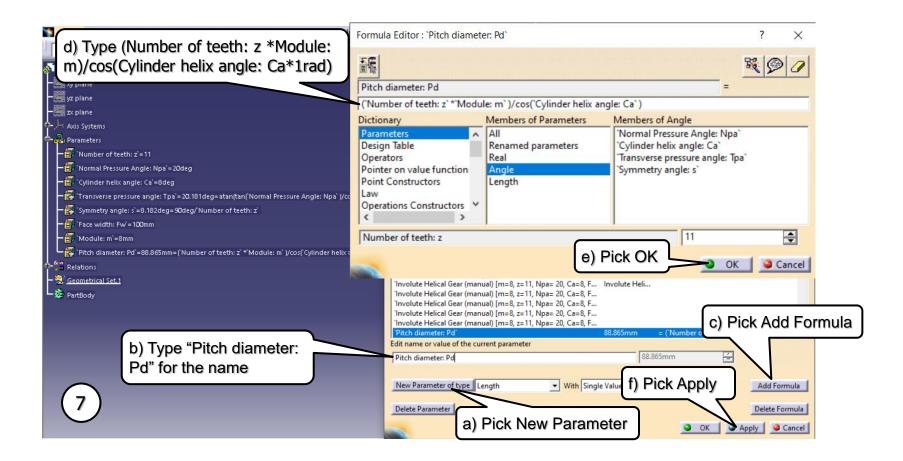
6) m // length parameter [Module: m = 8mm]

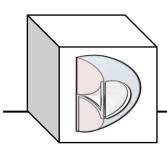






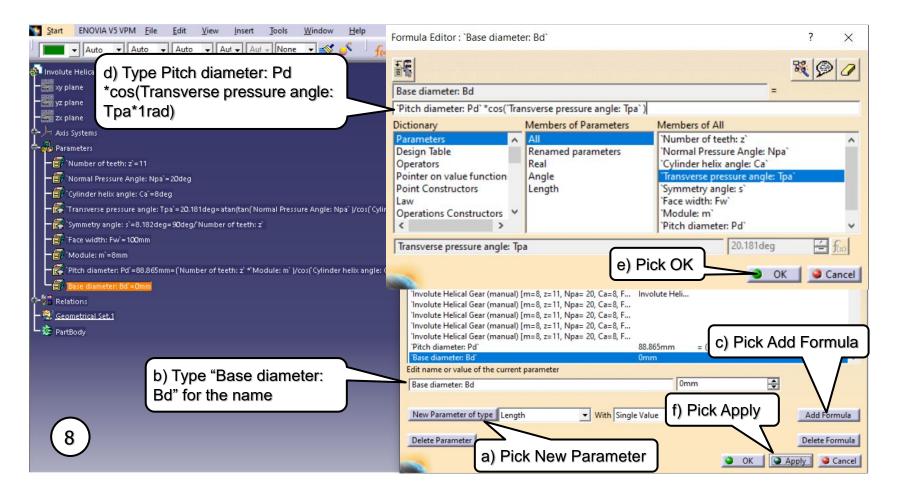
7) Pd // length parameter [Pitch diameter: Pd = (z*m)/cos(Ca*1rad)]

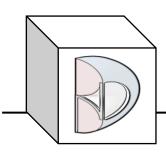






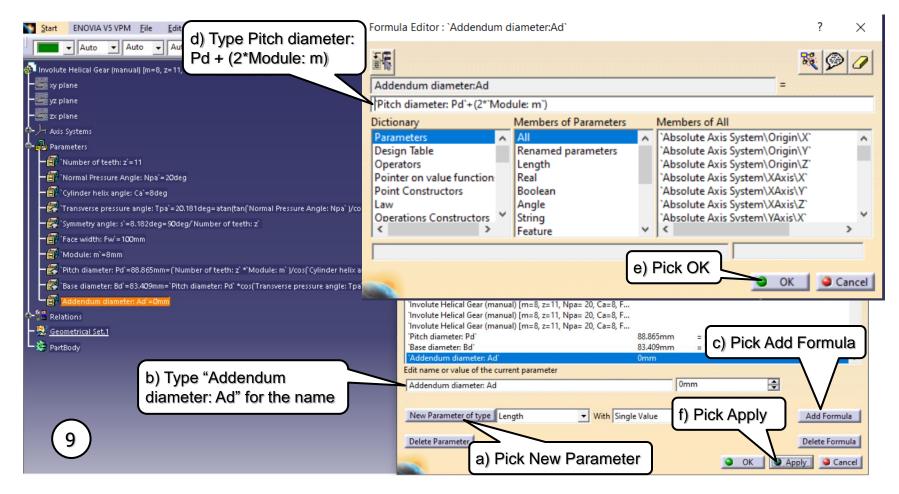
8) Bd // length parameter [Base diameter: Bd = Pd*cos(Tpa*1rad)]

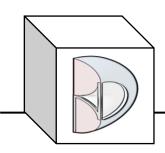






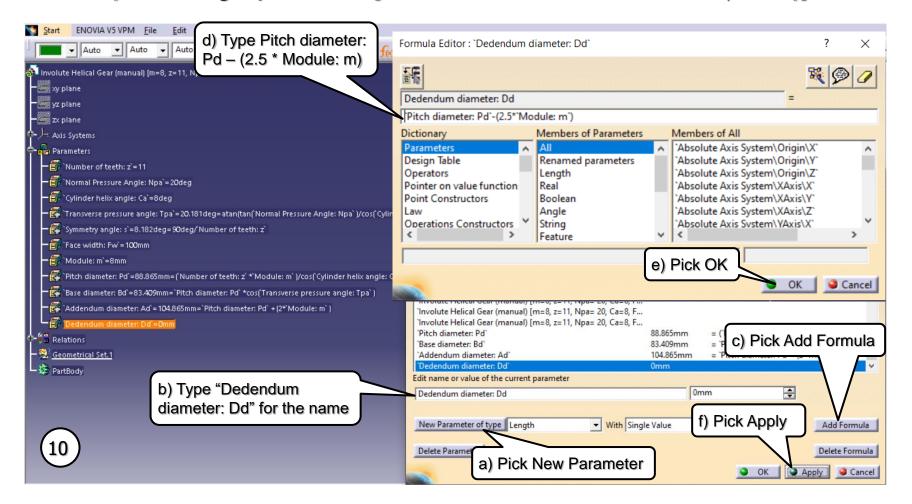
9) Ad // length parameter [Addendum diameter: Ad = Pd+(2*m)]

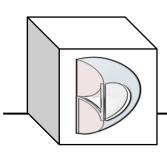






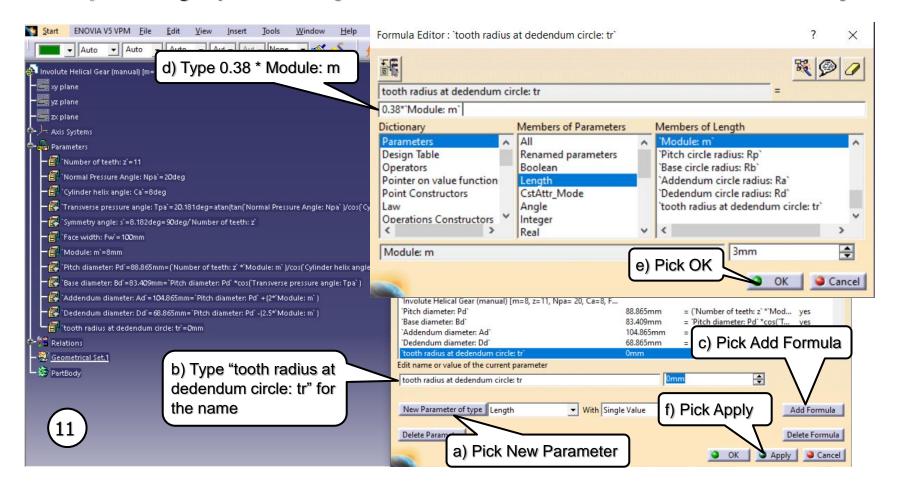
10) Dd // length parameter [**Dedendum diameter: Dd** = Pd-(2.5*m)]

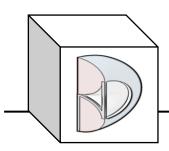






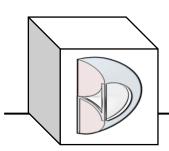
11) tr // length parameter [tooth radius at dedendum circle: tr = 0.38*m]





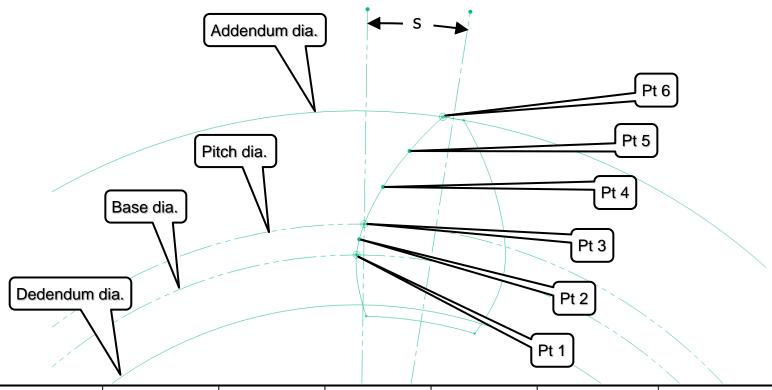


- Create the following six formulae for Involute points:
 - radius (Pt1) // length parameter [r (Pt1) = Bd/2]
 - radius (Pt2) // length parameter [r (Pt2) = ((Pd/2)+(Bd/2))/2]
 - radius (Pt3) // length parameter [r (Pt3) = Pd/2]
 - radius (Pt4) // length parameter [r (Pt4) = (Pd/2)+((Ad/2)-(Pd/2))*.33333]
 - radius (Pt5) // length parameter [r (Pt5) = (Pd/2)+((Ad/2)-(Pd/2))*.66667]
 - radius (Pt6) // length parameter [r (Pt6) = Ad/2]

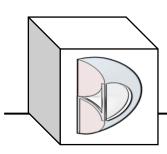




HELICAL GEAR TOOTH INVOLUTE POINTS

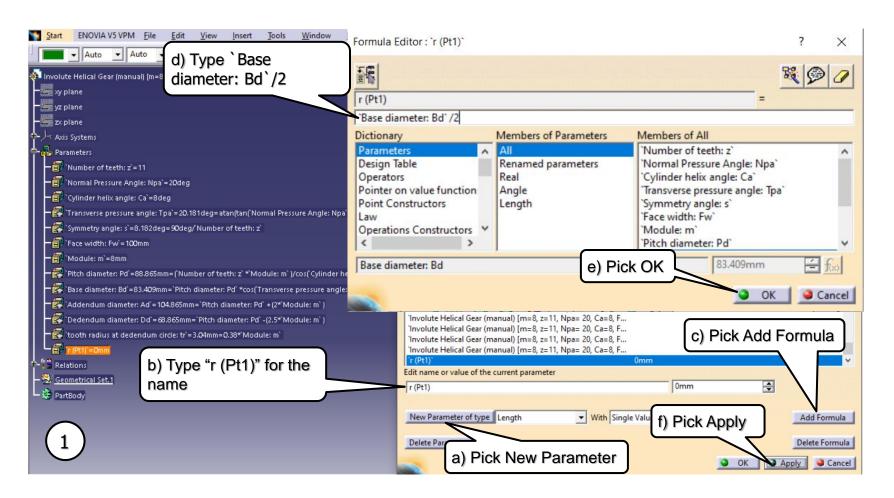


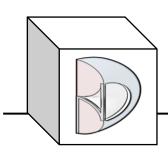
Point	r (formula)	r (mm)	α (radians)	inv α (radians)	x (mm)	y (mm)
Pt 1	[Base] Br	41.346	0.0000	0.0000	0.000	41.346
Pt 2	(Pr+Br)/2	42.673	0.2500	0.0053	0.228	42.673
Pt 3	[Pitch] Pr	44.000	0.3491	0.0149	0.656	43.995
Pt 4	Pr+(Ar-Pr)*.33333	46.667	0.4822	0.0412	1.922	46.627
Pt 5	Pr+(Ar-Pr)*.66667	49.333	0.5770	0.0739	3.642	49.199
Pt 6	[Addendum] Ar	52.000	0.6516	0.1111	5.766	51.679





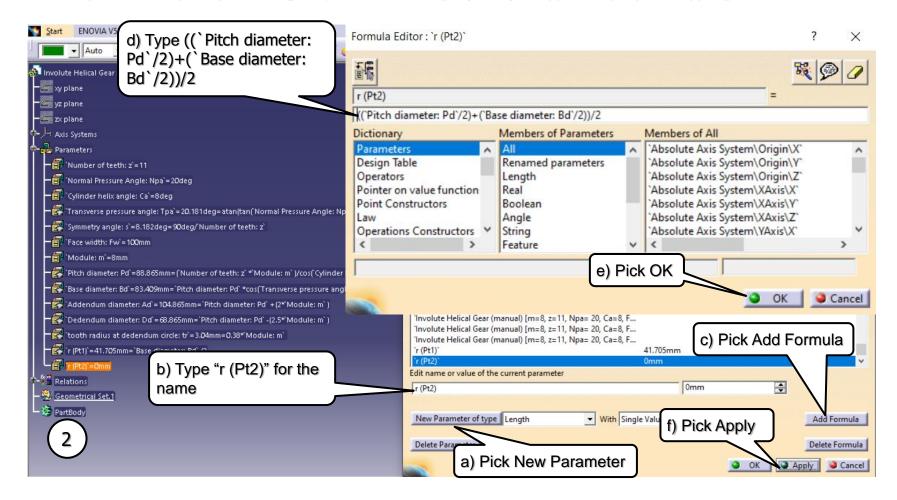
1) radius (Pt1) // length parameter [r (Pt1) = Bd/2]

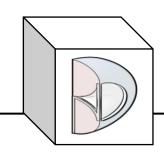






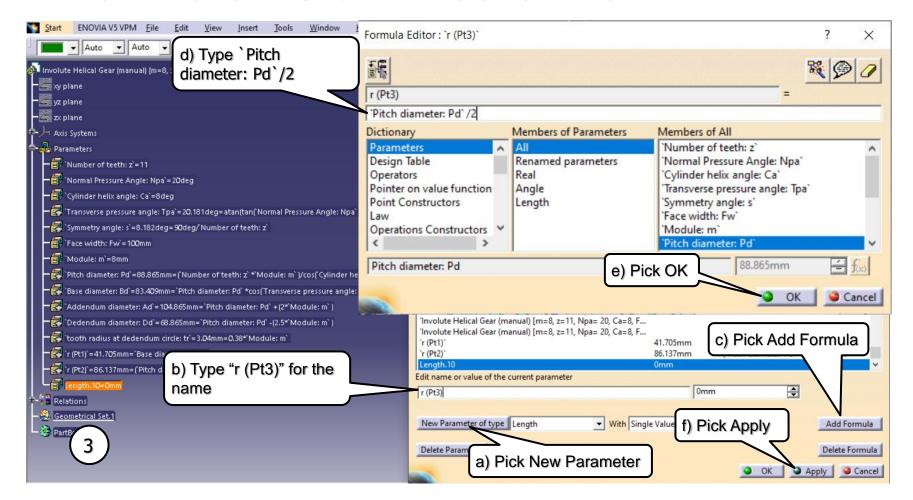
2) radius (Pt2) // length parameter [\mathbf{r} (Pt2) = ((Pd/2)+(Bd/2))/2]

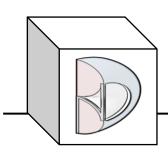






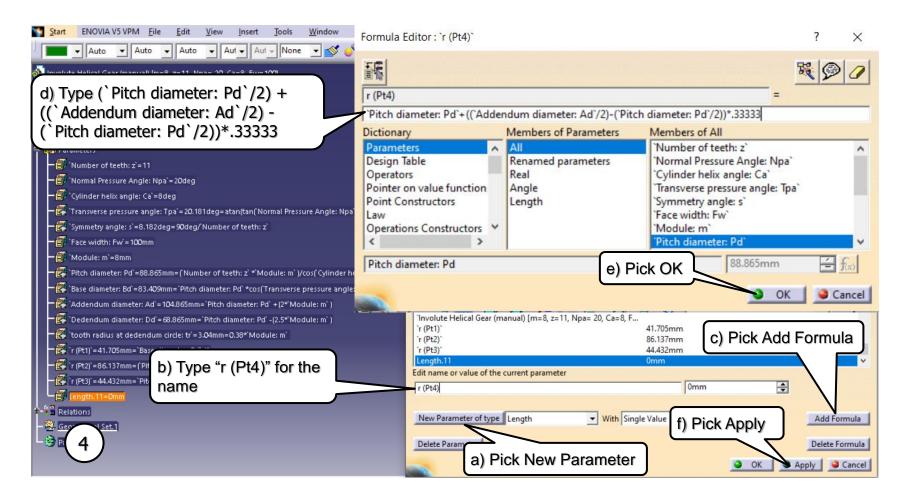
3) radius (Pt2) // length parameter [r (Pt3) = Pd/2]

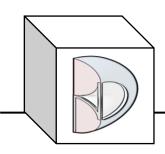






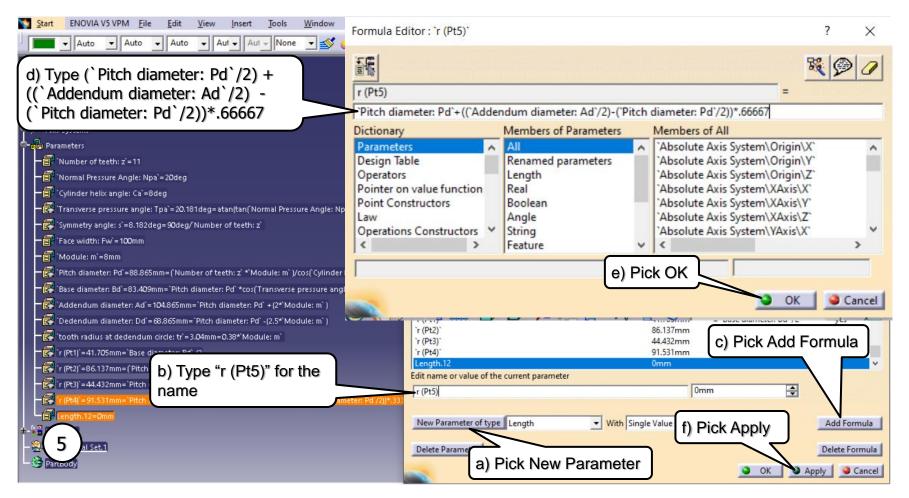
4) radius (Pt4) // length parameter [r (Pt4) = (Pd/2)+((Ad/2)-(Pd/2))*.33333]

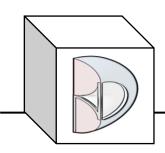






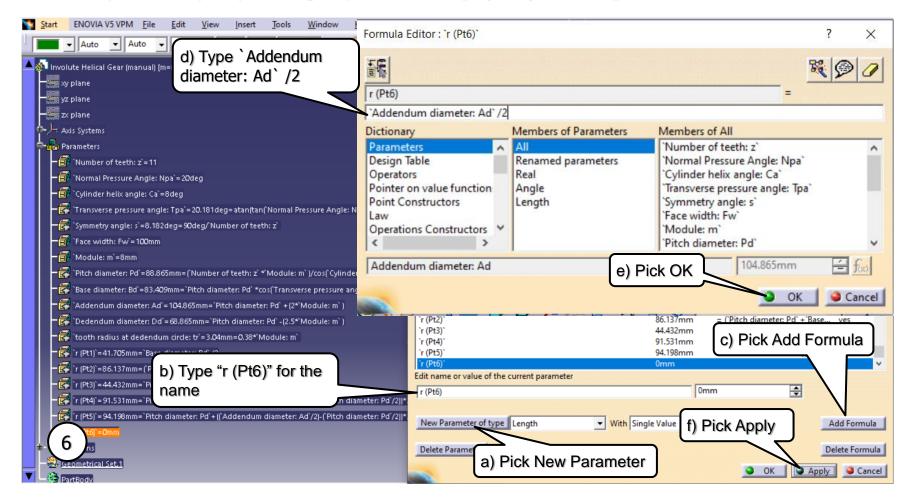
5) radius (Pt5) // length parameter [\mathbf{r} (Pt5) = (Pd/2)+((Ad/2)-(Pd/2))*.66667]

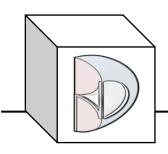






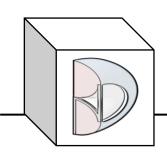
6) radius (Pt6) // length parameter [r (Pt6) = Ad/2]







Create the Involute Laws





How the parametric equations for the Involute points are developed:

From Excel Spreadsheet

inv
$$\alpha = \tan \alpha - \alpha$$
 (rad) (3.2)

With the center of the base circle O at the origin of a coordinate system, the involute curve can be expressed by values of x and y as follows:

$$\alpha = \cos^{-1} \frac{r_b}{r}$$

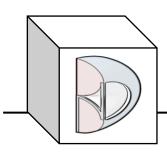
$$x = r \sin(inv \alpha)$$

$$y = r \cos(inv \alpha)$$
(3.3)

Point	r (formula)	r (mm)	α (radians)	inv α (radians)	x (mm)	y (mm)
Pt 1	[Base] Br	41.705	0.0000	0.0000	0.000	41.705
Pt 2	(Pr+Br)/2	43.069	0.2523	0.0055	0.237	43.068
Pt 3	[Pitch] Pr	44.432	0.3522	0.0153	0.681	44.427
Pt 4	Pr+(Ar-Pr)*.33333	47.099	0.4833	0.0415	1.955	47.058
Pt 5	Pr+(Ar-Pr)*.66667	49.766	0.5772	0.0740	3.677	49.630
Pt 6	[Addendum] Ar	52.432	0.6511	0.1109	5.801	52.111

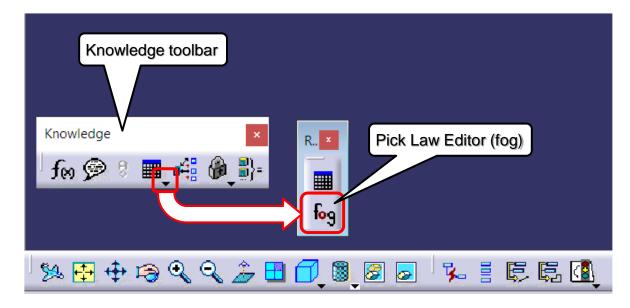
The parametric equations for involute curve points in CATIA:

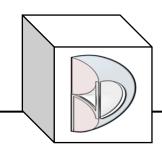
- x=r*sin((tan(acos(`r (Pt1)`/r))-(acos(`r (Pt1)`/r))))
- y=r*cos((tan(acos(`r (Pt1)`/r))-(acos(`r (Pt1)`/r))))



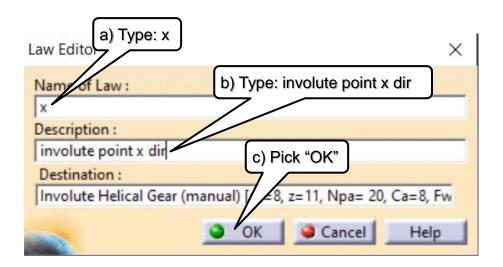


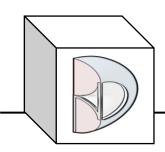
- Create the following two Laws for the creation of the Involute points:
 - x=r*sin((tan(acos(`r (Pt1)`/r))-(acos(`r (Pt1)`/r))))
 - y=r*cos((tan(acos(`r (Pt1)`/r))-(acos(`r (Pt1)`/r))))
- In the Knowledge Toolbar, pull out the Law Editor (fog).



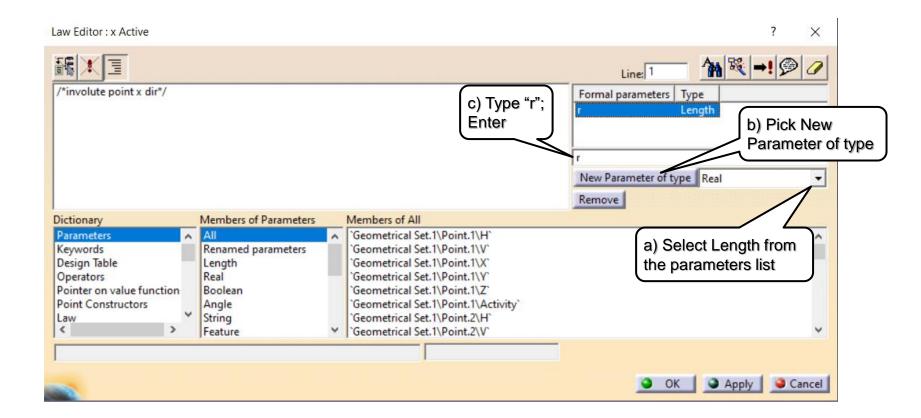


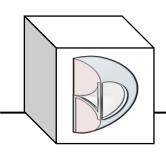




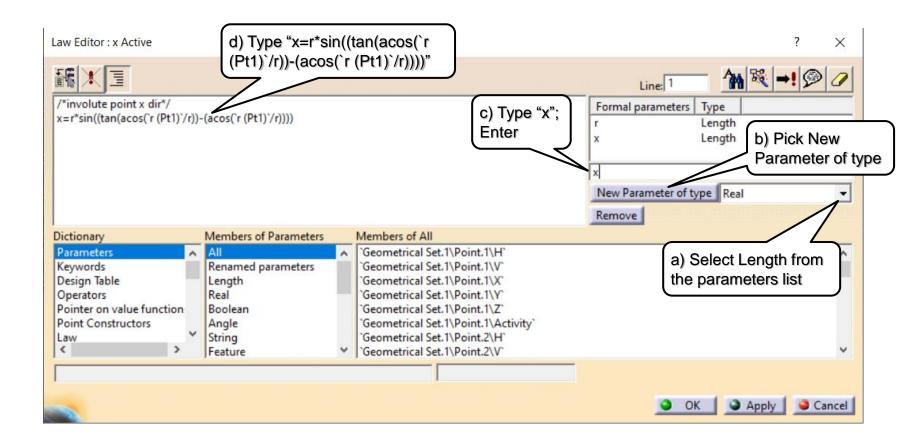


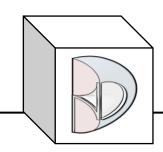




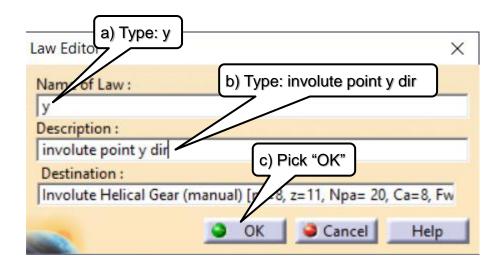


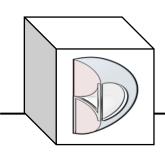




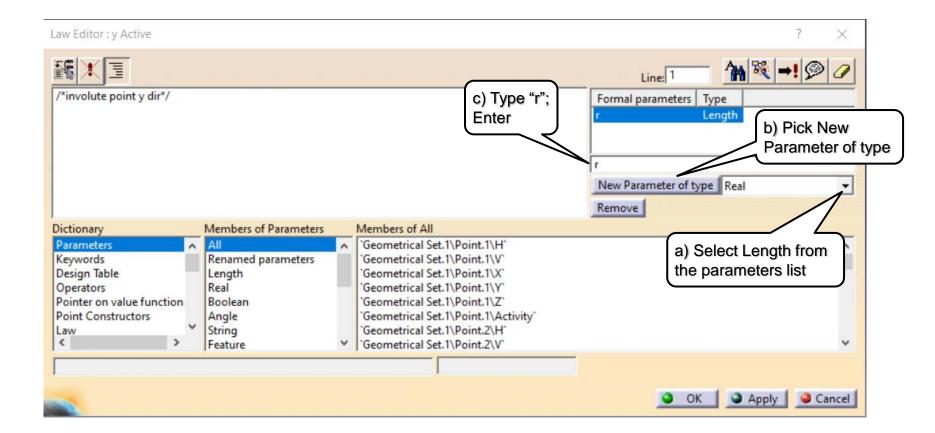


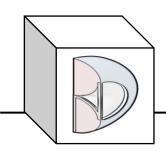




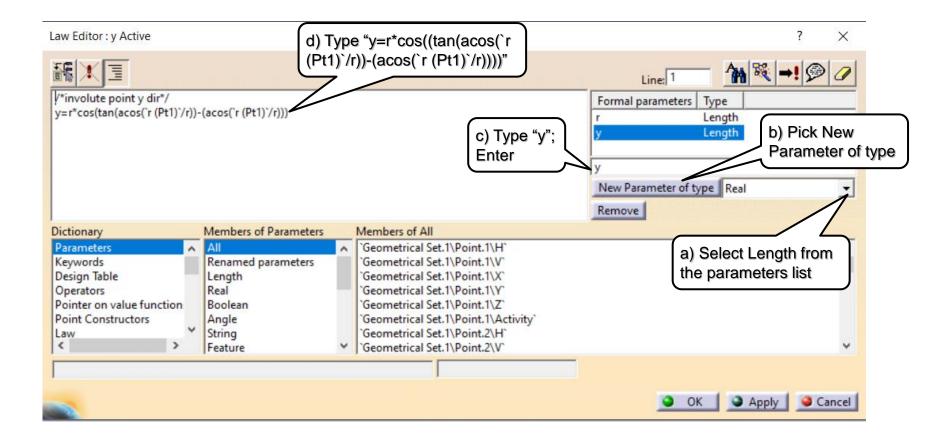


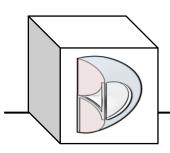








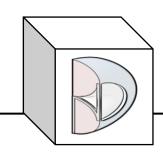






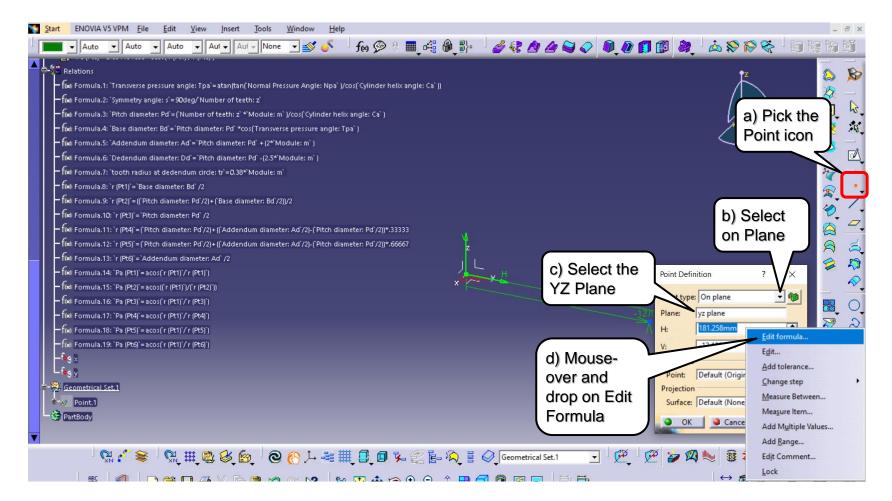
Create the Geometry

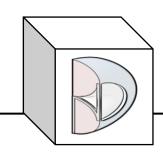
(Transverse Module Involute Helical Gear)





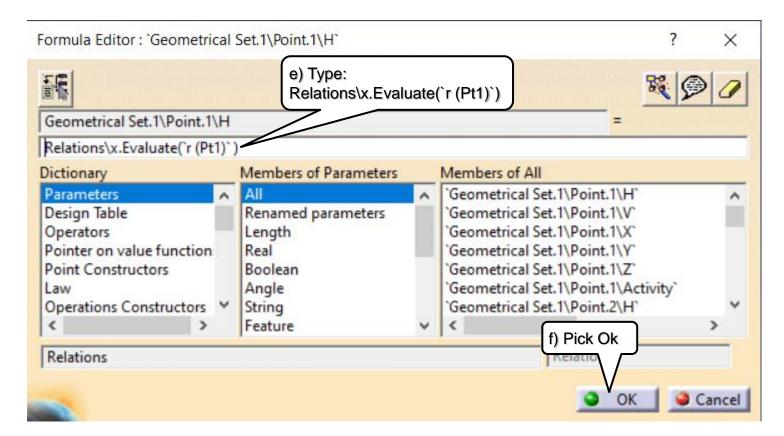
Create points on plane YZ for involute spline.

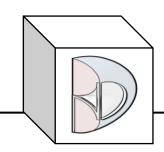






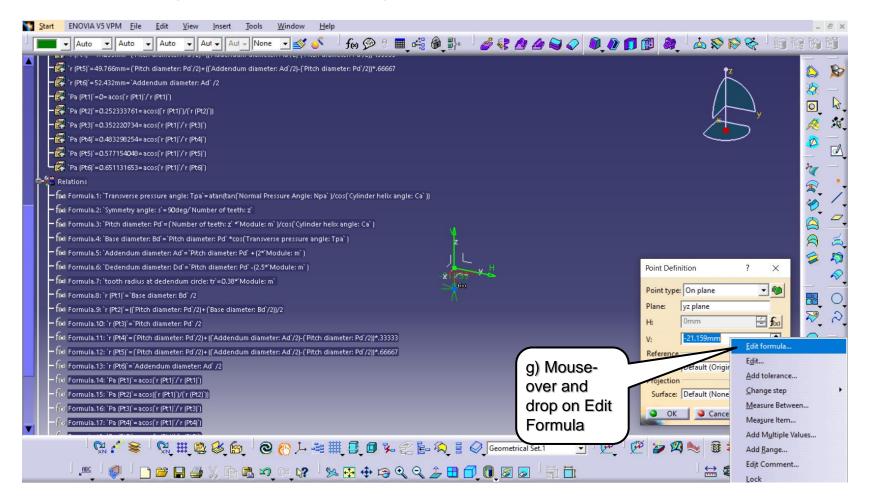
Create points for involute spline.

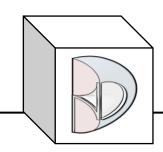






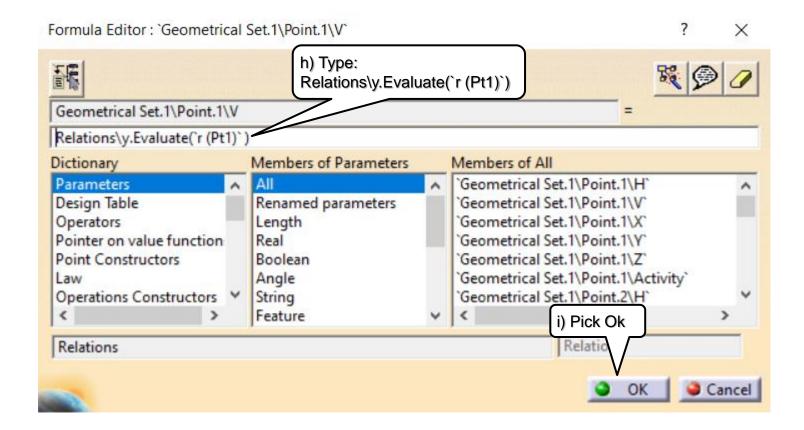
Create points for involute spline.

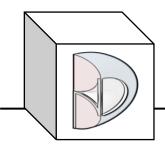






Create points for involute spline.







- Create points for involute spline.
- Repeat steps a) through i) for the next five points evaluating r (Pt_) for each sequential point.
- Afterwards there should be seven points as follows:

- Point 1:

- H direction = Relations\x.Evaluate ('r (Pt1)')
- V direction = Relations\y.Evaluate ('r (Pt1)')

- Point 2:

- H direction = Relations\x.Evaluate ('r (Pt1)')
- V direction = Relations\y.Evaluate ('r (Pt1)')

- Point 3:

- H direction = Relations\x.Evaluate ('r (Pt1)')
- V direction = Relations\y.Evaluate ('r (Pt1)')

- Point 4:

- H direction = Relations\x.Evaluate ('r (Pt1)')
- V direction = Relations\y.Evaluate ('r (Pt1)')

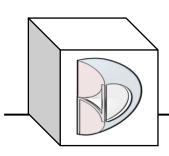
- Point 5:

- H direction = Relations\x.Evaluate ('r (Pt1)')
- V direction = Relations\y.Evaluate ('r (Pt1)')

- Point 6:

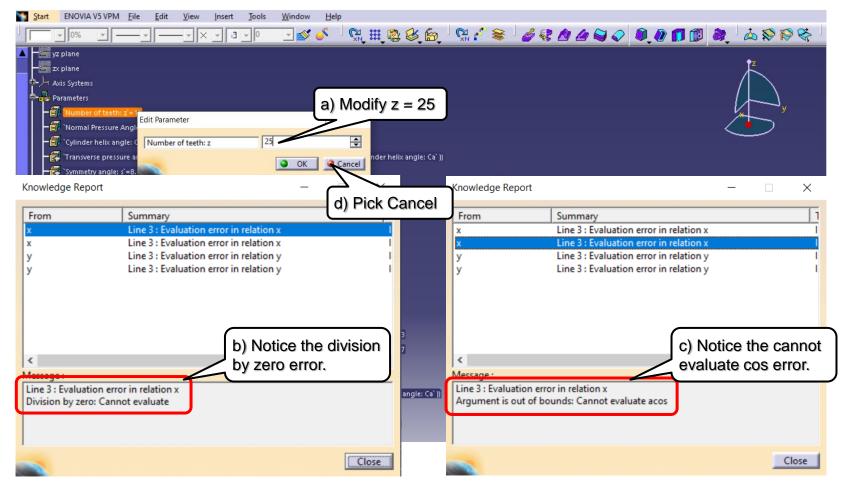
- H direction = Relations\x.Evaluate ('r (Pt1)')
- V direction = Relations\y.Evaluate ('r (Pt1)')

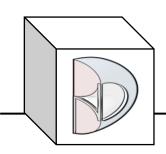
```
- fixi Formula.24: `Geometrical Set.1\Point.1\H` = Relations\x.Evaluate('r (Pt1)`)
- fixi Formula.25: `Geometrical Set.1\Point.1\V` = Relations\y.Evaluate('r (Pt1)`)
- fixi Formula.14: `Geometrical Set.1\Point.2\H` = Relations\y.Evaluate('r (Pt2)`)
- fixi Formula.15: `Geometrical Set.1\Point.2\V` = Relations\y.Evaluate('r (Pt2)`)
- fixi Formula.16: `Geometrical Set.1\Point.3\H` = Relations\y.Evaluate('r (Pt3)`)
- fixi Formula.17: `Geometrical Set.1\Point.3\V` = Relations\y.Evaluate('r (Pt3)`)
- fixi Formula.18: `Geometrical Set.1\Point.4\H` = Relations\y.Evaluate('r (Pt4)`)
- fixi Formula.20: `Geometrical Set.1\Point.5\H` = Relations\y.Evaluate('r (Pt4)`)
- fixi Formula.21: `Geometrical Set.1\Point.5\V` = Relations\y.Evaluate('r (Pt5)`)
- fixi Formula.22: `Geometrical Set.1\Point.6\H` = Relations\y.Evaluate('r (Pt5)`)
- fixi Formula.23: `Geometrical Set.1\Point.6\H` = Relations\y.Evaluate('r (Pt6)`)
```





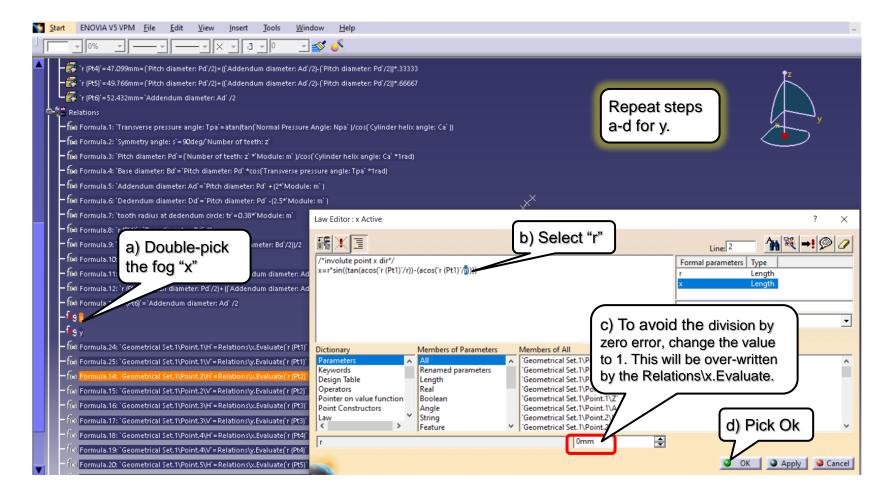
Modify parameters for Number of teeth: z to 25 and check the results.

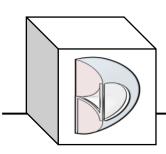






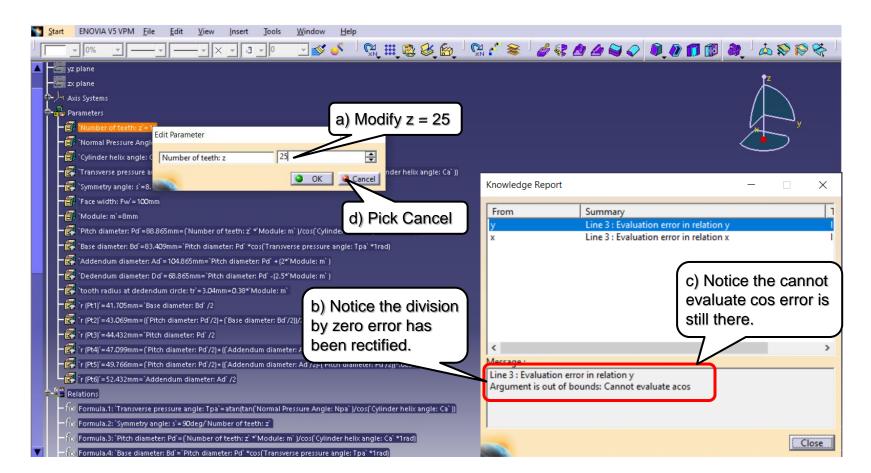
Correct the errors and check the results.

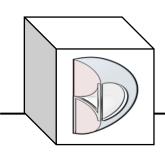






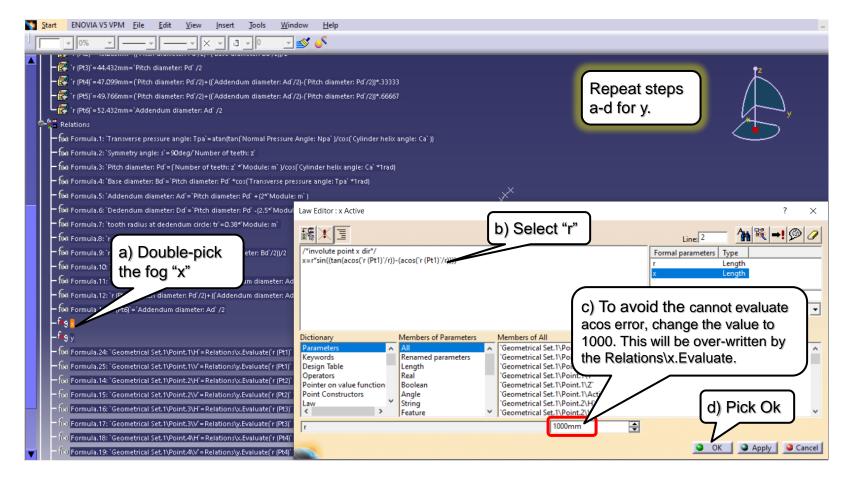
Modify parameters for Number of teeth: z to 25 and check the results.

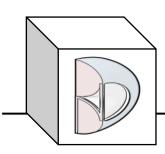






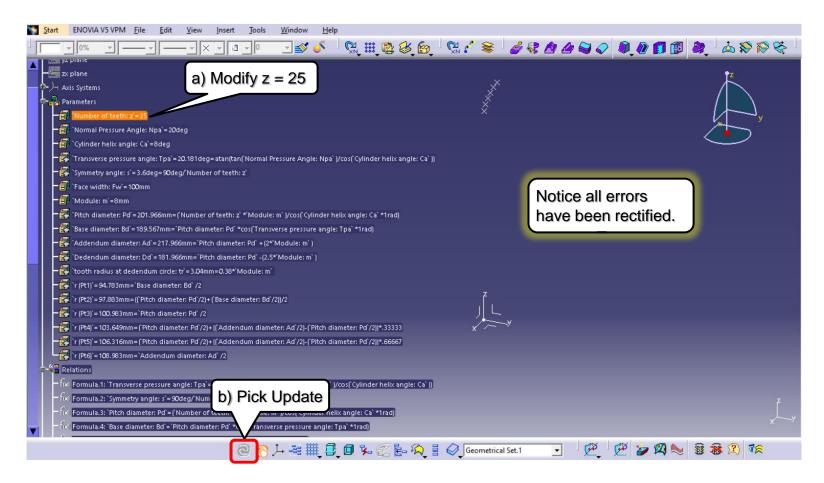
Correct the errors and check the results.

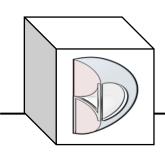






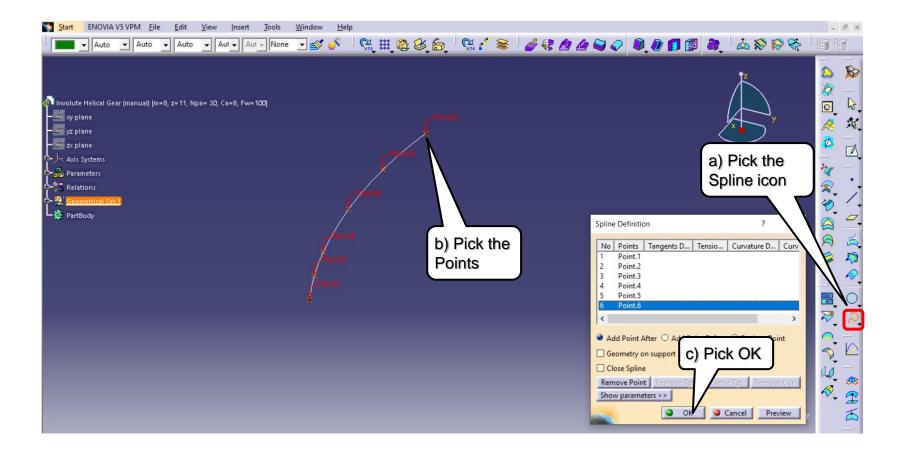
Modify parameters for Number of teeth: z to 25 and check the results.

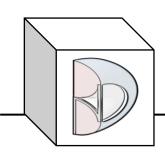






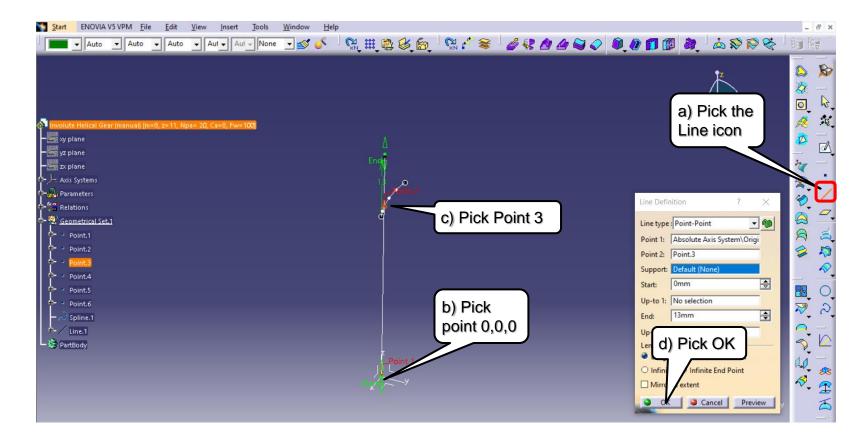
Create the involute spline through the points.

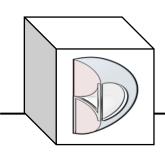






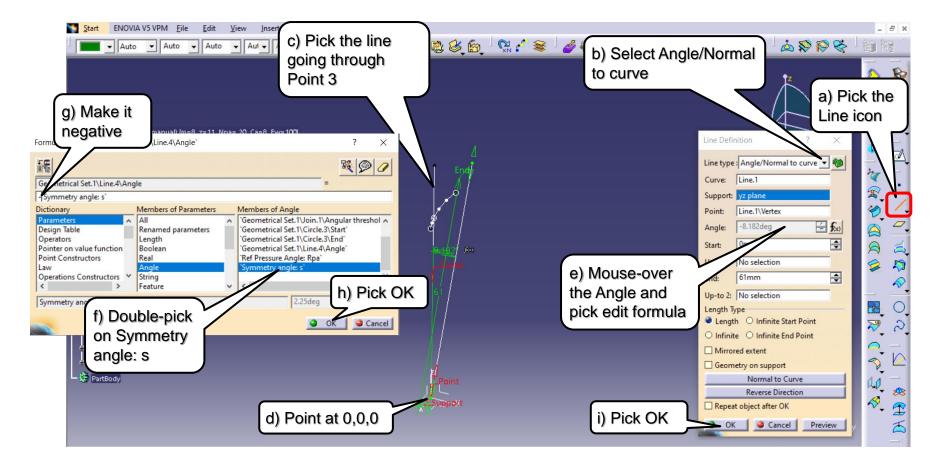
Create a line from 0,0,0 through Point 3.

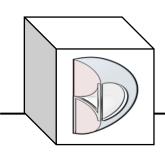






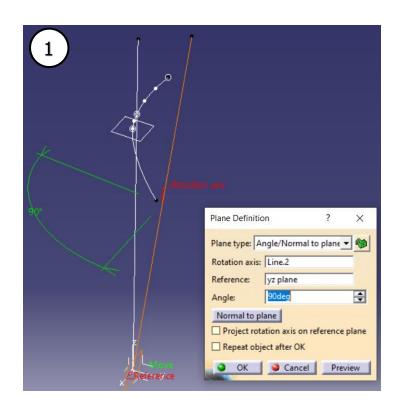
 Create an angle/normal to curve line from 0,0,0 relative to the line going through Point 3.

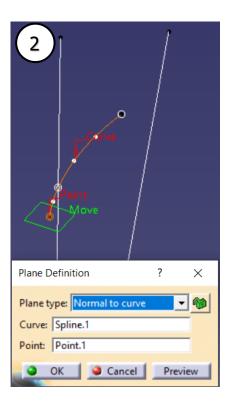


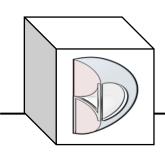




- Create two symmetry planes normal to plane YZ.
 - 1. Through Line 2.
 - 2. Normal to the spline and through Point 1.

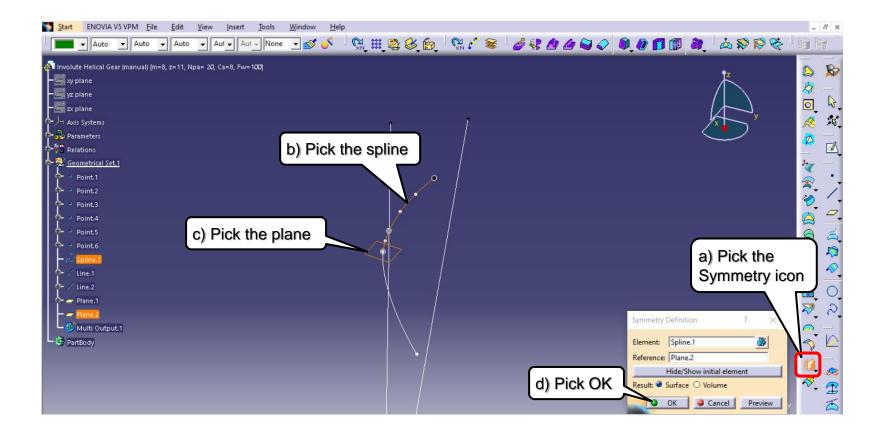


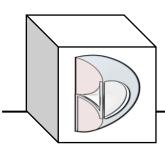






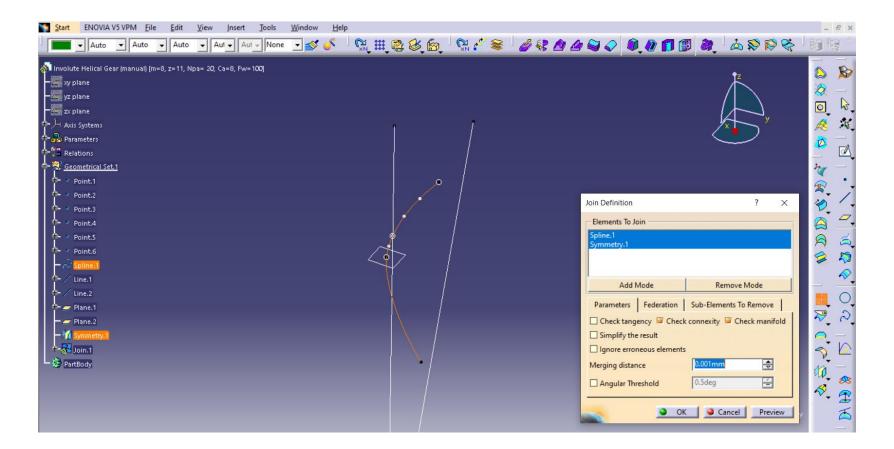
Symmetry the spline about Plane 2.

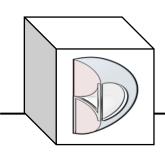






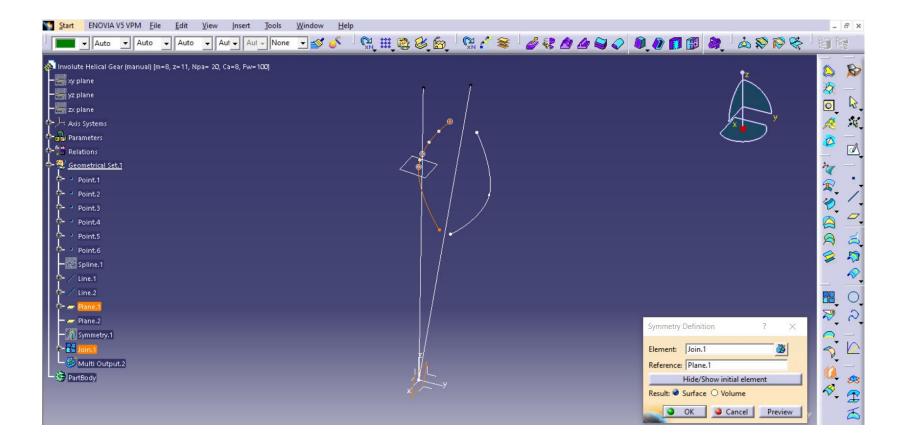
Join the two splines together.

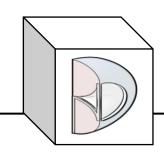






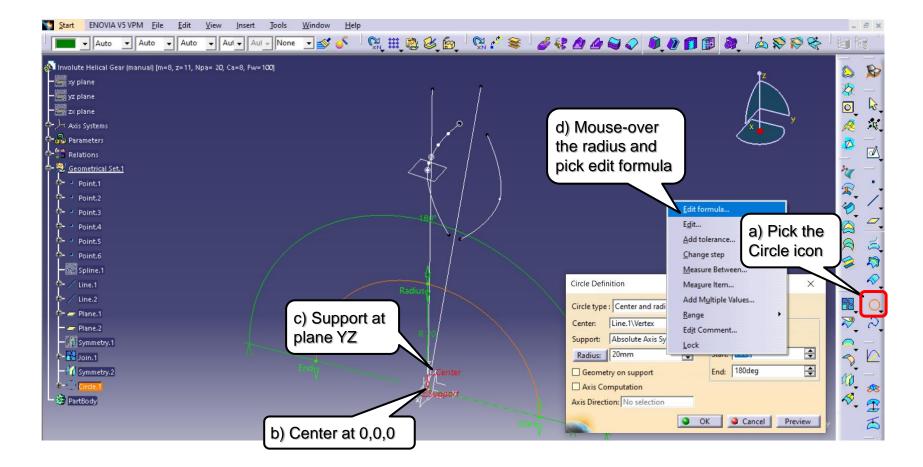
Symmetry the Join about Plane 1.

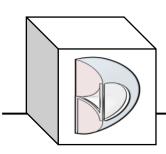






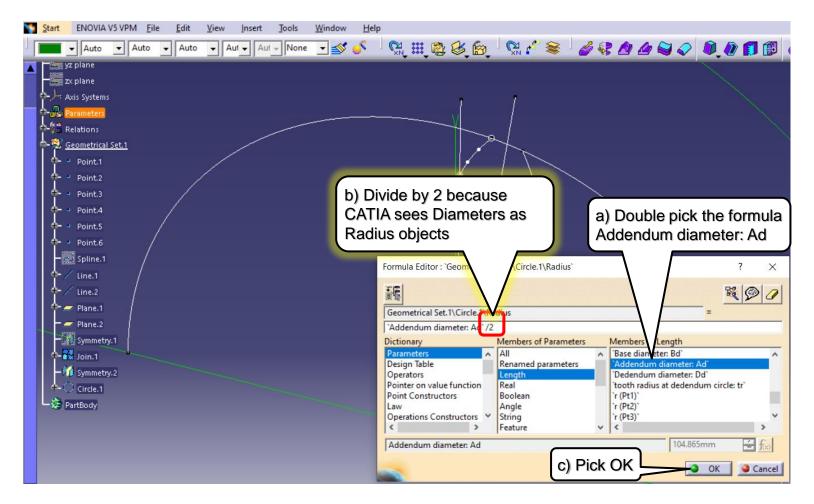
Create the Addendum diameter with center at 0,0,0 on plane YZ.

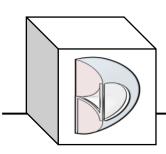






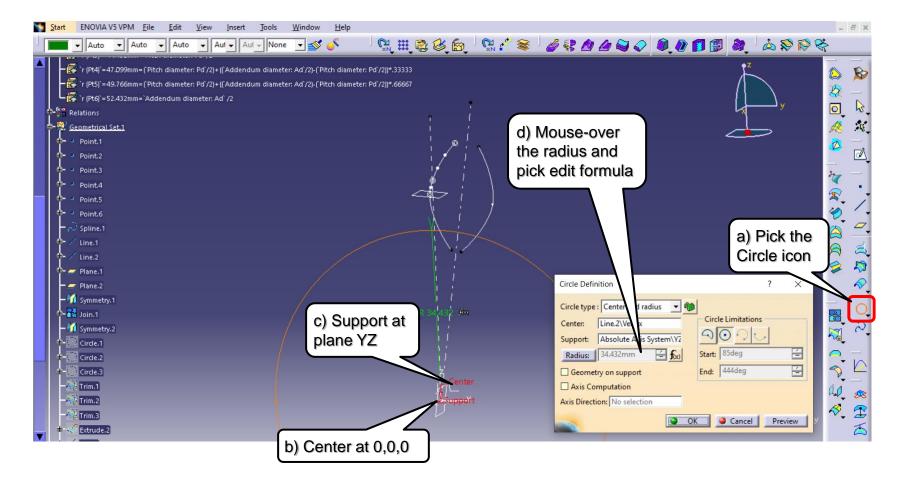
Apply the formula (Addendum diameter: Ad) to the addendum circle.

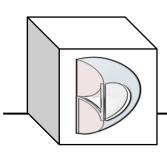






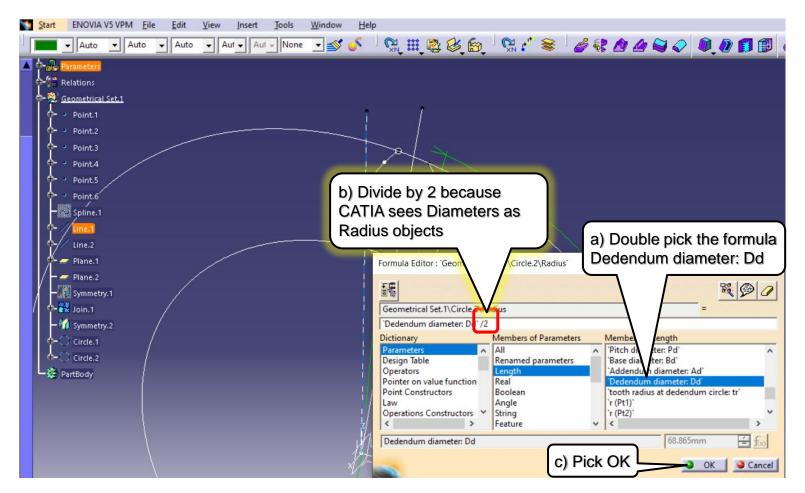
Create the Dedendum diameter with center at 0,0,0 on plane YZ.

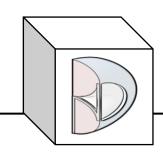






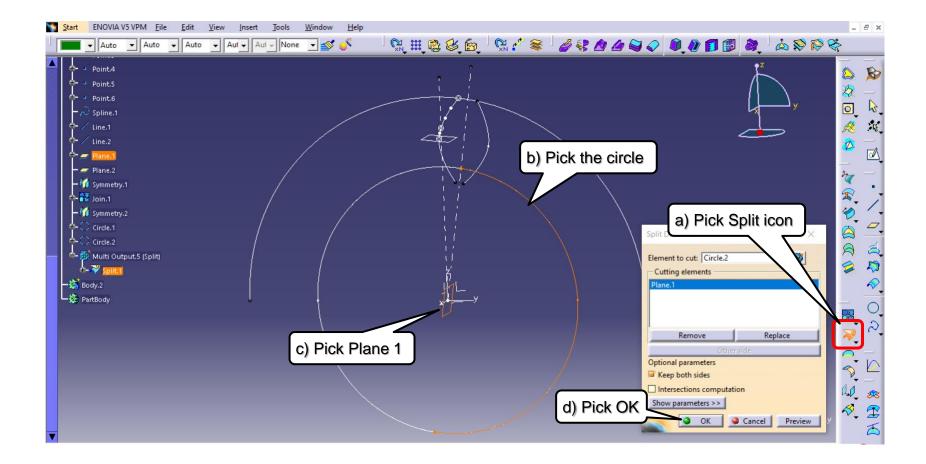
Apply the formula (Dedendum diameter: Dd) to the dedendum circle.

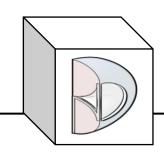




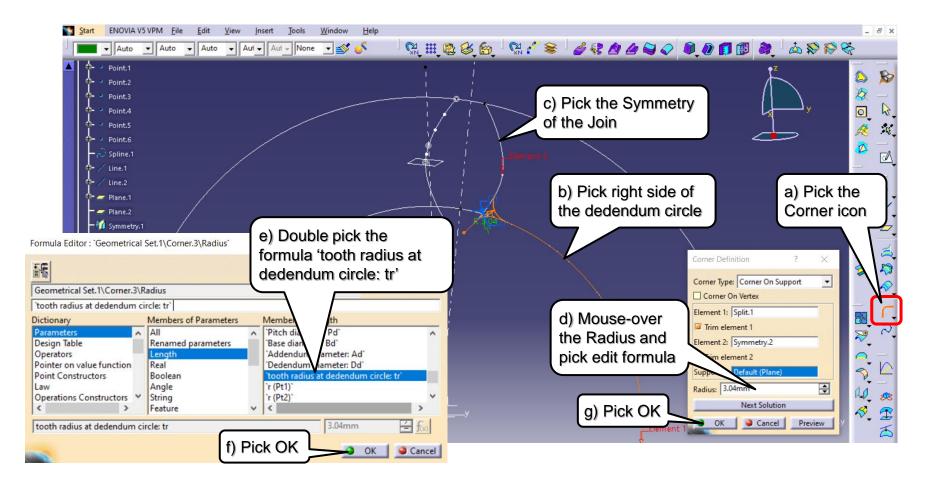


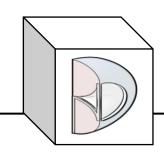
Split the dedendum circle with Plane 1.



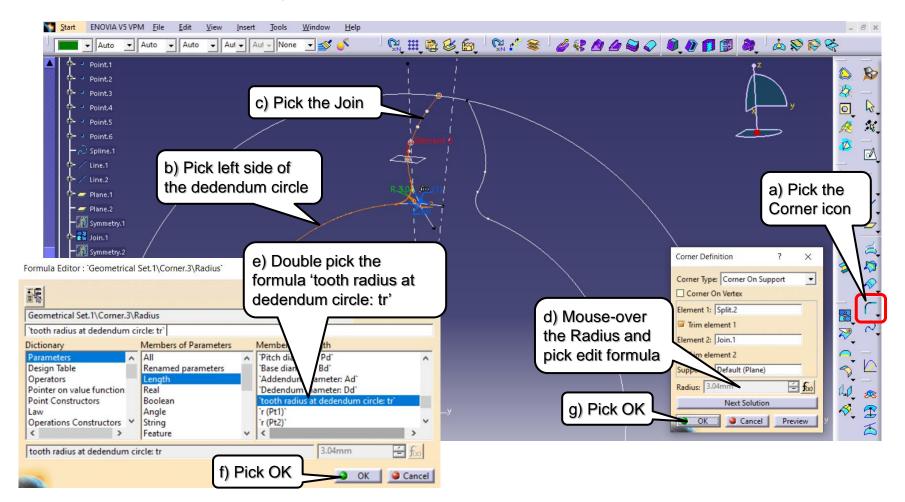


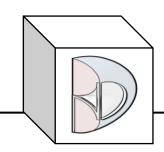




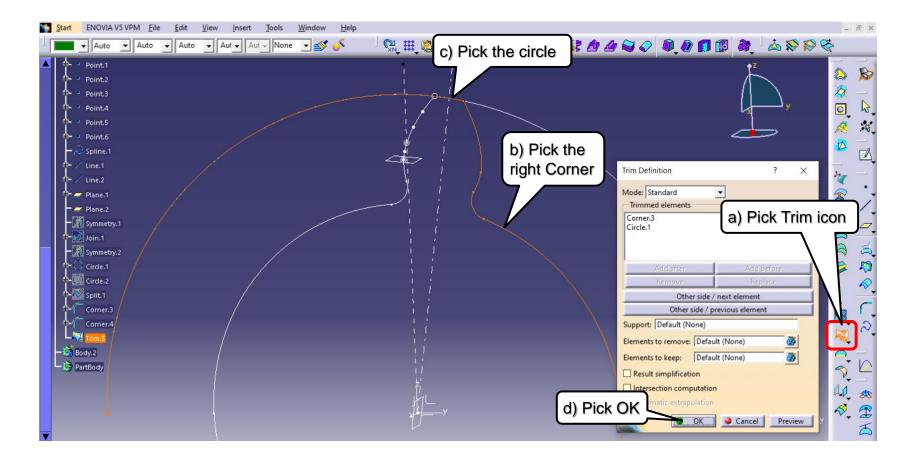


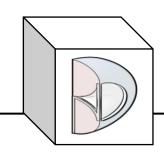




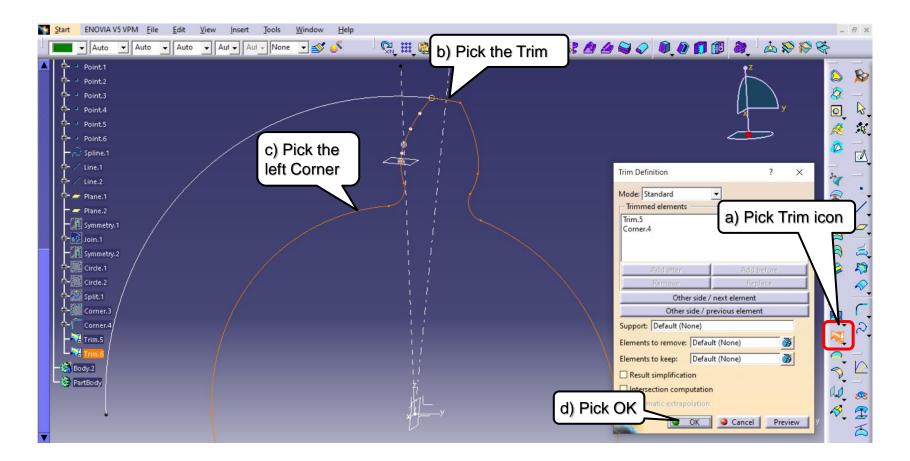


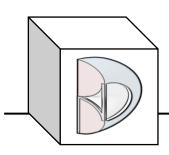






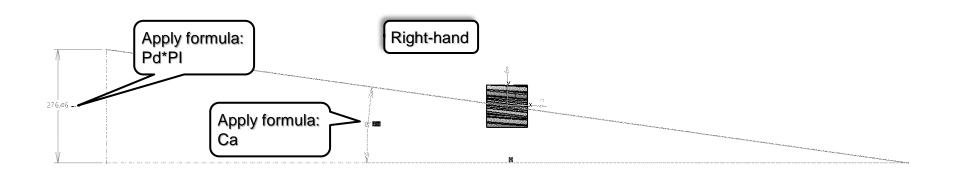


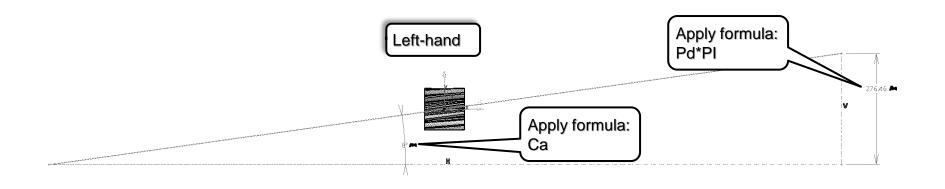


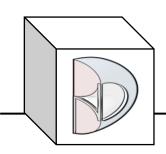




Create both Right & Left-hand Sketches on XY plane.

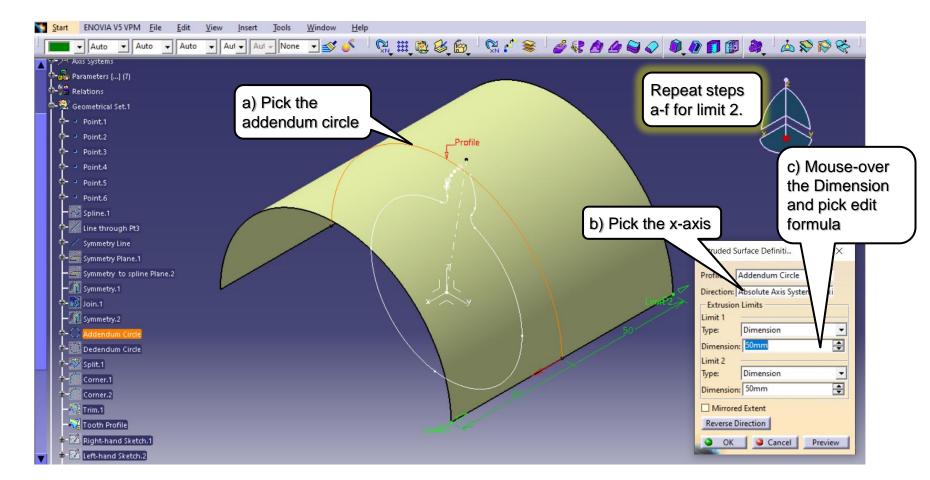


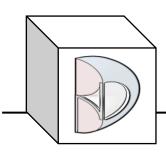






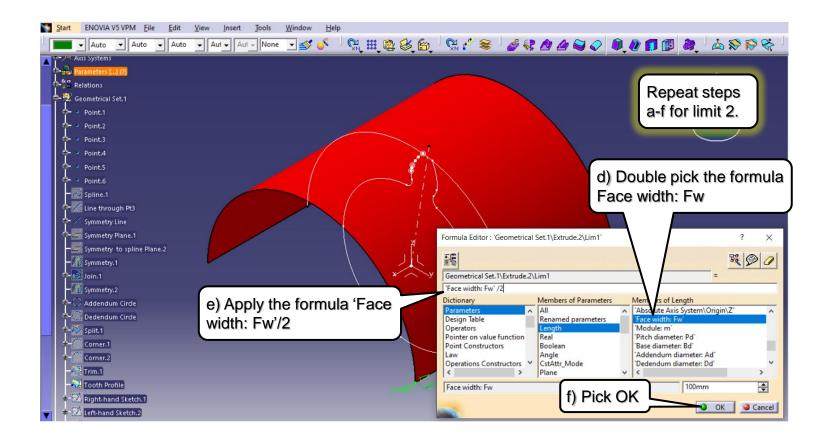
Extrude a surface using the Addendum circle.

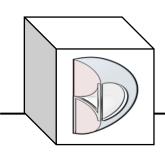






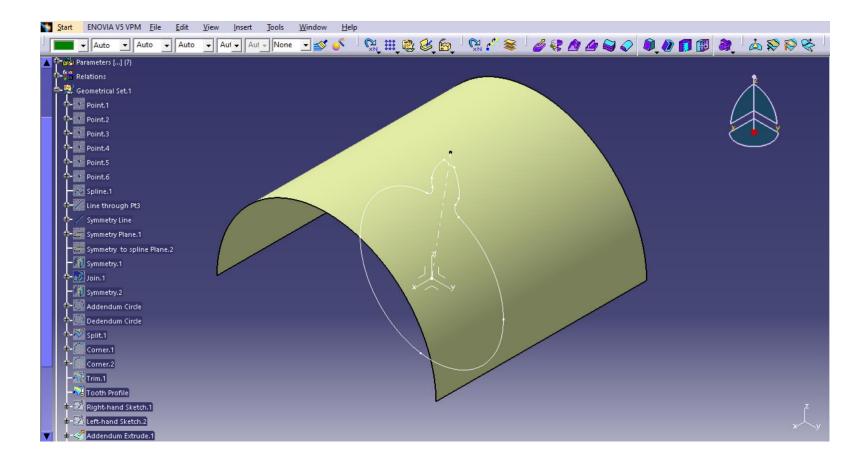
Apply the formula (Face width: Fw / 2) for the limits 1 & 2.

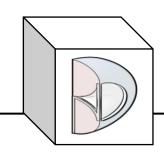






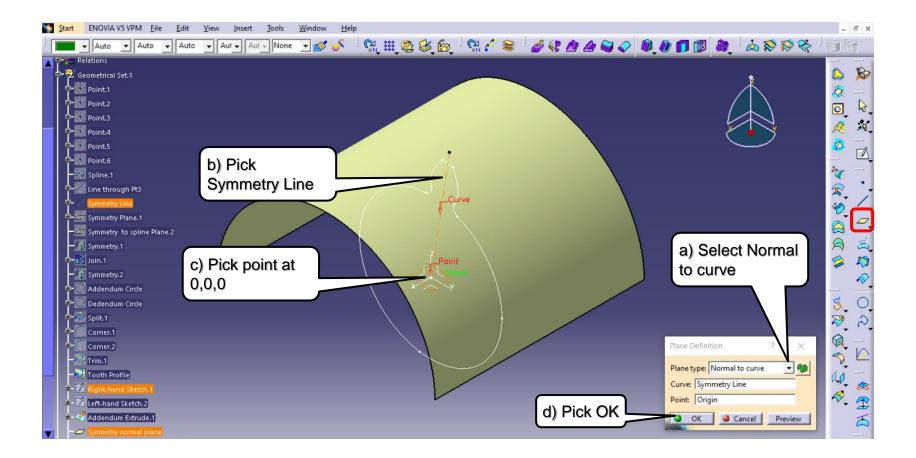
Clean up (hide) the construction geometry.

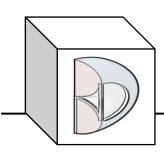






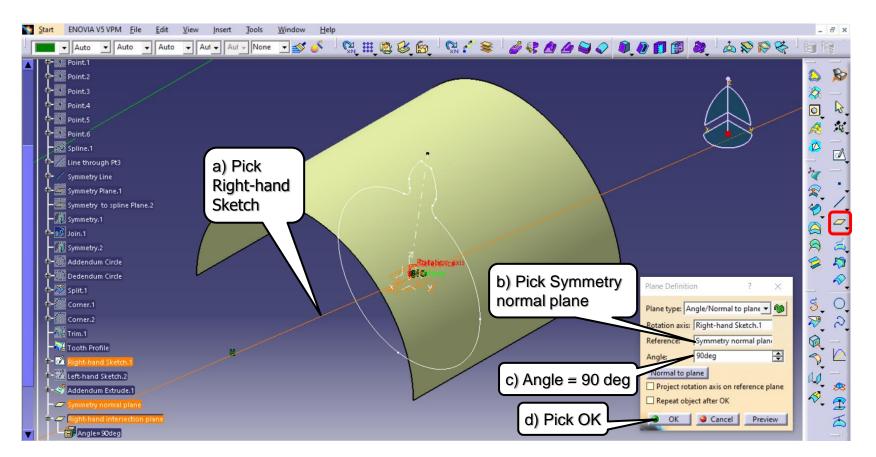
Create Symmetry normal plane.

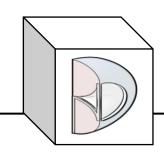






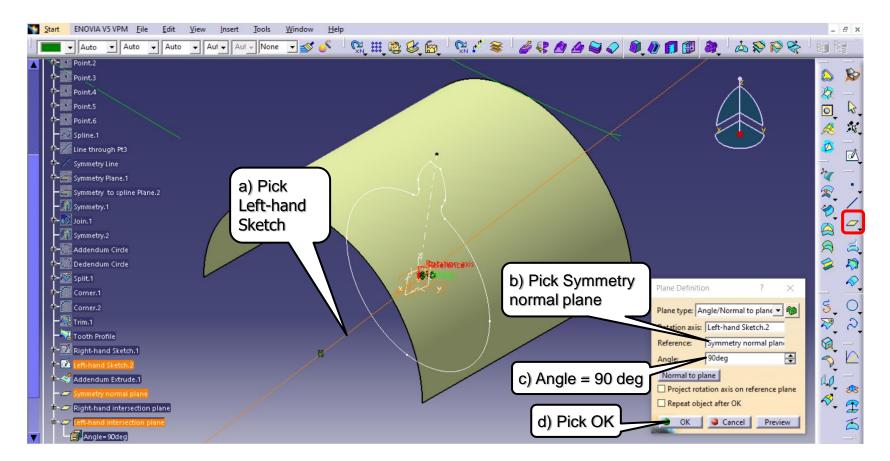
 Create the Right-hand intersection plane through Right-hand Sketch normal to the Symmetry normal plane.

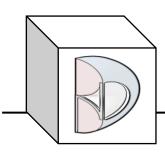






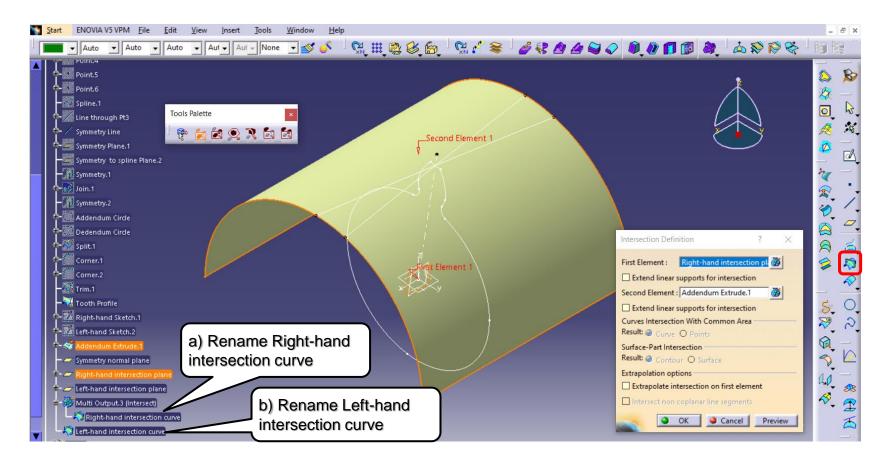
 Create the Left-hand intersection plane through Left-hand Sketch normal to the Symmetry normal plane.

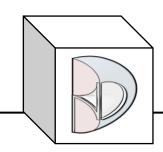






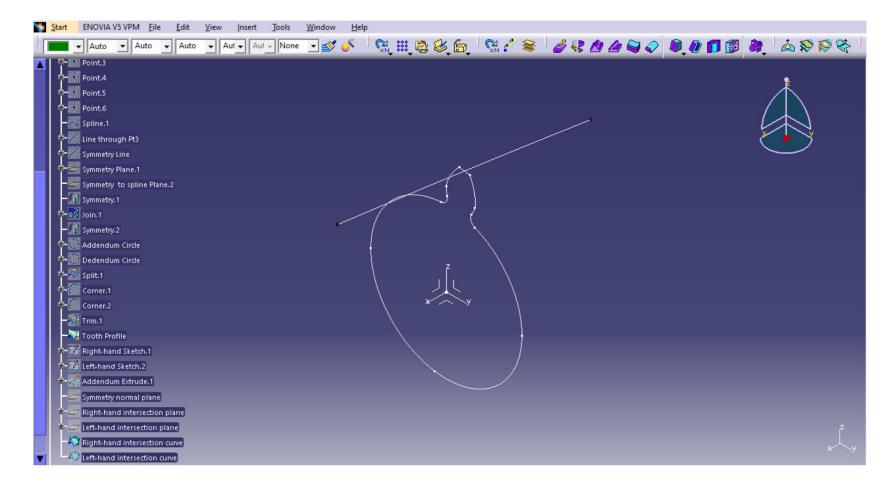
 Create the Right-hand and Left-hand intersection curves between the intersection planes and the extruded surface.

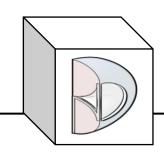






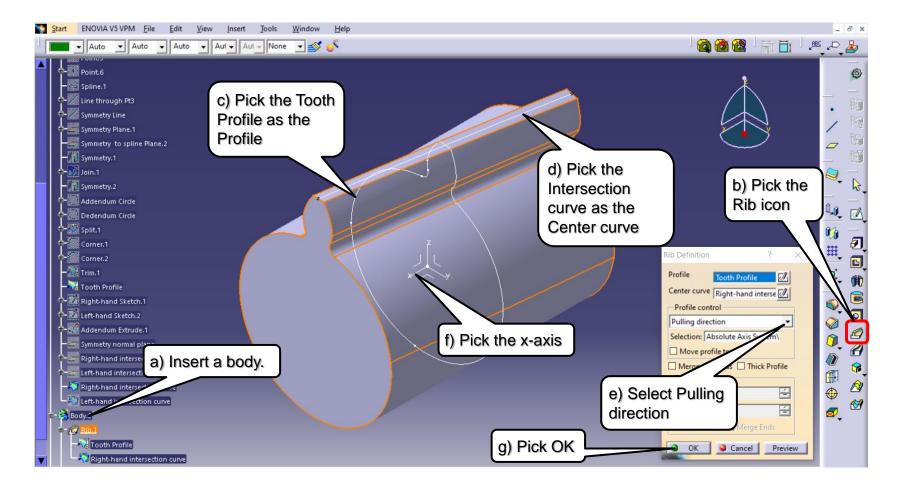
Clean up (hide) the construction geometry.

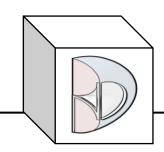




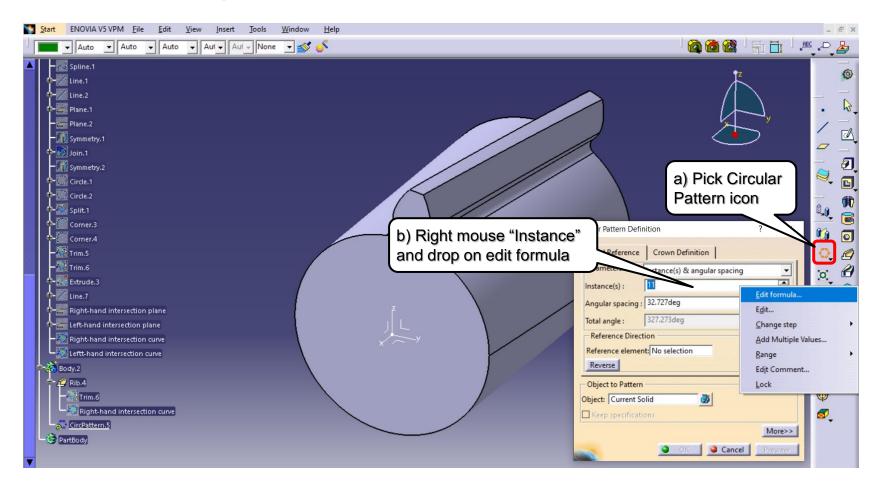


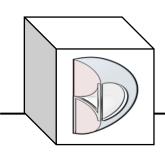
Create a solid rib using the Tooth Profile Trim as the Profile.



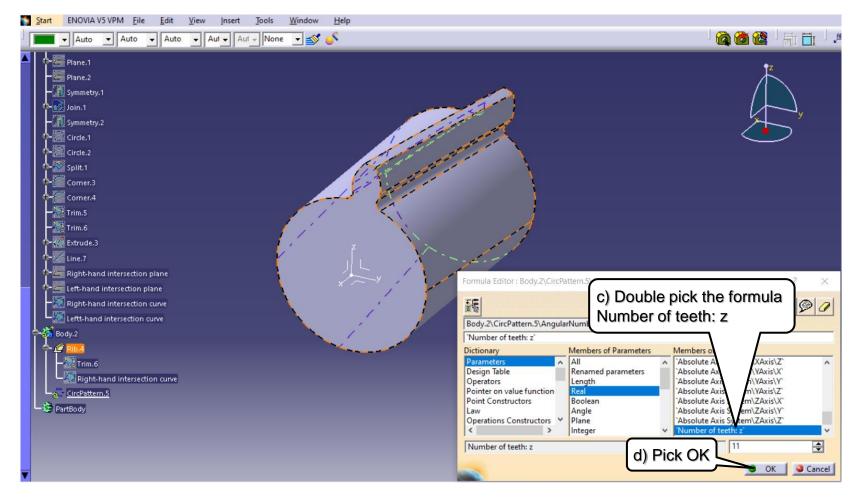


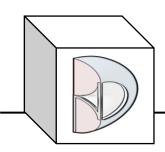




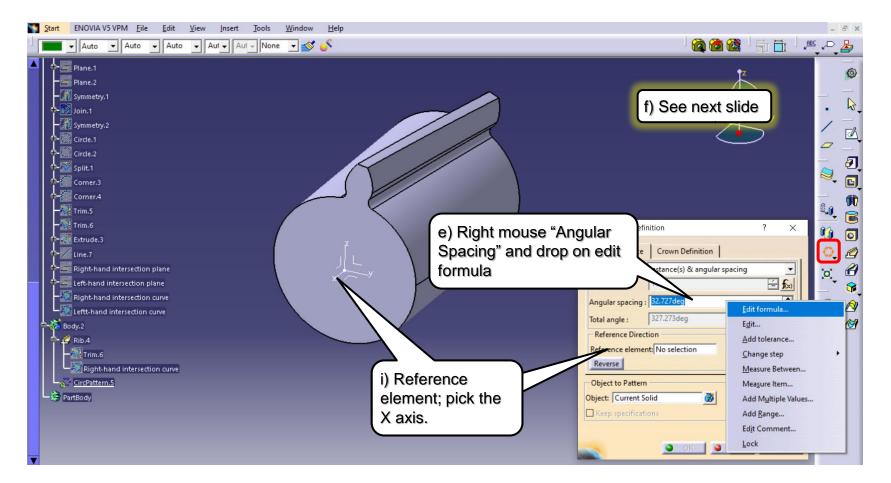


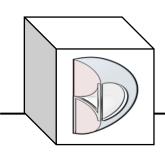




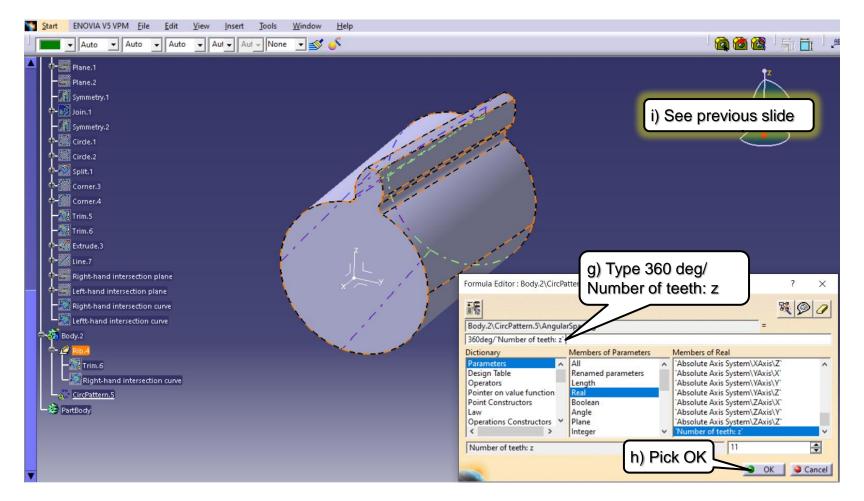


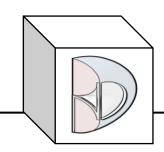






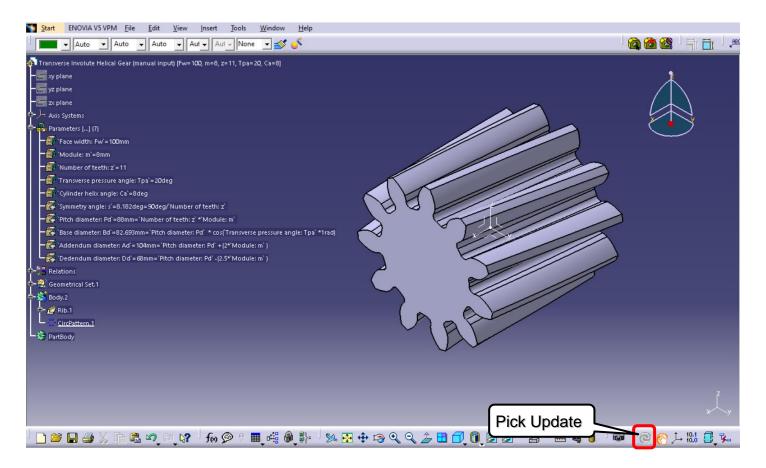


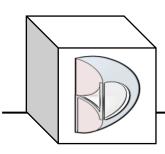






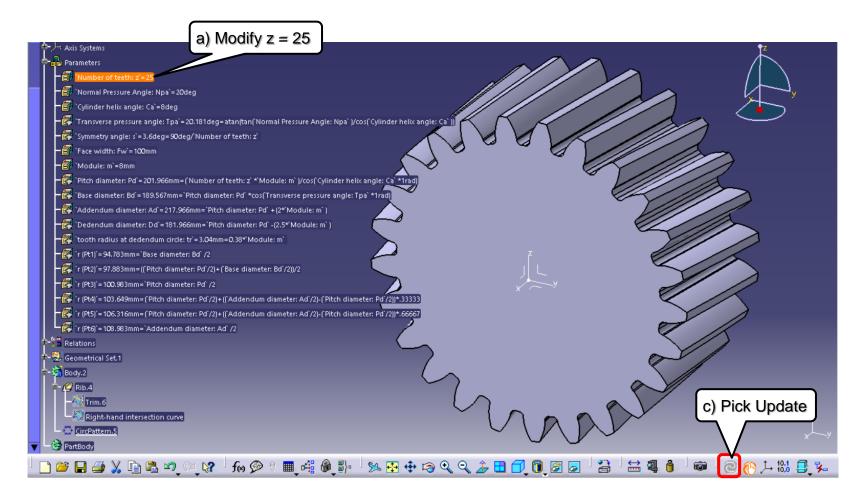
 And there you have it! An Involute Helical Gear which is modifiable through parameters.

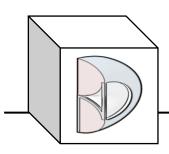






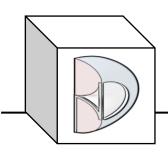
Modify parameters for Number of teeth: z to 25 and check the results.







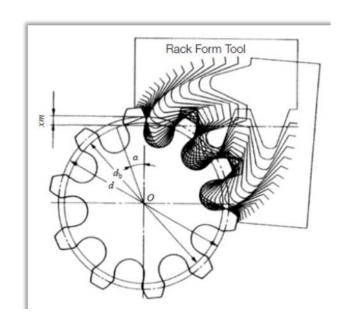
Undercutting

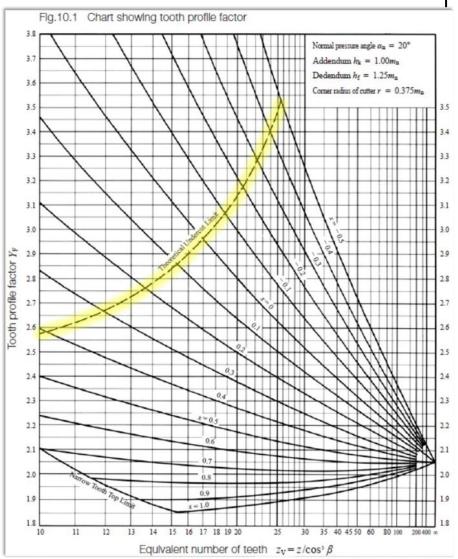


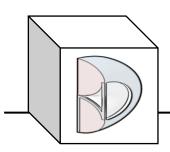


UNDERCUTTING

Undercutting occurs geometrically when the standard formulae allow for the dedendum circle to fall too far below the base circle. This can be rectified by adjusting both the addendum and dedendum circles.

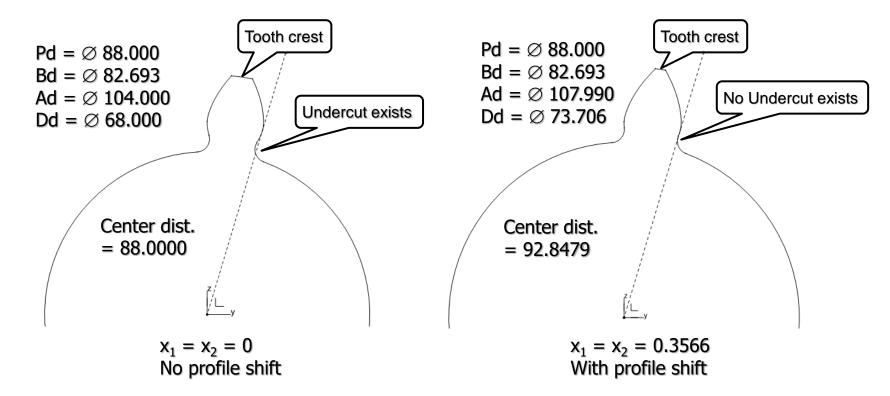


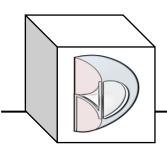




PROFILE SHIFT

- Profile shift (x) is not merely used to prevent undercut, it can also be used to adjust the center distance between two gears.
- If a positive correction is applied (addendum and dedendum circles increased) to prevent undercut in a gear, the tooth crest is sharpened.

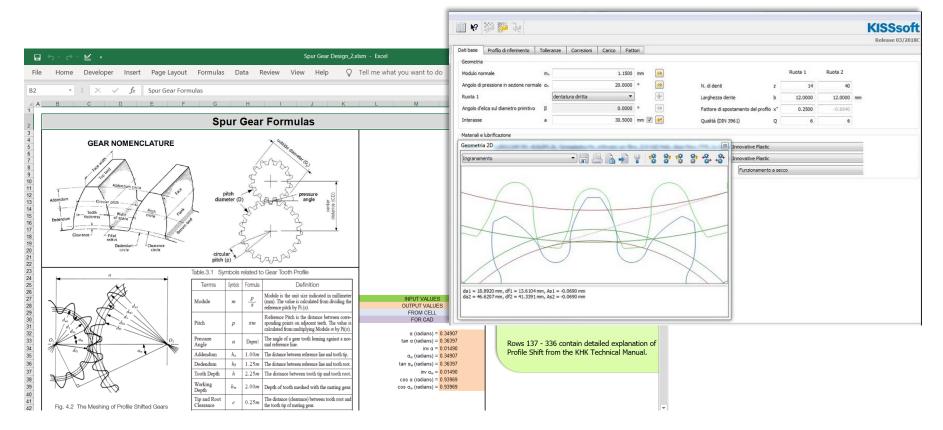


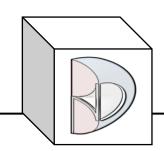




PROFILE SHIFT

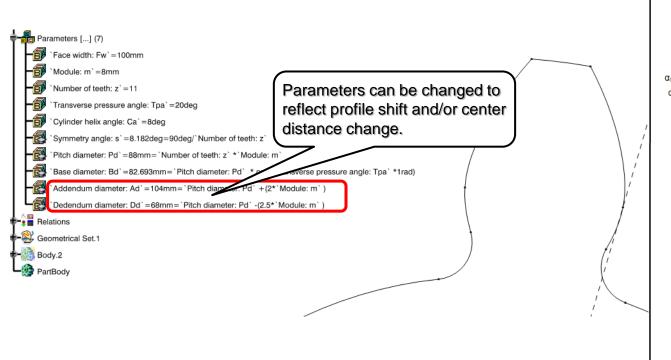
 By using a gear calculator (i.e. MS Excel) or a gear simulation software (i.e. KISSsoft) users can get the required values to optimize their design beyond the basic standard formulae we have used thus far.



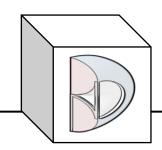




Parameters for Ad & Dd from standard formulae (no profile shift).

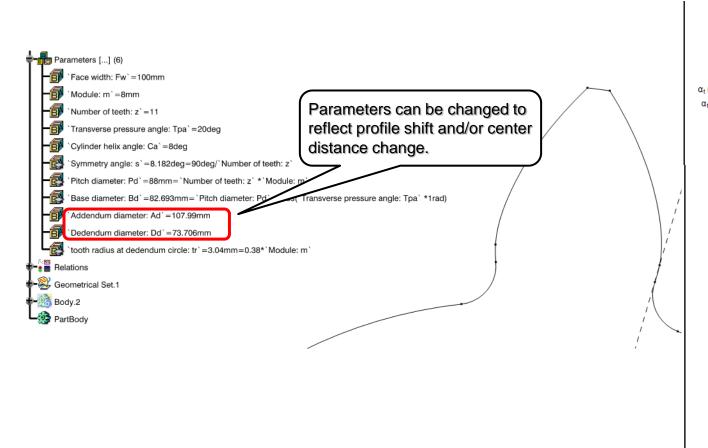


	Transverse System	
	Gear 1	Gear 2
Fw =	100	100
m _t =	8	8
t (degrees) =	20	20
α _t (radians) =	0.3491	0.3491
β =	8	8
z =	11	11
x _t =	0.00000	0.00000
Fw =	100.000	100.000
m =	8	8
z =	11	11
Tpa =	20	20
Ca =	8	8
s =	8.182	8.182
Pd =	88.000	88.000
Bd =	82.693	82.693
Ad =		104.000
Dd =	68.000	68.000
tr =	3.040	3.040
	00.000	
a =	88.0000	(88.)
y =	0.00000	
$\alpha_{wt} =$	20.0000	
u _{wt} –	20.0000	
$\Sigma x_n =$	0.000000	0.0000
		0.3566
x _n =	0.0000	0.0000

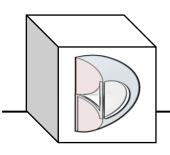




Parameters for Ad & Dd from updated formulae (with profile shift).

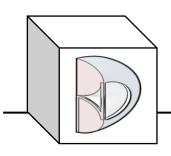


	Transverse System	
	Gear 1	Gear 2
Fw =	100	100
m _t =	8	8
(degrees) =	20	20
(radians) =	0.3491	0.3491
β =	8	8
, z =	11	11
x _t =	0.35660	0.35660
	(.3566)	
Fw =	100.000	100.000
m =	8	8
z =	11	11
Tpa =	20	20
Ca =	8	8
s =	8.182	8.182
Pd =	88.000	88.000
Bd =	82 693	82 693
Ad =	107.990	107.990
Dd =	73.706	73.706
tr =	3.040	3.040
a =	92.8479	(88.)
y =	0.60599	
α_{wt} =	27.0477	
$\Sigma x_n =$	0.713210	0.3566 0.3566
		0.5500
x _n =	0.3566	0.3566





- We now have a "template" part for our Transverse Module Involute Helical Gear.
- This part may be modified by simply changing the parameters of any/all of the following five formulae:
 - Fw Face width // length parameter [Fw = input]
 - z number of teeth // real parameter [z = input]
 - Npa Normal Pressure Angle // angle parameter [Npa = input]
 - 4. m module // length parameter [m = input]
 - Ca Cylinder helix angle // angle parameter [Ca = input]
- Next we will look at using CATIA V5 to create the Normal Module Involute Helical Gear.





Conclusion:

This is an example of designing Transverse Module Involute Helical Gear in CATIA V5 (manual input).

We hope this will help those who need this type of simulation.

As always, we are open to any discussions this may bring.

Please subscribe to our YouTube channel!

